Chapter 1: Pumps and Motors

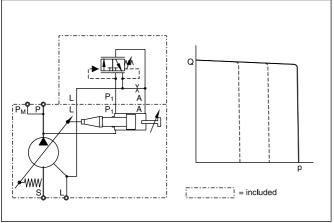
Series	Description	Characteri	stic values	Page	
Series	Description	p _N [bar]	V [cm³/rev]	rage	
	Axial piston pumps				
PV	With swash plate, adjustable, for open circuit	up to 350	16 - 270	1-2	
	Pump combinations			1-24	
	Compensators			1-27	
	Accessories General installation information			1-38 1-39	
PVM	With swash plate, adjustable, for open circuit	up to 280	16 - 92	1-39	
1 VIVI	Pump combinations	up to 200	10 32	1-57	
	Compensators			1-59	
	Accessories			1-61	
	General installation information			1-62	
PVP	With swash plate, adjustable, for open circuit	up to 250	16 - 140	1-64	
	Pump combinations			1-82	
	Compensators General installation information			1-84 1-86	
				1-00	
D1 (0	Vane pumps				
PVS	Adjustable	up to 140	8 - 50	1-88	
	Pump combinations Compensators			1-99 1-101	
	Accessories			1-101	
	General installation information			1-109	
	Gear pumps and motors				
PGP/PGM500	Aluminium body; for heavy-duty		0.8 - 52	1-111	
PGP/PGM620	Cast-iron body; for heavy duty	up to 275	16 - 52	1-157	
	Axial piston pumps and motors				
F11, F12	Fixed displacement, bent axis; for heavy duty	up to 480	4.88 - 242	1-168	
	Torgmotors TM		ı		
TE TE	TOTALITOTOLOIS				
TE,TF,	Low apped Coreter maters			1 407	
TG/BG,	Low-speed Gerotor motors			1-197	
TH, TK					

If you are interested in fast delivery, please follow this hint in our ordering codes when choosing your individual product:

Bold letters =Short-term availability



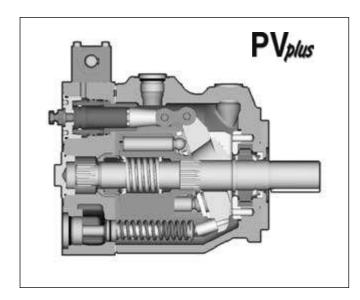
Introduction

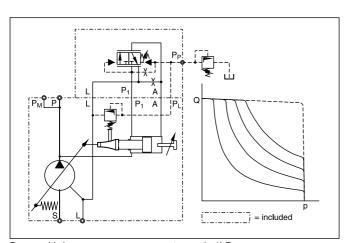


Pump with standard pressure compensator code F*S

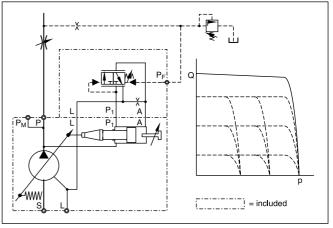
With thru drive for single and multiple pumps

Swash plate type for open circuit



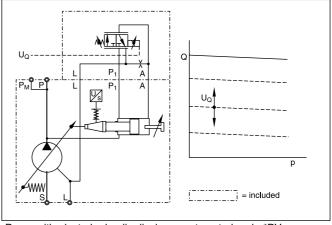


Pump with horse power compensator code *LB



Pump with load-sensing compensator code FFC

- Mounting interface according to VDMA-standards sheet 24560 part 1.
- Standard: 4-hole flange ISO 3019/2 (metric). optional: 4-hole flange ISO 3019/1 (SAE).
- Large servo piston with strong bias spring achieves fast response; e.g. for PV046 upstroke < 70 ms downstroke < 40 ms note; follow installation instructions.
- Reduced pressure peaks due to active decompression of system at downstroke.
- Also at low system pressure reliable compensator operation.
 lowest compensating pressure <10 bar.
- Nine piston and new precompression technology (precompression filter volume) result in unbeaten low outlet flow pulsation.
- Rigid and FEM-optimized body design for lowest noise level.
- Complete compensator program.
- Thru drive for 100% nominal torque.
- Pump combinations (multiple pumps) of same size and model and mounting interface for basically all metric or SAE mounting interfaces.



Pump with electrohydraulic displacement control code *PV



Characteristics

Technical data

Displacement	from 16 to 270 cm ³ /rev	
Operating pressures		
Outlet	nominal pressure p _N 350) bar
	max. pressure p _{max.} 420	bar ¹⁾
	drain port 2	bar1)
Inlet	min. 0.8 bar (absolute)	
	max. 16 bar	
Minimum speed	300 min ⁻¹	
Mounting		
interface	4-hole flange ISO 3019/2	
	optional ISO 3019/1, SAE	
Installation	drain port as high as possible	

¹⁾ peak pressure only



Pump with standard pressure comp.







Pump with horse power comp.



Combination PV/gear pump GP

Pump combinations

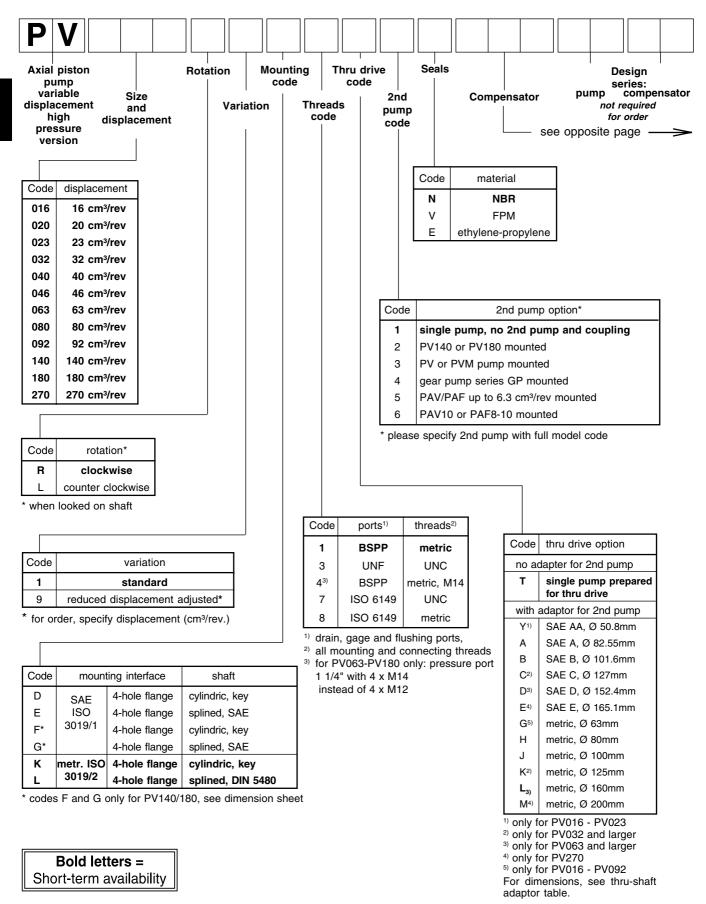
See pages 24 - 25.

Selection table

Model	Max. displacement in cm³/rev	Output flow in I/min at 1500 min ⁻¹	Input horse power in kW at 1500 min ⁻¹ and 350 bar	Max speed 1) in min ⁻¹	Weight in kg
PV016	16	24	15.5		
PV020	20	30	19.5	3000	19
PV023	23	34.5	22.5		
PV032	32	48	31		
PV040	40	60	39	2800	30
PV046	46	69	45		
PV063	63	94.5	61.5	2800	
PV080	80	120	78	2500	60
PV092	92	138	89.5	2300	
PV140	140	210	136	2400	90
PV180	180	270	175	2200	90
PV270	270	405	263	1800	172

¹⁾ The maximum speed ratings are shown for an inlet pressure of 1 bar (absolute) and for a fluid viskosity of v = 30 mm²/s.

Ordering Code



Ordering Code

P	V											
							Compen	nsator	D pum		series:	
				1						-	for orde	

Code Compensator option 0 0 1 No compensator Standard pressure compensator F D S 10 - 140 bar, spindle + lock nut F H S 40 - 210 bar, spindle + lock nut Remote compensator options F R remote pressure compensator F S variation R, for quick unload valve I load-sensing compensator F T two valve load-sensing compensator Variations for remote compensators Variations for remote compensators C1 external pressure pilot 1 NG6/D03 interface top side P pilot valve PVAC1Pmounted D DSAE1007P07KLAF mounted D pilot valve with DIN lock mounted										
Standard pressure compensator F D S 10 - 140 bar, spindle + lock nut F H S 40 - 210 bar, spindle + lock nut Remote compensator options F R remote pressure compensator F S variaton R, for quick unload valve I load-sensing compensator F T two valve load-sensing compensator Variations for remote compensators Variations for remote compensators C1 external pressure pilot 1 NG6/D03 interface top side P pilot valve PVAC1Pmounted D proportional pilot valve type DSAE1007P07KLAF mounted pilot valve with DIN lock mounted	(Cod	е	Compensator option						
F D S 10 - 140 bar, spindle + lock nut F H S 40 - 210 bar, spindle + lock nut F W S 70 - 350 bar, spindle + lock nut Remote compensator options F R remote pressure compensator F S variaton R, for quick unload valve Ioad-sensing compensator F T two valve load-sensing compensator Variations for remote compensators C1 external pressure pilot 1 NG6/D03 interface top side P pilot valve PVAC1Pmounted D proportional pilot valve type DSAE1007P07KLAF mounted pilot valve with DIN lock mounted	0	0	1	No compensator						
F H S 40 - 210 bar, spindle + lock nut F W S 70 - 350 bar, spindle + lock nut Remote compensator options F R remote pressure compensator variation R, for quick unload valve load-sensing compensator F T two valve load-sensing compensator Variations for remote compensators C1 external pressure pilot 1 NG6/D03 interface top side P pilot valve PVAC1Pmounted D proportional pilot valve type DSAE1007P07KLAF mounted pilot valve with DIN lock mounted		S	tand	ard pressure compensator						
F W S 70 - 350 bar, spindle + lock nut Remote compensator options F R remote pressure compensator variaton R, for quick unload valve load-sensing compensator two valve load-sensing compensator Variations for remote compensators Variations for remote compensators C1 external pressure pilot NG6/D03 interface top side pilot valve PVAC1Pmounted D proportional pilot valve type DSAE1007P07KLAF mounted pilot valve with DIN lock mounted	F	D	S	10 - 140 bar, spindle + lock nut						
Remote compensator options F R remote pressure compensator variation R, for quick unload valve load-sensing compensator two valve load-sensing compensator Variations for remote compensators C1 external pressure pilot 1 NG6/D03 interface top side P pilot valve PVAC1Pmounted D proportional pilot valve type DSAE1007P07KLAF mounted pilot valve with DIN lock mounted	F	н	s	40 - 210 bar, spindle + lock nut						
F R remote pressure compensator variaton R, for quick unload valve load-sensing compensator two valve load-sensing compensator Variations for remote compensators C¹) external pressure pilot NG6/D03 interface top side P pilot valve PVAC1Pmounted D proportional pilot valve type DSAE1007P07KLAF mounted pilot valve with DIN lock mounted	F	w	S	70 - 350 bar, spindle + lock nut						
F S variaton R, for quick unload valve F F F load-sensing compensator T variations for remote compensators C1 external pressure pilot NG6/D03 interface top side P pilot valve PVAC1Pmounted D proportional pilot valve type DSAE1007P07KLAF mounted pilot valve with DIN lock mounted		R	lemo	te compensator options						
F F T Ioad-sensing compensator two valve load-sensing compensator Variations for remote compensators C1 external pressure pilot 1 NG6/D03 interface top side P pilot valve PVAC1Pmounted D proportional pilot valve type DSAE1007P07KLAF mounted pilot valve with DIN lock mounted	F	FR		remote pressure compensator						
T two valve load-sensing compensator Variations for remote compensators C1 external pressure pilot NG6/D03 interface top side P pilot valve PVAC1Pmounted proportional pilot valve type DSAE1007P07KLAF mounted pilot valve with DIN lock mounted	F	s		variaton R, for quick unload valve						
Variations for remote compensators C1) external pressure pilot NG6/D03 interface top side P pilot valve PVAC1Pmounted proportional pilot valve type DSAE1007P07KLAF mounted pilot valve with DIN lock mounted	F	F		load-sensing compensator						
C1) external pressure pilot 1 NG6/D03 interface top side P pilot valve PVAC1Pmounted D proportional pilot valve type DSAE1007P07KLAF mounted L pilot valve with DIN lock mounted	F	Т		two valve load-sensing compensator						
1 NG6/D03 interface top side P pilot valve PVAC1Pmounted D proportional pilot valve type DSAE1007P07KLAF mounted L pilot valve with DIN lock mounted		٧	ariati	ons for remote compensators						
P pilot valve PVAC1Pmounted D proportional pilot valve type DSAE1007P07KLAF mounted L pilot valve with DIN lock mounted			C 1)	external pressure pilot						
D proportional pilot valve type DSAE1007P07KLAF mounted L pilot valve with DIN lock mounted			1	NG6/D03 interface top side						
DSAE1007P07KLAF mounted L pilot valve with DIN lock mounted			Р	pilot valve PVAC1Pmounted						
L pilot valve with DIN lock mounted			D							
1 1 11 '										
1 1 70/1			L	pilot valve with DIN lock mounted						
Z ² accessory mounted			$Z^{2)}$	accessory mounted						

Dis	plac	eme	nt		С	ode		Compensator option						
				Н	lorse	pov	wer o	compensator						
032 046		140	180	270				nominal horse power						
					В			3 kW						
					С			4 kW						
					D			5.5 kW						
					Е			7.5 kW						
					G			11 kW						
					Η		15 kW							
					K			18.5 kW						
					М			22 kW						
					S			30 kW						
					Т		37 kW							
					U			45 kW						
					W			55 kW						
					Υ			75 kW						
					Z			90 kW						
					2			110 kW						
					3			132 kW						
						fu	nctic	on						
						L		horse power compensator						
						С		horse power comp.& loadsensing						
					va	ıriati	on							
							Α	NG6/D03 interface top side						
							В	no pressure compensation						
						C adjustable pressure compensation								
							Z ²⁾	accessories mounted						

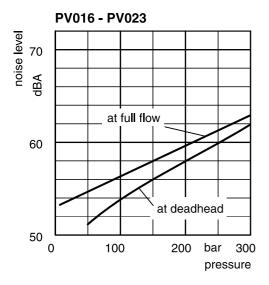
	Cod	е	Compensator option						
El	ectr	ohyd	raulic compensator						
Pi	lot p	ress	ure supply						
F			standard (internal) no shuttle valve						
W			with shuttle valve, compensator horizontal						
Fι	uncti	on							
Р			proportional displacement control						
Va	ariat	ion							
		٧	standard, no pressure compensation						
		R	remote pressure compensation NG 6 interface top side						
		D	proportional pilot valve type DSAE1007P07KLAF mounted						
		Z ²⁾	variation R, accessories mounted on topside interface						
		G	variation R, pressure sensor and proportional pilot valve mounted for pressure resp. horse power control						
		S	remote pressure compensation NG 6 interface top side, for quick unload valve						
		Т	variation S, pressure sensor and proportional pilot valve mounted for pressure resp. horse power control						
		Р	remote pressure compensation NG 6 interface top side, for preload and quick unload manifold						
		Е	variation P, pressure sensor and proportional pilot valve mounted for pressure resp. horse power control						

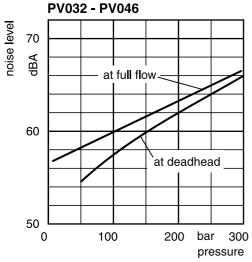
Bold letters = Short-term availability

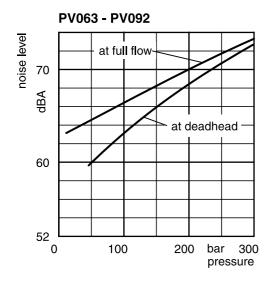


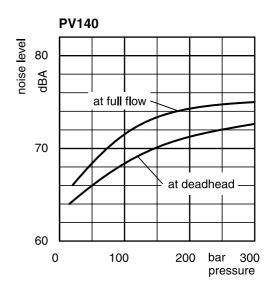
¹⁾ not for two-valve-compensator

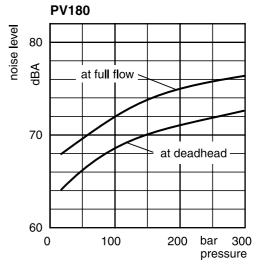
²⁾accessories not included, please specify on order with full model code

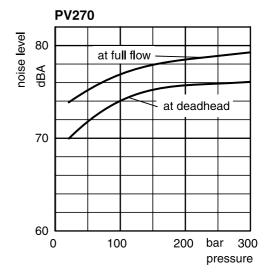












Typical sound level for single pumps, measured in unechoic chamber according to DIN 45 635, part 1 and 26. microphone distance 1 m. speed: n = 1500 min-1.

All data measured with mineral oil viscosity 30 mm 2 /s (cSt) at 50 $^{\circ}$ C.



Noise reduction measures

Operating noise of pumps

The normal operating noise of a pump and consequently the operating noise of the entire hydraulic system is largely determined by **where** and **how** the pump is mounted and how it is connected to the downstream hydraulic system.

Also size, style and installation of the hydraulic tubing have a major influence on the overall noise emitted by a hydraulic system

Noise reduction measures

Talking about operating noise of a hydraulic pump, primary and secondary pump noise has to be taken into consideration

Primary pump noise is caused by vibrations of the pump body due to internal alternating forces stressing the body structure.

Flexible elements help to prevent pump body vibration being transmitted to other construction elements, where possible amplification may occur. Such elements can be: Bell housing with elastic dampening flange with vulcanized labyrinth (1)

Floating and flexible coupling (2)

Damping rails (3) or silent blocks for mounting the electric motor or the foot mounting flange

Flexible tube connections (compensators) or hoses on inlet, outlet and drain port of the pump.

Exclusive use of gas tight tube fittings for inlet connections to avoid ingression of air causing cavitation and excessive noise.

Secondary pump noise is caused by vibration induced into all connected hydraulic components by the flow and pressure pulsation of the pump. This secondary noise adds typical 7 - 10 dBA to the noise of a pump measured in the sound chamber according to DIN 45 635 (see diagrams on opposite side). Therefore pipework, its mounting and the mounting of all hydraulic components like pressure filters and control elements has a major influence to the overall system noise level.

Pulsation reduction with precompression volume: The PV is equipped with a new technology for flow ripple reduction. This method reduces the pulsation at the pump outlet by **40 - 60** %. That leads to a significant reduction of the overall system noise without additional cost and without additional components (silencers etc.). The typical reduction reaches **2 - 4 dBA**. That means: with a pump of the PV series the secondary noise adds only some 5 - 7 dBA to the pump noise instead of the usually found 7 - 10 dBA. Figure 2 compares the measured pulsation of a system with 6 pumps of 180 cm³/rev each.

Last but not least the connection between pump and driving motor can be the cause of an unacceptably high noise emission.

Even when the mounting space is limited there are suitable means and components to reduce the noise significantly. The vibration of the pump body, created by high alternating forces in the rotating group and the pulsation of the output flow excite every part of the system connected to the pump mechanically or hydraulically.

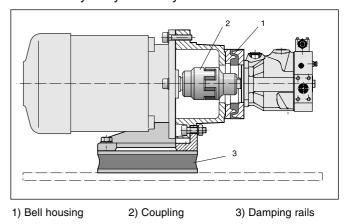


Figure 1: Components to avoid vibration transfer from the pump to the drive/installation and their position in the power unit (numbers refer to the text on the left)

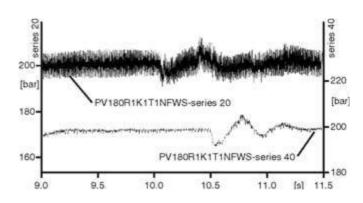


Figure 2: Comparison of the pressure pulsation in a system with 6 old PV pumps versus the same system with 6 PVplus pumps. The pulsation reduction effect of the precompression volume is evident.

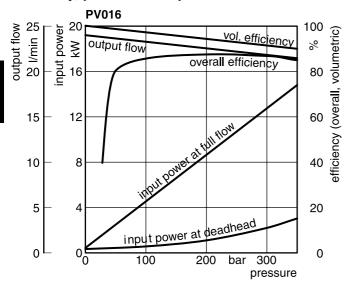
Other measures

Small diameter tubes do not only cause high flow speeds, turbulences inside the tubes and cavitation in the pump, they also produce noise.

Only correctly sized connections of the largest possible diameter according to the port size of the pump should be used.



Efficiency, power consumption

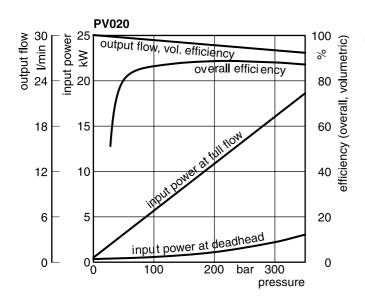


Efficiency and case drain flows PV016, PV020, PV023

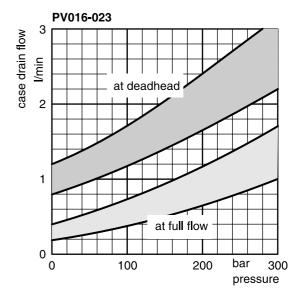
The efficiency and power graphs are measured at an input speed of $n = 1500 \text{ min}^{-1}$, a temperature of 50°C and a fluid viscosity of $30 \text{ mm}^{2}/\text{s}$.

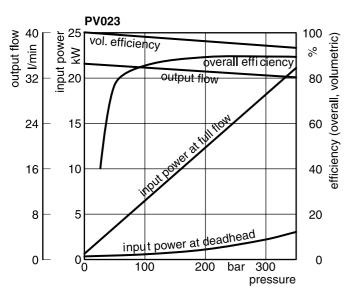
Case drain flow and compensator control flow leave via the drain port of the pump. To the values shown are to be added 1 to 1.2 l/min , if at pilot operated compensators (codes FR * , FF * , FT * , horse power compensator and p/Q-control) the control flow of the pressure pilot valve also goes through the pump.

Please note: The values shown below are only valid for static operation. Under dynamic conditions and at rapid compensation of the pump the volume displaced by the servo piston also leaves the case drain port. This dynamic control flow can reach up to 40 l/min! Therefore the case drain line is to lead to the reservoir at full size and without restrictions as short and direct as possible.



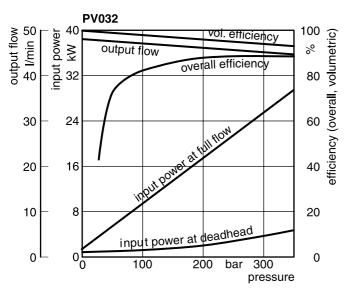
Case drain flows







Efficiency, power consumption

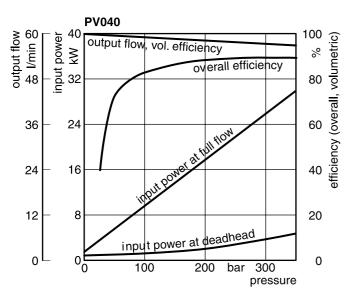


Efficiency and case drain flows PV032, PV040, PV046

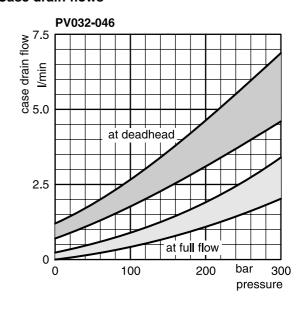
The efficiency and power graphs are measured at an input speed of $n = 1500 \text{ min}^{-1}$, a temperature of 50°C and a fluid viscosity of $30 \text{ mm}^{2}/\text{s}$.

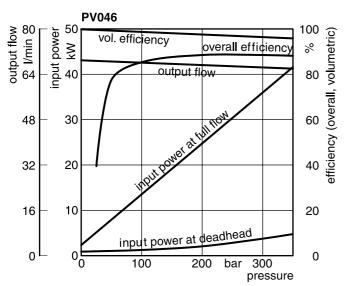
Case drain flow and compensator control flow leave via the drain port of the pump. To the values shown are to be added 1 to 1.2 l/min , if at pilot operated compensators (codes FR * , FF * , FT * , horse power compensator and p-Q-control) the control flow of the pressure pilot valve also goes through the pump.

Please note: The values shown below are only valid for static operation. Under dynamic conditions and at rapid compensation of the pump the volume displaced by the servo piston also leaves the case drain port. This dynamic control flow can reach up to 60 l/min! Therefore the case drain line is to lead to the reservoir at full size and without restrictions as short and direct as possible.

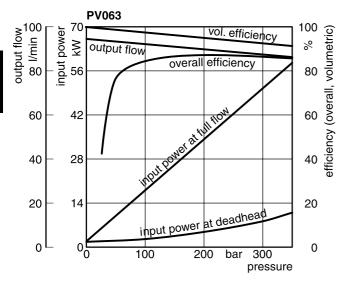


Case drain flows





Efficiency, power consumption

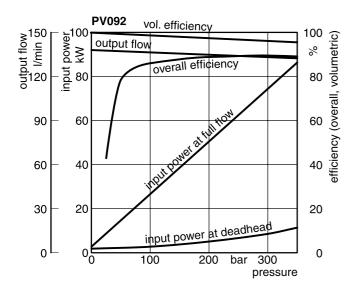


PV080 output flow 96 //min 75 100 efficiency (overall, volumetric) input power output flow, vol. efficiency % ₹ overall efficiency 80 72 48 60 32 40 48 16 20 24 input power at deadhead

100

200

bar 300 pressure



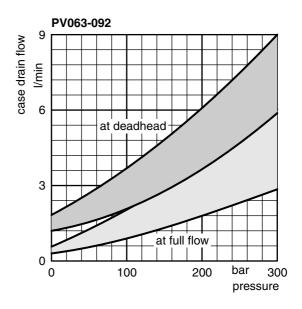
Efficiency and case drain flows PV063, PV080, PV092

The efficiency and power graphs are measured at an input speed of $n = 1500 \text{ min}^{-1}$, a temperature of 50°C and a fluid viscosity of 30 mm²/s.

Case drain flow and compensator control flow leave via the drain port of the pump. To the values shown are to be added 1 to 1.2 l/min , if at pilot operated compensators (codes FR * , FF * , FT * , horse power compensator and p-Q-control) the control flow of the pressure pilot valve also goes through the pump.

Please note: The values shown below are only valid for static operation. Under dynamic conditions and at rapid compensation of the pump the volume displaced by the servo piston also leaves the case drain port. This dynamic control flow can reach up to 80 l/min! Therefore the case drain line is to lead to the reservoir at full size and without restrictions as short and direct as possible.

Case drain flows



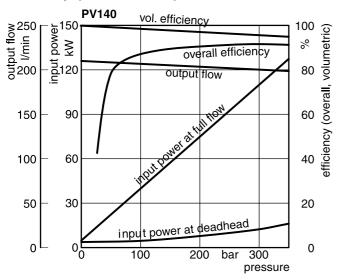
PV_GB.PM6.5MM

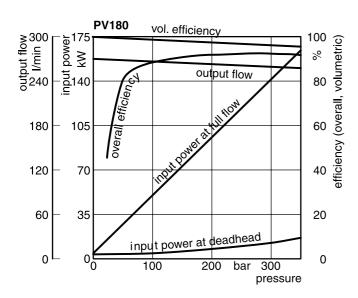
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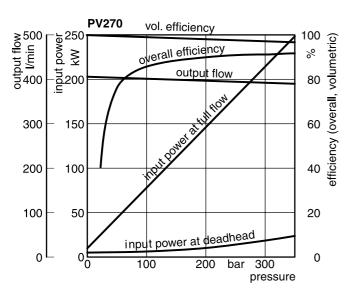


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Efficiency, power consumption







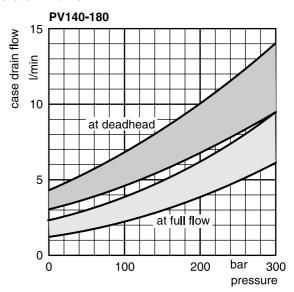
Efficiency and case drain flows PV140, PV180, PV270

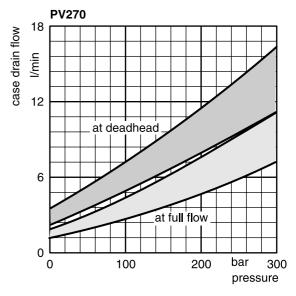
The efficiency and power graphs are measured at an input speed of $n = 1500 \text{ min}^{-1}$, a temperature of 50°C and a fluid viscosity of $30 \text{ mm}^{2}/\text{s}$.

Case drain flow and compensator control flow leave via the drain port of the pump. To the values shown are to be added 1 to 1.2 l/min , if at pilot operated compensators (codes FR * , FF * , FT * , horse power compensator and p-Q-control) the control flow of the pressure pilot valve also goes through the pump.

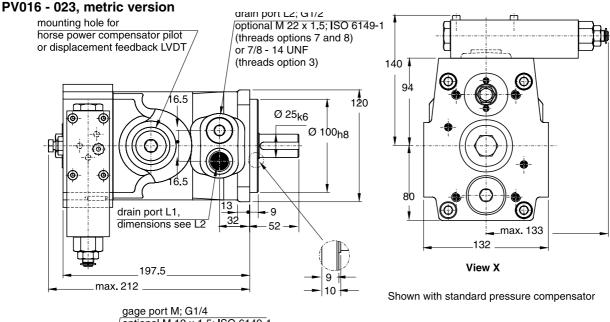
Please note: The values shown below are only valid for static operation. Under dynamic conditions and at rapid compensation of the pump the volume displaced by the servo piston also leaves the case drain port. This dynamic control flow can reach up to 120 l/min! Therefore the case drain line is to lead to the reservoir at full size and without restrictions as short and direct as possible.

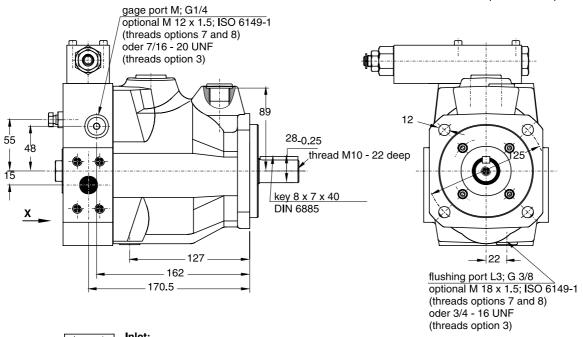
Case drain flows

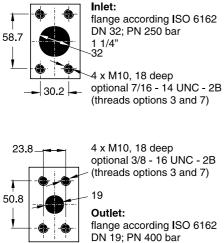






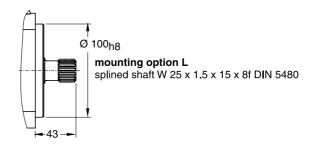






3/4"

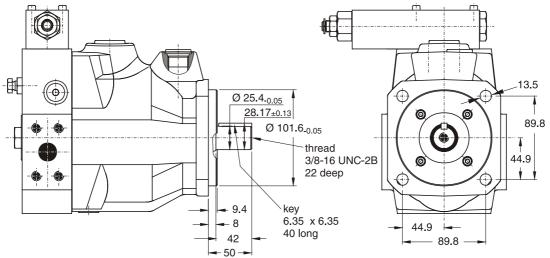
The pump shown above has **mounting option K** and **thru drive option T** (prepared for thru drive).





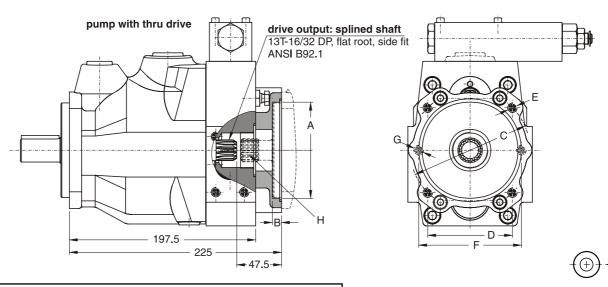


PV016 - 023, SAE version and thru drive



Shown above is mounting option D



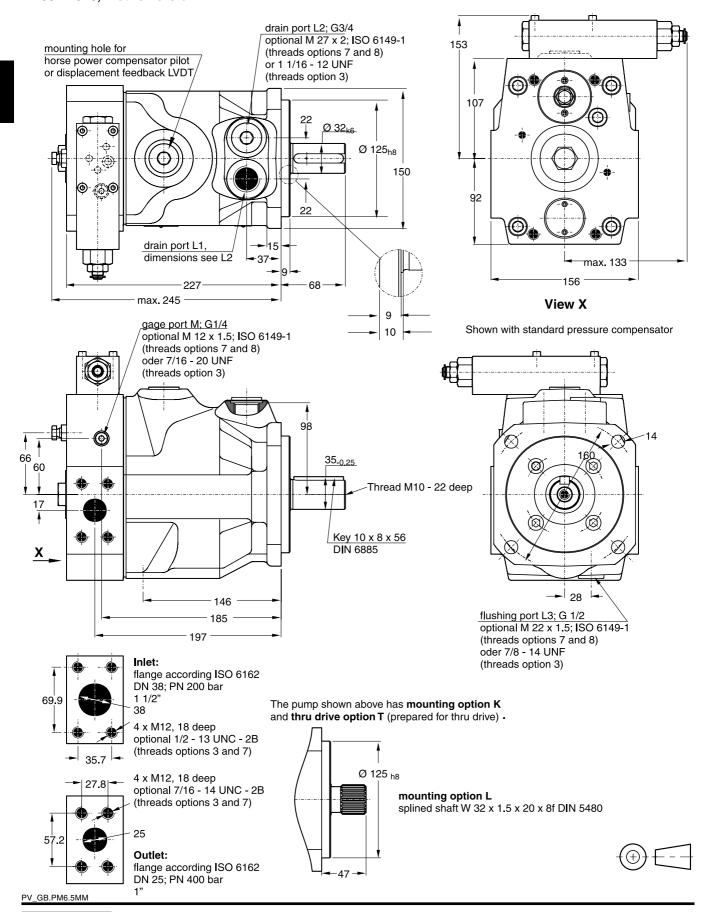


Thru s	shaft adapt	tors are av	ailable wit	h the follo	wing dime	nsions:
Α	В	С	D	E	F	G
63	10	85	-	M8	100	M8
80	10	103	-	M8	109	M10
100	10.5	125	-	M10	n. avail.	n. avail.
50.8	10	-	-	-	82	M8
82.55	10	-	-	-	106	M10
101.6	10.5	-	89.8	M12	n. avail.	n. avail.

Dimension H and available couplings see page 22. At threads options 3 and 7 the dimensions E and G are UNC - 2B threads.

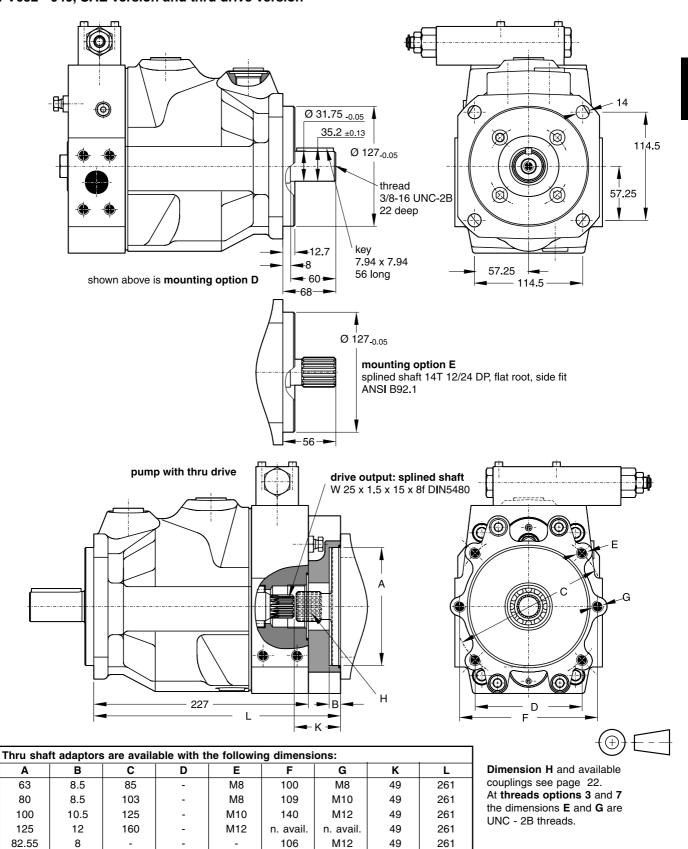


PV032 - 046, metric version





PV032 - 046, SAE version and thru drive version



PV_GB.PM6.5MM

101.6

127



11

13.5

89.8

114.5

M12

M12

146

n. avail.

M12

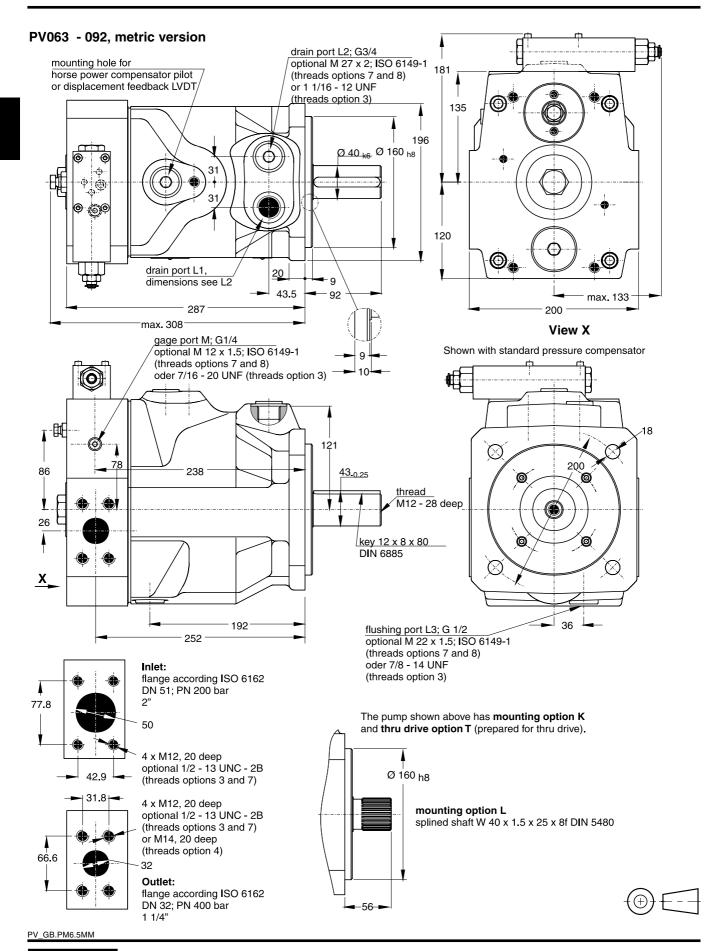
n. avail.

49

64

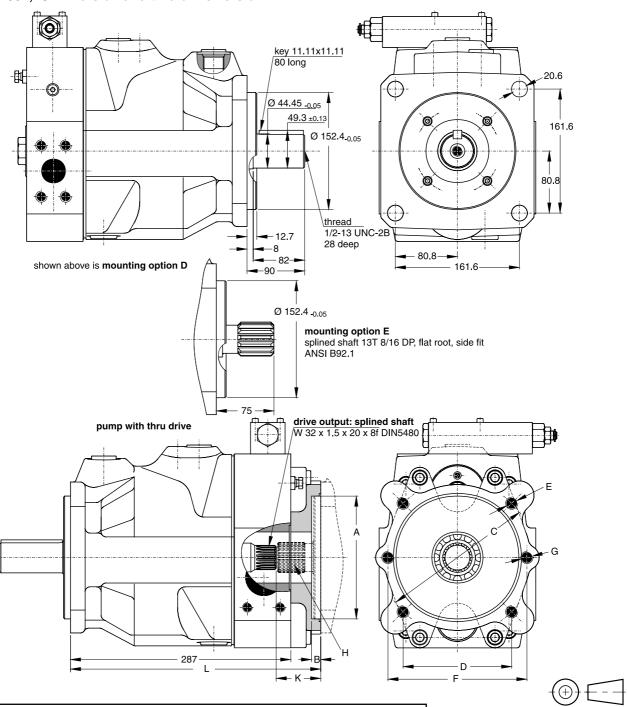
261

276





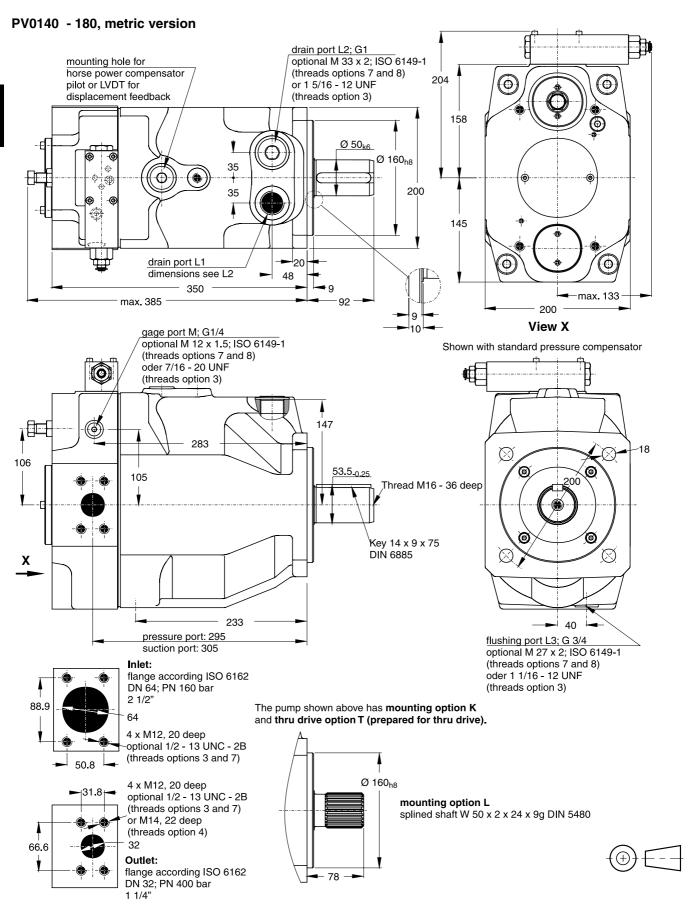
PV063 - 092, SAE version and thru drive version



	Thru s	Thru shaft adaptors are available with the following dimensions:												
Α	В	С	D	E	F	G	K	L						
63	10	85	-	M8	100	M8	58	326						
80	10	103	-	M8	109	M10	58	326						
100	12	125	-	M10	140	M12	58	326						
125	12	160	-	M12	180	M16	58	326						
160	12	200	-	M16	n. avail.	n. avail.	58	326						
82.55	10	-	-	-	106	M10	58	326						
101.6	12	-	89.8	M12	146	M12	58	326						
127	14	-	114.5	M12	181	M16	58	326						
152.4	14	-	161.6	M16	n. avail.	n. avail.	83	351						

Dimension H and available couplings see page 1-22. At threads options 3 and 7 the dimensions E and G are UNC - 2B threads.

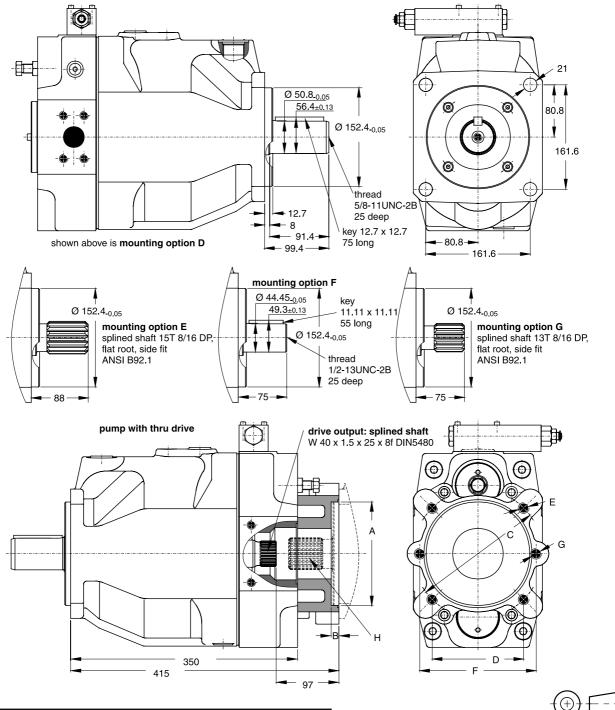








PV140 - 180, SAE version and thru drive version

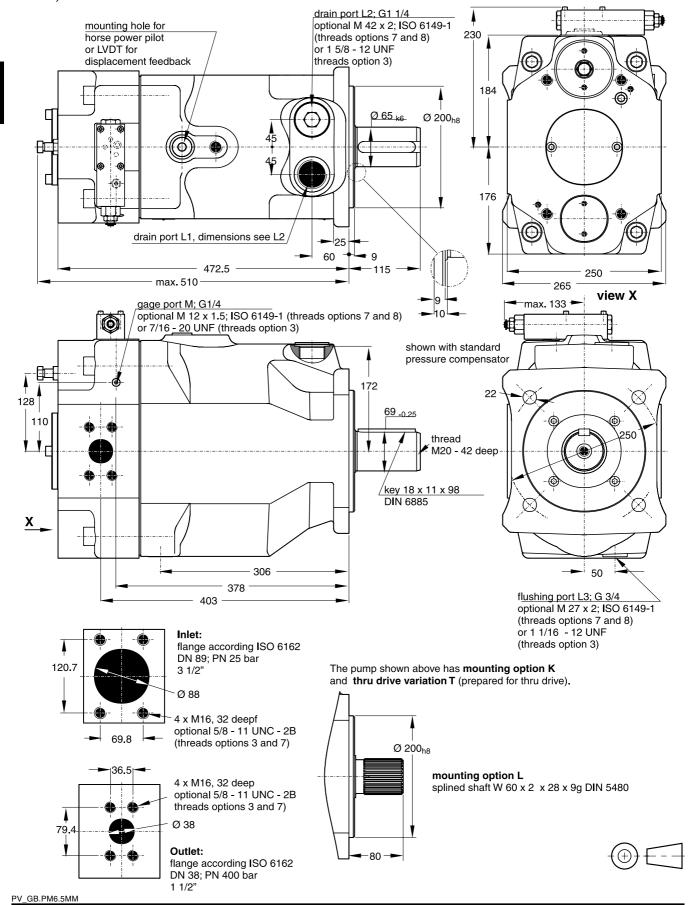


Thru s	shaft adapt	tors are av	ailable wit	h the follo	wing dime	nsions:
Α	В	С	D	E	F	G
80	10	103	-	M8	109	M10
100	12	125	-	M10	140	M12
125	12	160	-	M12	180	M16
160	12	200	-	M16	n. avail.	n. avail.
82.55	10	-	-	-	106	M10
101.6	12	-	89.8	M12	146	M12
127	14	-	114.5	M12	181	M16
152.4	14	-	161.6	M16	n. avail.	n. avail.

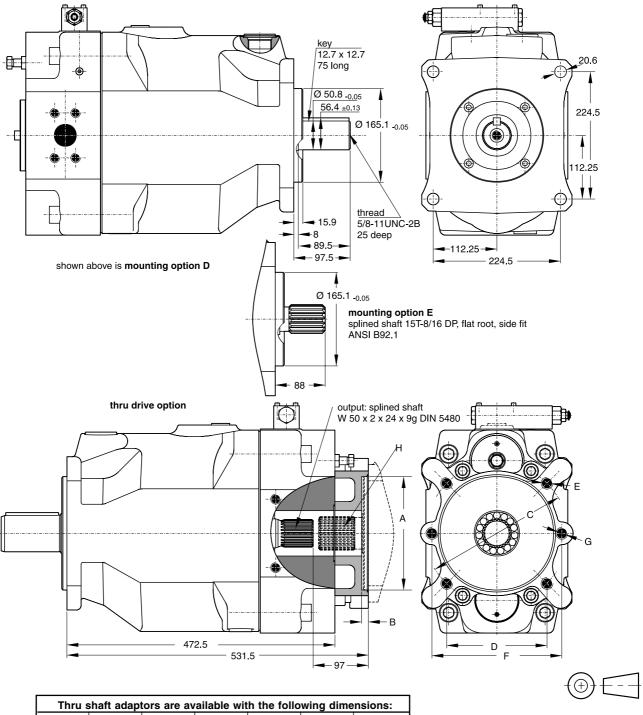
Dimension H and available couplings see page 1-22. At threads options 3 and 7 the dimensions E and G are UNC - 2B threads.



PV 270, metric version



PV 270, SAE version and thru drive version

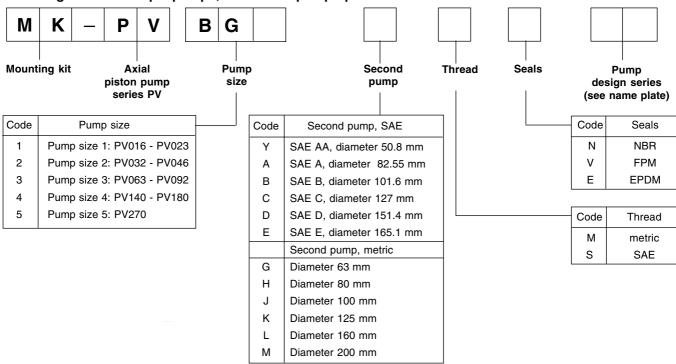


Thru s	shaft adapt	tors are av	ailable wit	h the follo	wing dime	nsions:
Α	В	С	D	Е	F	G
80	8.5	103	-	M8	109	M10
100	10.5	125	-	M10	140	M12
125	10.5	160	-	M12	180	M16
160	13.5	200	-	M16	224	M20
200	13.5	250	-	M20	n. avail.	n. avail.
82.55	8	-	-	-	106	M10
101.6	11	-	89.8	M12	146	M12
127	13.5	-	114.5	M12	181	M16
152.4	13.5	-	161.6	M16	229	M20
165.1	17	-	224.5	M20	n. avail.	n. avail.

Dimension H and available couplings see page 1-22. At threads options 3 and 7 the dimensions E and G are UNC - 2B threads.

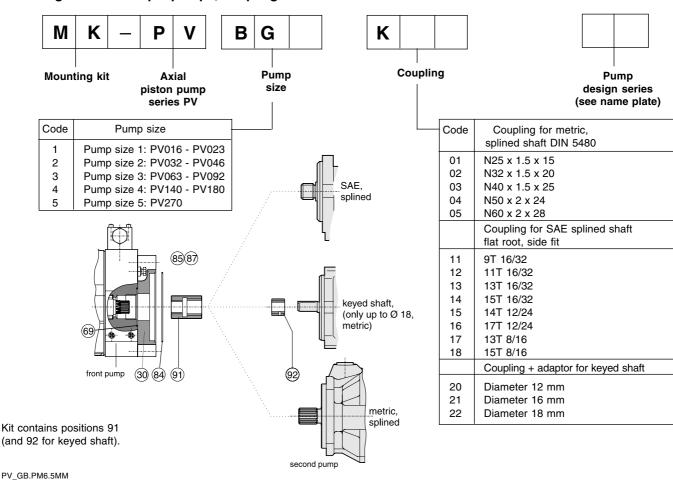


Mounting kits for multiple pumps, for second pump option

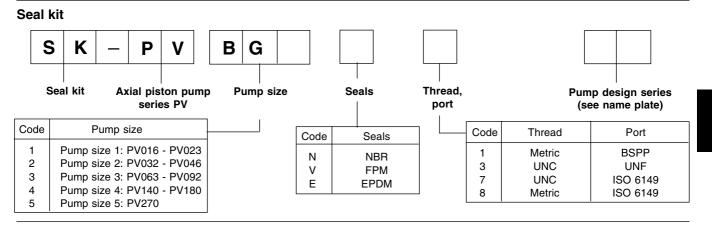


Kit contains positions 30, 69, 84 85 and 87, see drawing below.

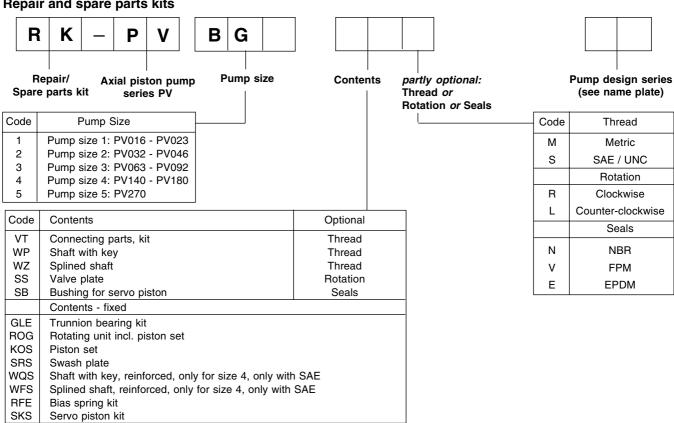
Mounting kits for multiple pumps, couplings



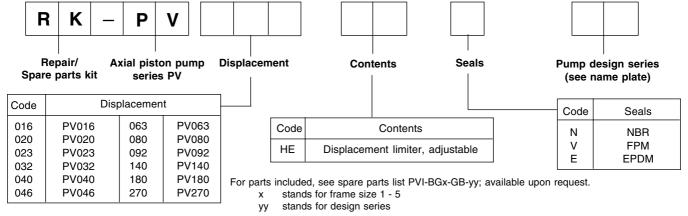
Kits



Repair and spare parts kits



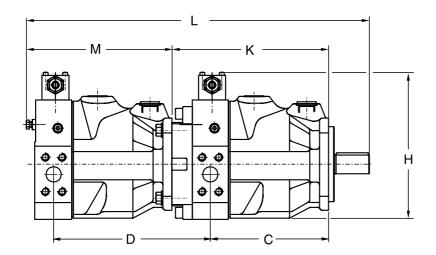
Repair and spare parts kits for adjustable displacement limiter

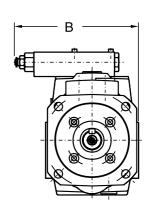




Series

Combinations PV/PV, PV/PVM (metric version)



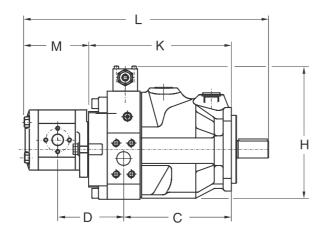


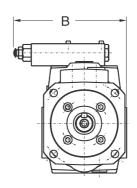
Main pump	Second pump	Interface main pump	L	В	С	D	н	K	М
PV016, 020 or 023	PV016, 020 or 023	100 B4 HW	489	196	170.5	225	220	225	212
F)/000 040 040	PV016, 020 or 023	125 B4 HW	541	208	197	235.5	245	261	212
PV032, 040 or 046	PV032, 040 or 046	125 04 1100	574	208	197	261	245	261	245
	PV016, 020 or 023		630	232	252	244.5	301	326	212
PV063, 080 or 092	PV032, 040 or 046	160 B4 HW	663	232	252	271	301	326	245
	PV063, 080 or 092		724	232	252	326	301	326	306
	PV016, 020 or 023		719	230	305	280.5	349	415	212
PV140 or 180	PV032, 040 or 046		752	230	305	307	349	415	245
	PV063, 080 or 092	160 B4 HW	813	230	305	362	349	415	306
	PV140 or 180 ¹⁾		878	230	305	415	349	415	385
	PV016, 020 or 023		860	255	403	299	406	531.5	212
PV270	PV032, 040 or 046		893	255	403	325.5	406	531.5	245
	PV063, 080 or 092	200 B4 HW	954	255	403	380.5	406	531.5	306
	PV140 or 180		1033	255	403	433.5	406	531.5	385
	PV270 ¹⁾		1134	255	403	531.5	406	531.5	510

¹⁾Combinations PV140/180 + PV140/180 and PV270 + PV270 only with splined shaft on main pump due to high torque.

Pump Combinations

Combinations PV/GP





Main pump	Second pump	Interface main p.	L*	В	С	D*	Н	К	М
PV016, 020 or 023	PGP511	100 B4 HW	420	196	170.5	124	220	225	99 -143
DV000 040 040	PGP511	125 B4 HW	472	208	197	133.5	245	261	99 -143
PV032, 040 or 046	PGP517	125 D4 11W	506	208	197	152	245	261	132 -177
PV063, 080 or 092	PGP511	160 B4 HW	561	232	252	143.5	301	326	99 -143
1 4000, 000 01 002	PGP517		595	232	252	162	301	326	132 -177
PV140 or 180	PGP511	160 B4 HW	650	230	305	179.5	349	415	99 -143
	PGP517		684	230	305	198	349	415	132 -177
PV270	PGP511	200 B4 HW	790.5	255	403	198	406	531.5	99 -143
	PGP517		824.5	255	403	216.5	406	531.5	132 -177

^{*} maximum length with largest displacement of a gear pump frame size

Thru drive, shaft load limitations

Max. transferable torque in [Nm] for the different shafts options:

Shaft code	PV016-023	PV032-046	PV063-092	PV140-180	PV270
D	300	550	1320	2000	2000
E	300	610	1218	2680	2680
F				1320	
G				1640	
К	300	570	1150	1900	2850
L	405	675	1400	2650	3980
Max. torque transmitt. cap. at shaft end	140	275	560	1100	1650

Important notice

The max. allowable torque of the individual shaft must not be exceeded. For 2-pump combinations there is no problem because PV series offers 100% thru torque. For 3-pump combinations (and more) the limit torque could be reached or exceeded.

Therefore it is necessary to calculate the torque factor and compare it with the allowed torque limit factor in the table.

Required: calculated torque factor < torque limit factor

To make the necessary calculations easier and more user friendly it is not required to calculate actual torque requirements in Nm and compare them with the shaft limitations. The table on the right shows limit factors that include material specification, safety factors and conversion factors.

The **total torque factor** is represented by the sum of the individual torque factors of all pumps in the complete pump combination.

The **torque factor of each individual pump** is calculated by multiplying the max. operating pressure p of the pump (in bar) with the max. displacement Vg of the pump (in cm³/rev).

Pump	Shaft	Torque Limit Factor
PV016-023	D E K L	17700 17700 17700 20130
PV032-046	D E K L	32680 36380 33810 40250
PV063-092	D E K L	77280 72450 67620 83720
PV140-180	D E F G K L	118400 158760 78750 97650 113400 157500
PV270	D E K L	119000 159700 170100 236250

Total torque factor of the combination

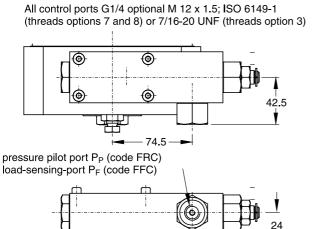
= sum of individual torque factors of all pumps

Torque factor of any pump

= p x Vg (pressure in bar x displacement in cm³/rev)



Compensators dimensions



2 x M5 - 10 deep optional 10-32 UNC (threads option 3 and 7)
31.5 Interface NG6, CETOP 03

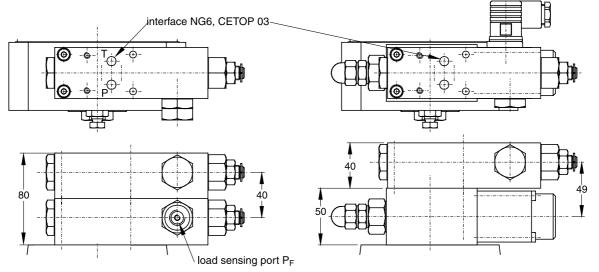
2 x M5 - 10deep in pump body (see note above)

ad-sensing-port P_F (code FF1)

load-sensing-port P_F (code FF1) (plugged for code FR1)

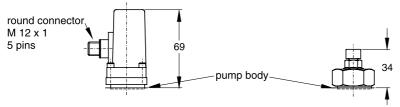
Remote pressure compensator, code FRC load-sensing compensator, code FFC

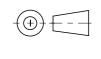
Remote pressure comp. with NG6 interface, code FR1 load-sensing comp. with NG6 interface, code FF1



2-valve-compensator, code FT1

Proportional p/Q-compensator, code FPR (for code FPV lower valve only without interface)

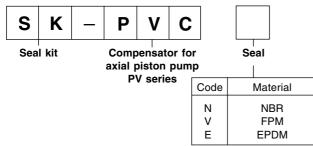




LVDT for prop. compensator

Pilot valve for horse power compensator

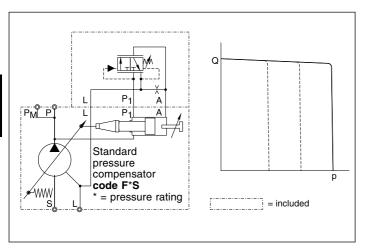
Seal kit, compensator

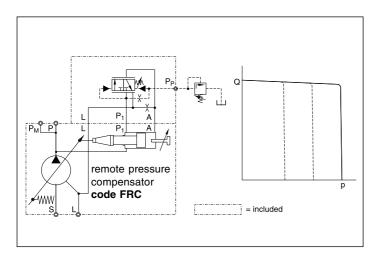


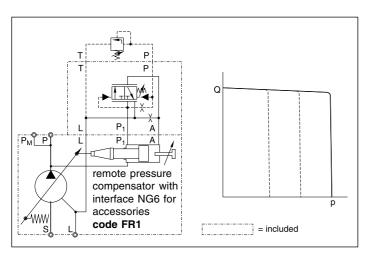
Seal kit includes all seals for all single compensator options and the seals for LVDT and horse power pilot valve. For 2-valve-compensators two seal kits are to be ordered.

Spare parts lists and ordering codes for replacement compensator valves see manual PVI-PVC-GB; available upon request









Standard pressure compensator code F*S

The standard pressure compensator adjusts the pump displacement according to the actual need of the system in order to keep the pressure constant.

As long as the system pressure at outlet port P is lower than the set pressure (set as spring preload of the compensator spring) the working port A of the compensator valve is connected to the case drain and the piston area is unloaded. Bias spring and system pressure on the annulus area keep the pump at full displacement.

When the system pressure reaches the set pressure the compensator valve spool connects port P1 to A and builds up a pressure at the servo piston resulting in a downstroking of the pump. The displacement of the pump is controlled in order to match the flow requirement of the system.

Remote pressure compensator code FRC

While at the standard pressure compensator the pressure is set directly at the compensator spring, the setting of the remote pressure compensator can be achieved by any suitable pilot pressure valve connected to pilot port $P_{\rm p}$. The pilot flow supply is internal through the valve spool.

The pilot flow is 1 - 1.5 l/min. The pilot valve can be installed remote from the pump in some distance. That allows pressure setting e. g. from the control panel of the machine. The remote pressure compensator typically responds faster and more precisely than the standard pressure compensator and is able to solve instability problems that may occur with a standard pressure compensator in critical applications.

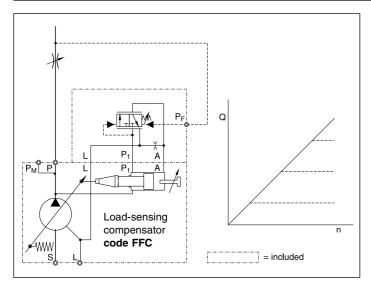
The pressure pilot valve can also be electronically controlled (proportional pressure valve) or combined with a directional control valve for low pressure standby operation.

Remote pressure compensator code FR1

Version FR1 of the remote pressure compensator provides on its top side an interface NG6, DIN 24340 (CETOP 03 at RP35H, NFPA D03).

This interface allows a direct mounting of a pilot valve. Beside manual or electrohydraulic operated valves it is also possible to mount complete multiple pressure circuits directly on the compensator body. Parker offers a variety of these compensator accessories ready to install. See pages 36 and 37 of this catalogue. All remote pressure compensators have a factory setting of 15 bar differential pressure. With this setting, the controlled pressure at the pump outlet is higher than the pressure controlled by the pilot valve.

Load-sensing compensators

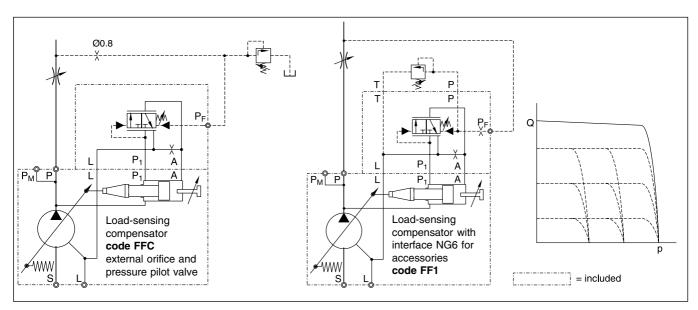


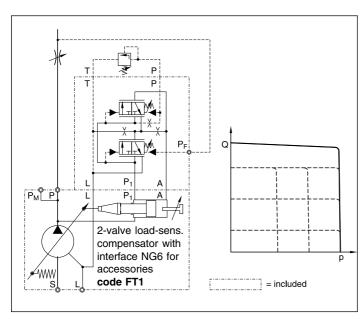
Load-sensing compensator code FFC

The load-sensing compensator has an external pilot pressure supply. Factory setting for the differential pressure is 10 bar. The input signal to the compensator is the differential pressure at a main stream resistor. A load-sensing compensator represents mainly a flow control for the pump output flow, because the compensator keeps the pressure drop at the main stream resistor constant.

A variable input speed or a varying load(-pressure) has consequently no influence on the output flow of the pump and the speed of the actuator.

By adding a pilot orifice (Ø 0.8mm) and a pressure pilot valve pressure compensation can be added to the flow control function. See the circuit diagram below, left.





Shown above is **load sensing compensator code FF1** with an NG6 interface on top of the control valve.
That allows direct mounting of a pilot valve for pressure compensation. This version includes the pilot orifice.
Due to the interaction of flow and pressure compensation this package has not the "ideal" control characteristic.
The deviation is caused by the pilot valves characteristic. If a more accurate pressure compensation is required, the **2-valve load-sensing compensator code FT1**

shown left.

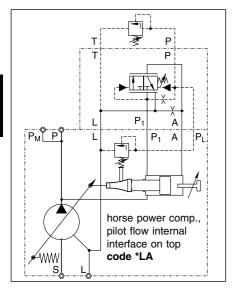
Here the interaction of the two control functions is avoided by using two separate control valves for flow and pressure compensation.

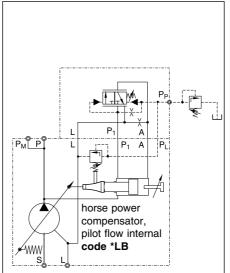
can be used. The circuit diagram of this version is

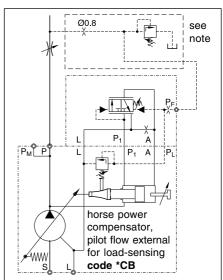
The 2-valve compensator is equipped with an interface NG6 on the compensators top side.

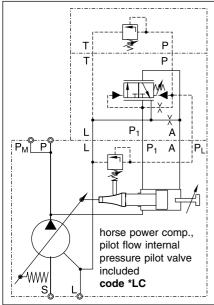


Horse power compensators









Hydraulic-mechanical horse power compensator

The hydraulic-mechanical horse power compensator consists of a modified remote pressure compensator (Code *L*) or of a modified load-sensing compensator (Code *C*) and a pilot valve. This pilot valve is integrated into the pump and is adjusted by a cam sleeve. The cam sleeve has a contour that is designed and machined for the individual displacement and the nominal horse power setting.

At a large displacement the opening pressure (given by the cam sleeve diameter) is lower than at small displacements. This makes the pump compensate along a constant horse power (torque) curve (see diagrams on opposite page).

For all nominal powers of standard electrical motors Parker offers a dedicated cam sleeve. The exchange of this cam sleeve (e. g.: to change horse power setting) can easily be done without disassembly of the pump.

On top of that an adjustment of the horse power setting can be done within certain limits by adjusting the preload of the pilot control cartridge spring . That allows an adjustment of a constant horse power setting for other than the nominal speeds (1500 min $^{-1}$) or for other horse powers.

The ordering code for the horse power option is as follows:

The first digit designates the horse power setting:

Code B = 3.0 kW etc. up to

Code 3 = 132.0 kW

The second digit designates the pilot flow source:

Code L internal pilot pressure, remote pressure function.

Code C external pilot pressure, combines horse power compensation with load-sensing compensation.

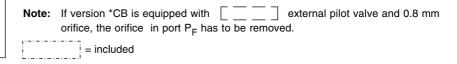
The third digit designates the possibility to adjust the overriding pressure compensation:

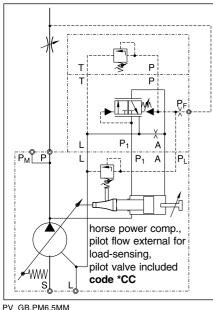
Code A comes with a top side NG6/D03 interface on the control valve to mount any suitable pilot valve or Parker pump accessories.

Code B has a threaded pilot port to connect a remote pilot valve with piping.

Code C includes a pilot valve for manual pressure adjustment. Max. setting: 350 bar.

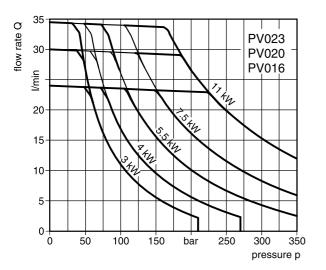
Page 31 shows the typical control characteristics and the available control sleeves for the different pump sizes and displacements.

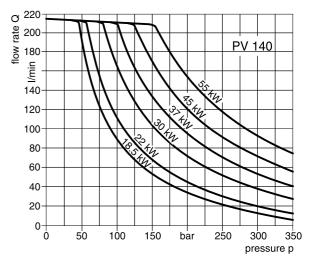


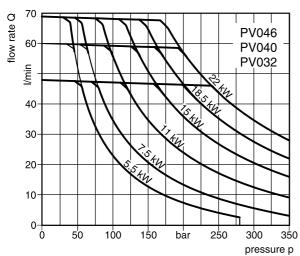


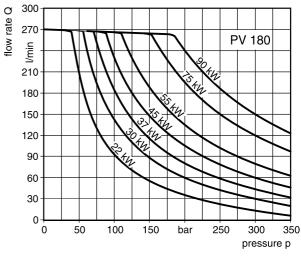
Characteristic curves, horse power compensators

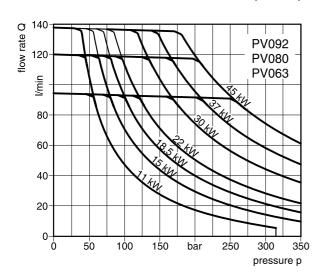
Horse power compensators, diagrams

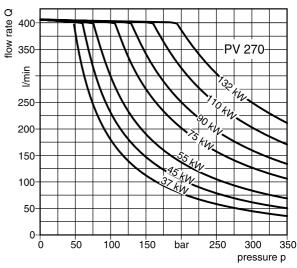












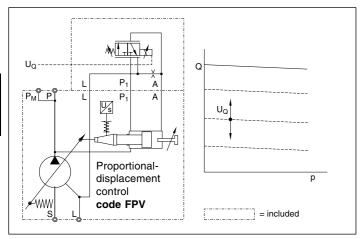
The diagrams shown are only valid for the following working conditions:

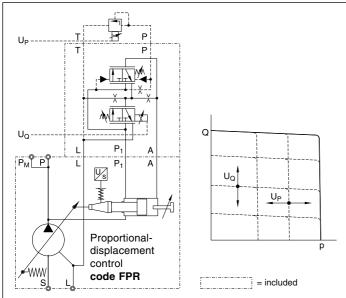
speed n = 1500 rev/min

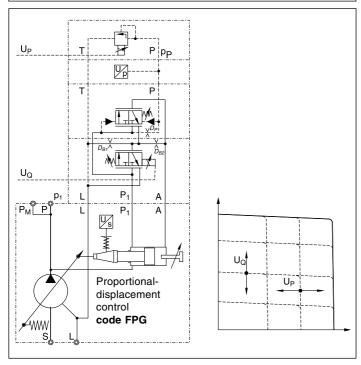
temperature: t = 50 °C

mineral oil HLP, ISO VG46 fluid viscosity $v = 46 \text{ mm}^2/\text{s} \text{ at } 40^{\circ}\text{C}$









Proportional displacement control, code FPV

The proportional displacement control allows the adjustment of the pumps output flow with an electrical input signal.

The actual displacement of the pump is monitored by an LVDT and compared with the commanded displacement in an electronic control module PQ0*-F (see opposite side). The command is given as an electrical input signal (0 - 10V or 0 resp. 4 - 20mA) from the supervising machine control. The command can also be provided by a potentiometer. The electronic control module offers a stabilized 10V source to supply the potentiometer.

The electronic control compares permanently input command and actual displacement and powers the proportional solenoid of the control valve. A deviation from the commanded displacement leads to a modulation of the input current to the solenoid. The control valve then changes the control pressure (port A) until the correct displacement is adjusted.

Version FPV of the proportional control does not provide a pressure compensation. The hydraulic circuit must be protected by a pressure relief valve.

Proportional displacement control with overriding pressure control, codes FPR, FPZ and FPG

In version FPR an additional pressure compensator valve can override the electrohydraulic displacement control. That adds pressure compensation to this control.

The compensator valve has an NG6/D03 interface on top to mount a pressure pilot valve. When using a proportional pressure pilot valve an electro-hydraulic p/Q-control can be realized. The electronic driver modules are tuned for the valve type DSAE1007P07KLAF to give best performance.

The electronic control module PQ0*-P.. (see opposite page) contains, beside the displacement control unit, also the driver electronics for the a. m. proportional pressure valves.

Using **ordering code FPZ** and specifying the desired pilot valve/compensator accessory, a complete multiple pressure adjustment can be mounted in our factory (see compensator accessories, pages 1-36 and 1-37) and the complete unit will be tested and shipped together with the pump.

With **ordering code FPG** the proportional pressure pilot valve and a pressure transducer (Parker SCP 8181 CE) are included with the pump control. In combination with control module PQ0*-Q.. a closed loop pressure control of the pump outlet pressure is available. Module PQ0*-L.. offers an electronic horse power limiter in addition to the closed loop pressure control.

Parker variable displacement pumps have a large servo piston. That leads to a extremely robust and stable pump control. On the other hand that requires high control flows (up to > 100 l/min). Parker has therefore chosen the 2-valve-p/Q-control concept, because in this case a hydraulic-mechanical compensator valve takes care of the pressure compensation of the pump. That allows a very fast pressure compensation and makes this the control unsensitive to fluid contamination. We see the 2-valve-concept as a contribution to system and pressure control safety.

For more details see manual PVI017.



The electronic modules to power the displacement control and the pressure control are snap-on type modules. They can be mounted on installation rails according to EN 50022. A card holder is not required.

The modules have potentiometers to adjust up and down ramps (ramp time up to 5s) and a min. and max. adjustment for optimum resolution and sensitivity as required by the application.

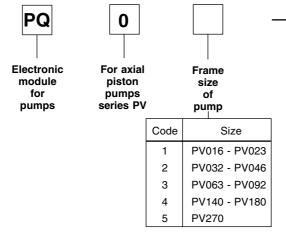
They comply with the latest legal requirements and confirm to European law. They are EMC approved and carry the CE mark.

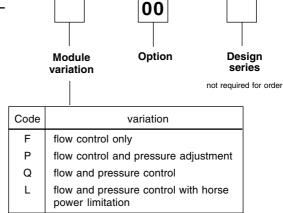


Electronic modul PQ0*-P00 to operate the p/Q-control for PV pumps



Ordering code





NOTE!

The electronic modules are not included in the pump compensator. Please order separately. More technical information on these modules can be found in chapter 10, Electronics.

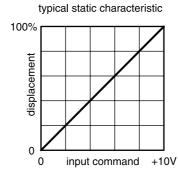
Technical Data

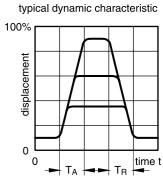
Minimum control pressure required 15 bar (at internal pressure supply = minimum system pressure) Repeatability ± 0.75 % Proportional flow compensator (solenoid): - nominal voltage 16 V - environmental temperature 50 °C 100 % - duty cycle - protection class IP54 ISO 4400 - connector Inductive position feedback (LVDT): - supply voltage 18 to 36 VDC - current requirement <50 mA - output voltage 3.5 to 11.5 VDC

- environmental temperature
- load to output signal
- connector

0 to 50 °C
> 5 kOhm
(short circuit protected)
round connector
M12 x 1.5pin

Diagrams

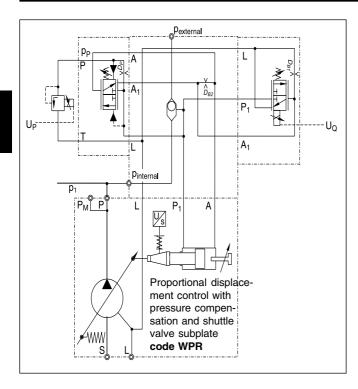




Response times

Size	TA [ms]	TR [ms]		
PV023	50	50		
PV046	70	70		
PV092	90	90		
PV180	150	150		
PV270	200	200		





Р

Proportional displacement control with

pressure compen-

sation and preload

valve

Ua

Proportional displacement control with shuttle valve subplate, code WP*

Because of the servo spring the proportional displacement control needs a minimum pump outlet pressure of 12 - 15 bar to adjust the pumps output flow according to an electrical input

If the system does not provide enough back pressure - especially at low displacement and low load - there are two options: the shuttle valve subplate and the preload valve option.

If an external auxiliary pressure is available, the control option WP* is equipped with a shuttle valve circuit according to the diagram left. At a low pump outlet pressure the pump displacement adjustment circuit is supplied by the auxiliary pressure and allows adjustment of the pump to zero flow at zero pressure. If the pump outlet pressure exceeds the auxiliary pressure, the shuttle valve shifts to internal pressure supply. Depending of the pump size and the response requirements an auxiliary power source of 20 - 30 bar and 20 - 40 l/min is recommended.

Note: pressure control is only available above the auxiliary pressure level and operation of the pump at zero flow and zero pressure requires extreme care to maintain lubrication of the rotating group.

shuttle valve subplate for proportional displacement

control

displacement control valve interface



pressure compensator interface

interface for pump mounting

Preload valve for proportional displacement control, code **PVAPVV***

An alternative solution is the use of a direct operated preload valve. The preload valve is offered in a manifold for direct mounting to the pressure port of the pump.

The opening pressure of the valve is set to approx. 20 bar and at 30 bar load pressure the valve is fully open and does cause a pressure drop of less than 1 bar.

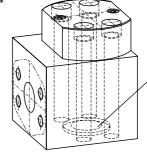
The ordering code for the preload valve is **PVAPVV***. The * stands for frame size of the pump, thread and port option and seal material option. For details see next page.

The design of the preload valve is shown below. For dimensions see installation manual PVI-017-UK (available upon request).

preload valve manifold

code PVAPVV*

outlet: flange port ISO 6162



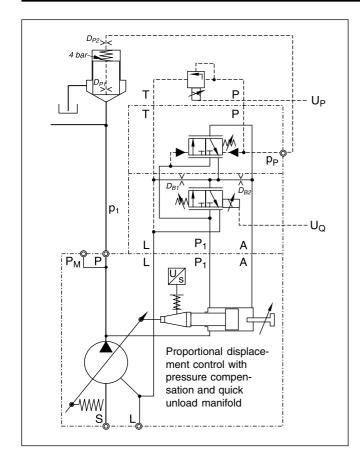
inlet: flange port ISO 6162, flanged direct to pump pressure port

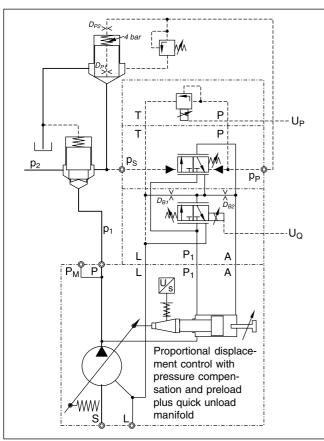
PV GB.PM6.5MM

 $P_M L P$

Ĺ







Quick pressure relief manifold for proportional pump control, code PVAPSE*

When working with a proportional pressure control on variable displacement pumps, pressure decrease can be slow. When the pump strokes to deadhead, there is no active pressure relief. To achieve a response similar to a valve controlled system, the quick unload manifold can be mounted to the pump outlet.

This manifold includes a cartridge valve with a 4 bar spring preload. The pilot pressure supply for the compensator valve is passing this cartridge valve and creates a pressure drop across the poppet. At normal working conditions this pressure drop does not exceed 3 bar and the poppet stays closed. In a dynamic response situation the pressure drop can exceed 4 bar and the cartridge actively reuces the system pressure according to the setting of the proportional pilot valve.

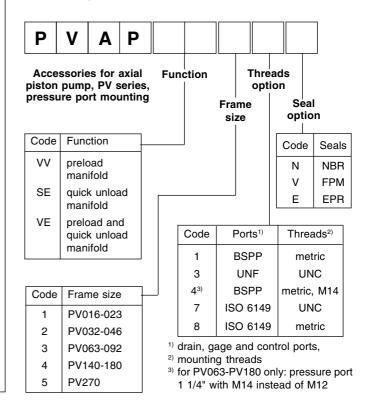
As the pilot pressure supply is fed through this manifold the compensator spool needs to be without internal orifice. Ordering code for the proportional displacement and pressure control for combination with the quick unload manifold is FPS for pressure compensation and FPT for closed loop pressure control (pressure transducer and proportional pressure pilot valve included).

Preload and quick unload manifold, code PVAPVE*

The combination of the preload and the quick unload function into one manifold can be ordered under the code PVAPVE*. This manifold is also designed for direct pump outlet mounting. To maintain a secure function under all conditions the pressure compensator requires an external sensing line connected to the system side of the preload valve (see diagram left).

The ordering code for this proportional displacement control option is FPP for pressure compensation and FPE for closed loop pressure control.

The ordering code for the function manifolds is shown below:





Hydraulic cicuit, ordering examples

Ordering Examples

1) PV pump with fast response remote pressure control, relief valve with 2 pressure stages, electrical pressure selection, nitrile seals, spindle adjustment, 24 VDC solenoid, plug to DIN 46350 accessories **fitted:**

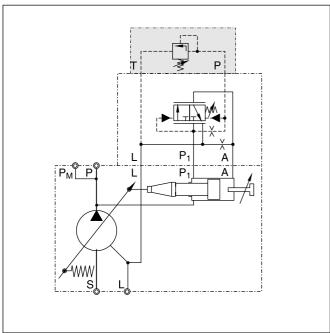
PV * * * * * * * FR**Z** PVAC2PCMNSJP

2) Same pump, accessories **not fitted:** PV * * * * * * * * FR**1** PVAC2PCMNSJP

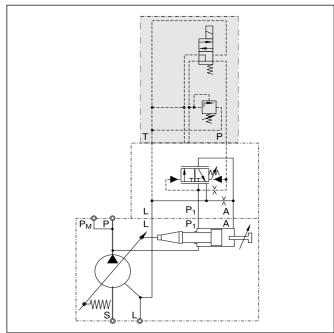
3) Usable for horsepower control and proportional volume control, too.

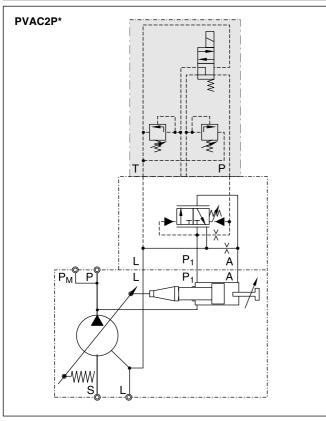
Schemactis

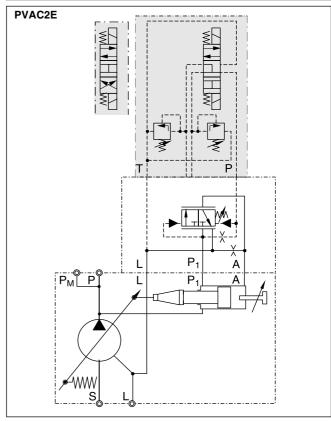
PVAC1P*



PVAC1E*

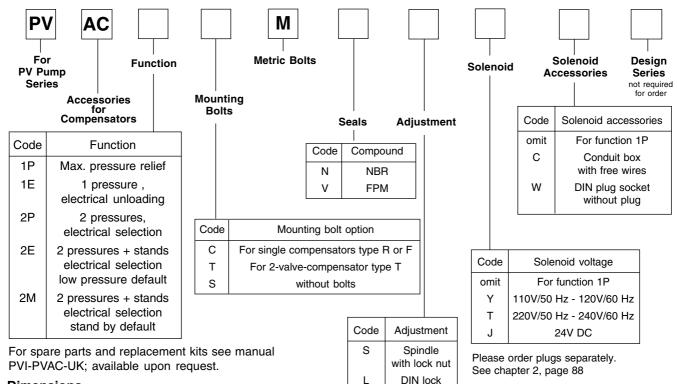




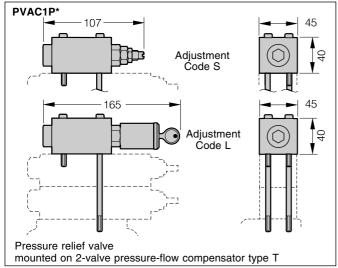


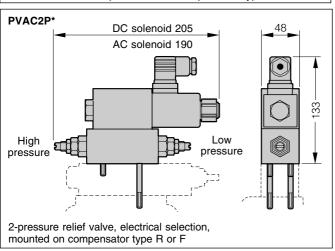


Ordering code, dimensions

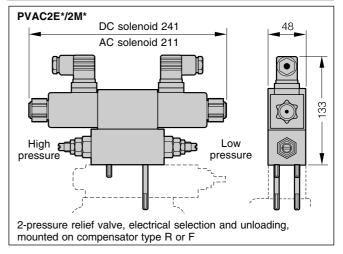


Dimensions



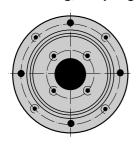


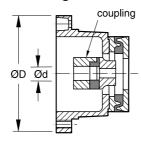
Pressure relief valve, electrical unloading, mounted on compensator type R or F

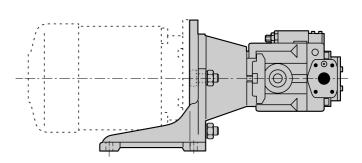




Bell housing, coupling and foot flange







Can be purchased at:

Raja

Rahmer + Jansen GmbH

Vorthstr. 1

58775 Werdohl, Germany

Tel.: (+2392) 5090, fax: (+2392) 4966

or KTR

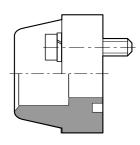
Kupplungstechnik GmbH

Rodder Damm

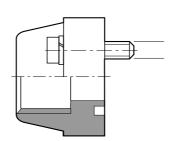
48432 Rheine, Germany

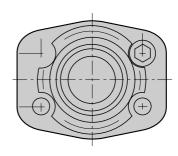
Tel.: (+5971) 798-0, fax: (+5971) 798443

Welding flange









Can be purchased at:

Parker Fluid Connectors, Tube Fittings Division

Am Metallwerk 9

33659 Bielefeld, Germany

Tel.: (+521) 4048-0, fax: (+521) 4048280

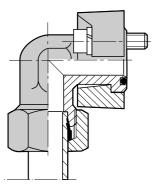
Havit Hydraulik GmbH & Co.

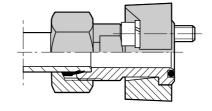
Münchner Str. 11

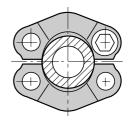
85123 Karlskron, Germany

Tel.: (+8450) 7031/7032, fax: (+8450) 7033

SAE-flange connections, pipe connection in accordance to DIN 2353







Elbow SAE-flange connection WFS

Straight SAE-flange connection GFS

Can be purchased at:

Parker Fluid Connectors, Tube Fittings Division

Am Metallwerk 9

33659 Bielefeld, Germany

Tel.: (+521) 4048-0, fax: (+521) 4048280



General installation information

Fluid recommendations

Premium quality hydraulic mineral oil fluids are recommended, like H-LP oils to DIN 51524, part 2. The viscosity range should be 25 to 50 mm²/s (cSt) at 50° C.

Normal operating viscosity range between 12 and 100 mm 2 /s (cSt). Maximum start-up viscositiy is 320 mm 2 /s. Operating temperature -10 to + 70 $^\circ$ C.

For other fluids such as phosphoric acid esters or for other operating conditions consult your Parker representative for assistance.

Seals

NBR (nitrile) seals are used for operation with hydraulic fluids based on mineral oil. For synthetic fluids, as perhaps phosphoric acid esters, Fluorocarbon seals are required. Consult your Parker representative for assistance.

Filtration

For maximum pump and system component functionability and life, the system should be protected from contamination by effective filtration.

Fluid cleanliness should be in accordance with ISO classification ISO 4406. The quality of filter elements should be in accordance with ISO standards.

Minimum requirement for filtration rate x (mm):

General hydraulic systems for satisfactory operation:

Class 19/15, to ISO 4406 $x = 25 \text{ mm} (\beta_{25} \ge 75) \text{ to ISO } 4572$

Hydraulic systems with maximised component life and functionability:

Class 16/13, to ISO 4406 $x = 10 \text{ mm} (\beta_{10} \ge 75) \text{ to ISO } 4572$

It is recommended to use return line or pressure filters. Parker Filter Division offers a wide range of these filters for all common applications and mounting styles. The use of suction filters should be avoided, especially with fast response pumps.

Bypass filtration is a good choice for best filter efficiency.

Installation and mounting

<u>Horizontal mounting</u>: Outlet port side or top. Inlet port side or bottom, drain port always uppermost.

Vertical mounting: Shaft pointing upwards.

Install pump and suction line in such a way that the maximum inlet vacuum never exceeds 0.8 bar absolute. The inlet line should be as short and as straight as possible. A short suction line cut to 45° is recommended

when the pump is mounted inside the reservoir, to improve the inlet conditions. All connections to be leak-free, as air in the suction line will cause cavitation, noise, and damage to the pump.

Drain port

Compensation may cause short-term (20 to 30 ms) flow increase, e.g. 30 l/min (PV 016 to 023), 40 l/min (PV 032 to 046), 60 l/min (PV 063 to 092),80 l/min (PV 140 to 180) and/or 120 l/min (PV270). Please consider for dimensioning.

Drain line

The drain line must lead directly to the reservoir without restriction. The drain line must not be connected to any other return line. The end of the drain line must be below the lowest fluid level in the reservoir and as far away as possible from the pump inlet line. This ensures that the pump does not empty itself when not in operation and that hot aireated oil will not be recirculated.

For the same reason, when the pump is mounted inside the reservoir, the drain line should be arranged in such a way that a siphon is created. This ensures that the pump is always filled with fluid. The drain pressure must not exceed 2 bar. Drain line length should not exceed 2 metres. Minimum diameter should be selected according to the port size and a straight low pressure fitting with maximised bore should be used.

Shaft rotation and alignment

Pump and motor shafts must be aligned within 0.25mm T.I.R. maximum. A floating coupling must be used. Bellhousings and couplings can be ordered at manufacturers listed in this catalogue. Please follow the coupling manufacturer's installation instructions. Consult your Parker representative for assistance on radial load type drives.

Start up

Prior to start up, the pump case must be filled with hydraulic fluid (use case drain port). Initial start up should be at zero pressure with an open circuit to enable the pump to prime. Pressure should only be increased once the pump has been fully primed.

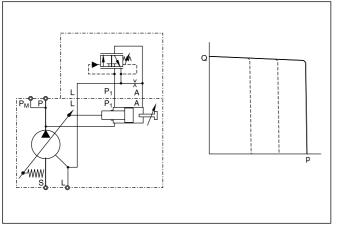
Attention: Check motor rotation direction.

Compare chapter 12.

For more details see Installation manual PVI016.



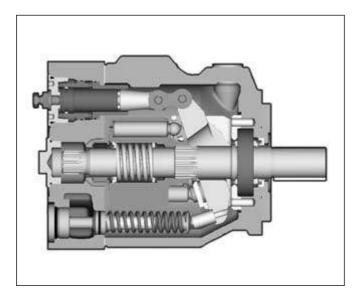
Introduction

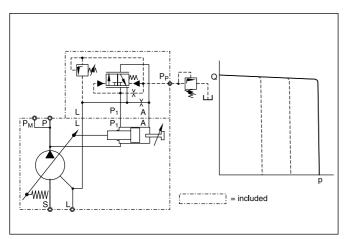


Pump with standard pressure compensator, Code MS

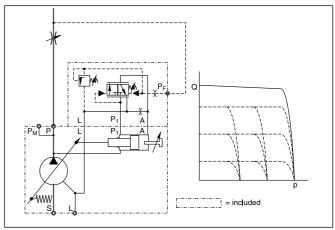
With thru drive for single and multiple pumps

Swash plate type for open circuit





Pump with remote pressure compensator RC PVM-GB.PM6.5MM



Pump with load-sensing compensator, Code FC

- Mounting interface according to VDMA-standards sheet 24560 part 1.
- Standard: 4-hole flange ISO 3019/2 (metric). Optional: 4-hole flange ISO 3019/1 (SAE).
- Large servo piston with strong bias spring achieves fast response; e.g. for PVM046 upstroke < 80 ms

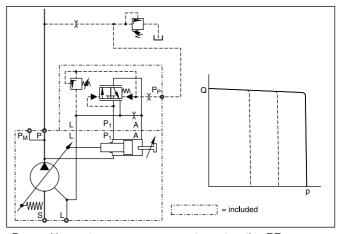
downstroke < 50 ms

Note: follow installation instructions.

- Reduced pressure peaks due to active decompression of system at downstroke.
- Also at low system pressure reliable compensator operation.

Lowest compensating pressure <10 bar

- 9 piston and new precompression technology (precompression filter volume) result in unbeaten low outlet flow pulsation.
- Rigid and FEM-optimized body design for lowest noise
- Thru drive for 100% nominal torque.
- Pump combinations (multiple pumps) of same size and model and mounting interface for basically all metric or SAE mounting interfaces.



Pump with remote pressure compensator, ext. option RE



Technical data

Displacement	from 16 to 92 cm ³ /rev	
Operating pressures		
Outlet	nominal pressure p _N 280	bar
	max. pressure p _{max.} 350	bar ¹⁾
	drain port 2	bar1)
Inlet	min. 0.8 bar (absolute)	
	max. 16 bar	
Minimum speed	300 min ⁻¹	
Mounting		
interface	4-hole flange ISO 3019/2	
	optional ISO 3019/1, SAE	
Installation	drain port as high as possible	

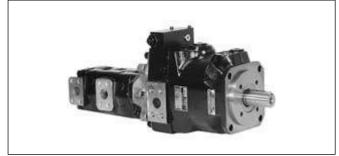
¹⁾ peak pressure only







Pump with remote press. comp.



Pump with gear pump

Pump combinations

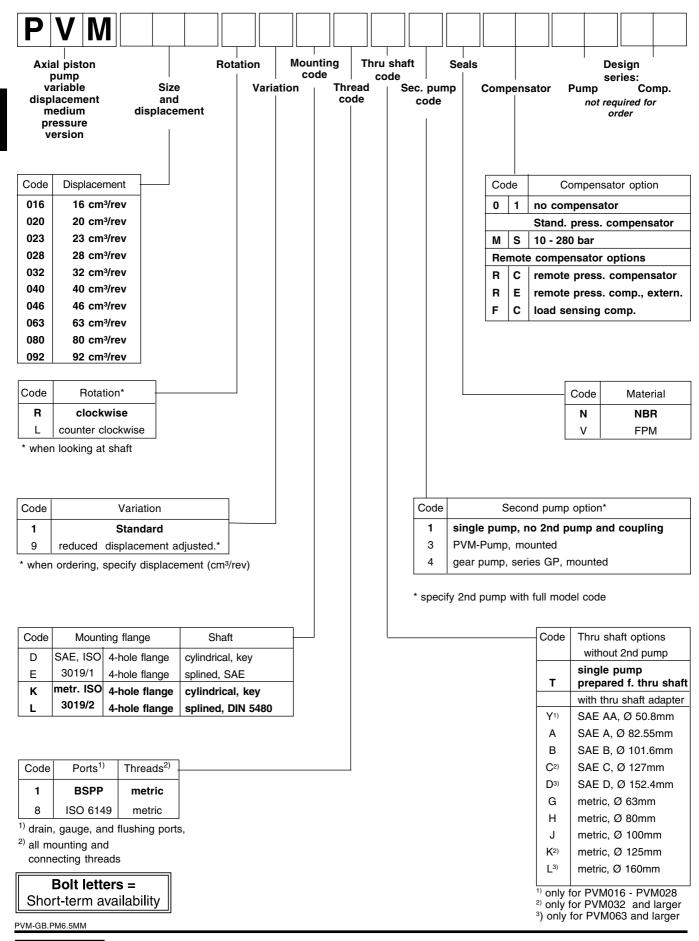
see pages 56 - 57.

Selection table and technical data

Model	Max.displacement in cm³/rev	Output flow in I/min at 1500 min ⁻¹	Input horse power in kW at 1500 min ⁻¹ and 280 bar	* Max speed in min ⁻¹	Weight in kg
PVM016	16	24	11.0		
PVM020	20	30	14.0	3000	19
PVM023	23	34.5	16		
PVM028	28	42	19.5		
PVM032	32	48	22		
PVM040	40	60	28	2800	30
PVM046	46	69	32		
PVM063	63	94.5	44	2800	
PVM080	80	120	55.5	2500	60
PVM092	92	138	64	2300	

^{*} The maximum speed ratings are shown for an inlet pressure of 1 bar (absolute) and for a fluid viscosity of $v = 30 \text{mm}^2/\text{s}$.

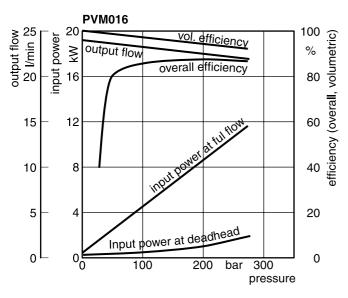
Ordering Code





Efficiency and case drain flows

Efficiency, power consumption



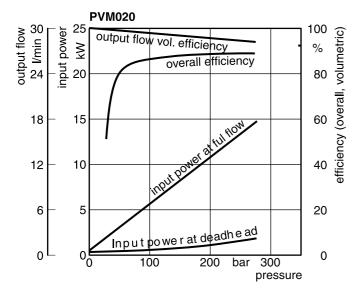
Efficiency and case drain flows PVM016, PVM020, PVM023, PVM028

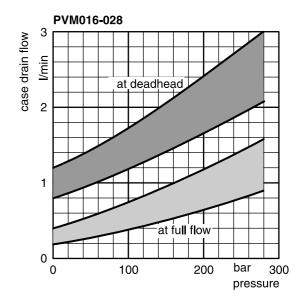
The efficiency and power graphs are measured at an input speed of $n = 1500 \text{ min}^{-1}$, a temperature of 40°C and a fluid viscosity of $46 \text{ mm}^{2}/\text{s}$.

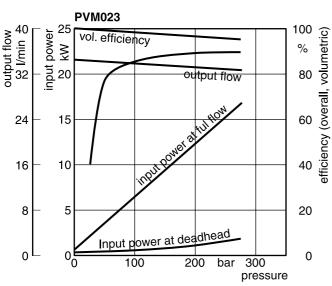
Case drain flow and compensator control flow leave via the drain port of the pump. 1 to 1.2 l/min are to be added to the values shown for the pilot operated compensators (codes RC, RE, FC) because the control flow of the pressure pilot valve also goes through the pump.

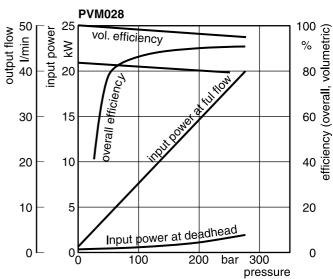
Please note: The values shown below are only valid for static operation. Under dynamic conditions and at rapid compensation of the pump the volume displaced by the servo piston also leaves the case drain port. This dynamic control flow can reach up to 40 l/min! Therefore the case drain line is to lead to the reservoir at full size and without restrictions as short and direct as possible.

Case drain flows



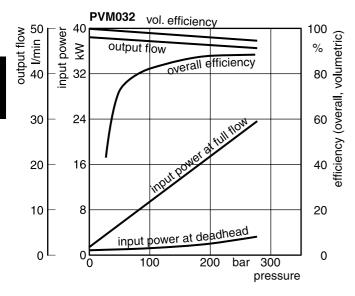




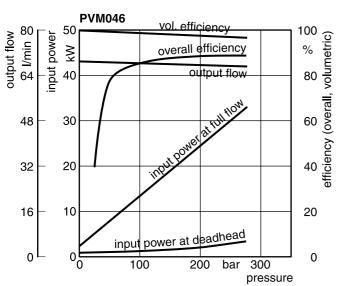


Effiency and case drain flows

Efficiency, power consumption



PVM040 60 100 output flow input power efficiency (overall, volumetric) output flow vol. efficiency uim/1 48 ≷ % overall efficiency 32 80 60 36 24 16 24 12 8 20 input power at deadhead 0 0 100 200 bar 300 pressure



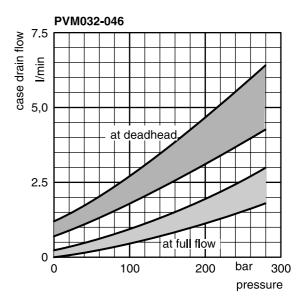
Efficiency and case drain flows PVM032, PVM040, PVM046

The efficiency and power graphs are measured at an input speed of $n = 1500 \text{ min}^{-1}$, a temperature of 40°C and a fluid viscosity of $46 \text{ mm}^{2}/\text{s}$.

Case drain flow and compensator control flow leave via the drain port of the pump. 1 to 1.2 l/min are to be added to the values shown for the pilot operated compensators (codes RC, RE, FC) because the control flow of the pressure pilot valve also goes through the pump.

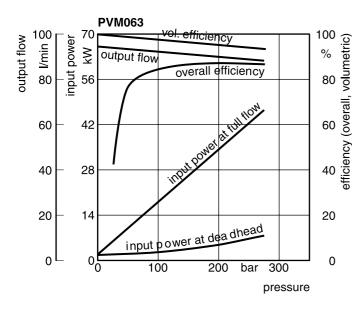
Please note: The values shown below are only valid for static operation. Under dynamic conditions and at rapid compensation of the pump the volume displaced by the servo piston also leaves the case drain port. This dynamic control flow can reach up to 50 l/min! Therefore the case drain line is to lead to the reservoir at full size and without restrictions as short and direct as possible.

Case drain flows



Effiency and case drain flows

Efficiency, power consumption



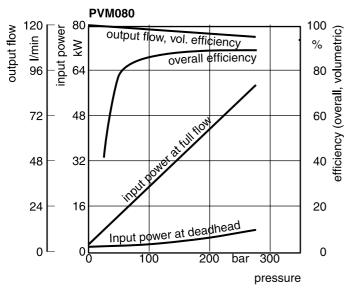
PVM063, PVM080, PVM092 The efficiency and power graphs

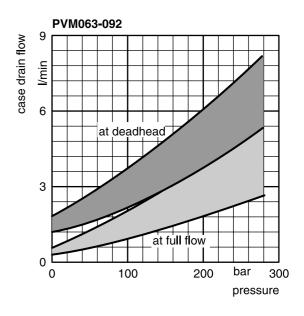
The efficiency and power graphs are measured at an input speed of $n = 1500 \text{ min}^{-1}$, a temperature of 40°C and a fluid viscosity of $46 \text{ mm}^2/\text{s}$.

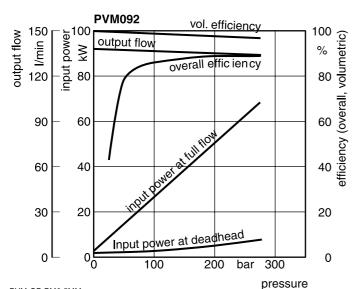
Case drain flow and compensator control flow leave via the drain port of the pump. 1 to 1.2 l/min are to be added to the values shown for the pilot operated compensators (codes RC, RE, FC) because the control flow of the pressure pilot valve also goes through the pump.

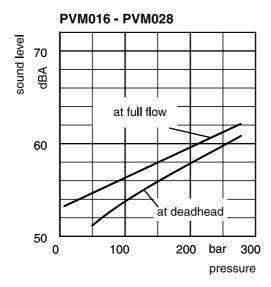
Please note: The values shown below are only valid for static operation. Under dynamic conditions and at rapid compensation of the pump the volume displaced by the servo piston also leaves the case drain port. This dynamic control flow can reach up to 70 l/min! Therefore the case drain line is to lead to the reservoir at full size and without restrictions as short and direct as possible.

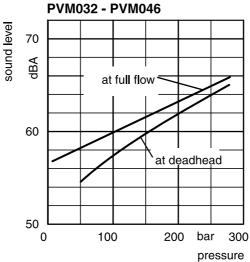
Case drain flows

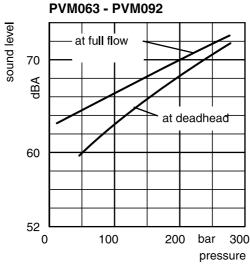












Typical sound level for single pumps, measured in unechoic chamber according to DIN 45 635, part1 and 26. microphone distance 1 m. Speed: n = 1500 min-1.

All data measured with mineral oil viscosity 30 mm²/s (cSt) at 50°C.



Noise reduction measures

Operating noise of pumps

The normal operating noise of a pump and consequently the operating noise of the entire hydraulic system is largely determined by **where** and **how** the pump is mounted and how it is connected to the downstream hydraulic system.

Also size, style and installation of the hydraulic tubing have a major influence on the overall noise emitted by a hydraulic system

Noise reduction measures

Talking about operating noise of a hydraulic pump, primary and secondary pump noise has to be taken into consideration

Primary pump noise is caused by vibrations of the pump body due to internal alternating forces stressing the body structure.

Flexible elements help to prevent pump boda vibration being transmitted to other construction elements, where possible amplification may occur. Such elements can be: Bell housing with elastic dampening flange with vulcanized labyrinth (1)

Floating and flexible coupling (2)

Damping rails (3) or silent blocks for mounting the electric motor or the foot mounting flange

Flexible tube connections (compensators) or hoses on inlet, outlet and drain port of the pump.

Exclusive use of gas tight tube fittings for inlet connections to avoid ingression of air causing cavitation and excessive noise.

Secondary pump noise is caused by vibration induced into all connected hydraulic components by the flow and pressure pulsation of the pump. This secondary noise adds typical 7 - 10 dBA to the noise of a pump measured in the sound chamber according to DIN 45 635 (see diagrams on opposite side). Therefore pipework, its mounting and the mounting of all hydraulic components like pressure filters and control elements has a major influence to the overall system noise level.

Pulsation reduction with precompression volume: The PVM is equipped with a new technology for flow ripple reduction. This method reduces the pulsation at the pump outlet by **40 - 60** %. That leads to a significant reduction of the overall system noise without additional cost and without additional components (silencers etc.). The typical reduction reaches **2 - 4 dBA**. That means: with a pump of the PVM series the secondary noise adds only some 5 - 7 dBA to the pump noise instead of the usually found 7 - 10 dBA. Figure 2 compares the measured pulsation of a system with 6 pumps of 180 cm³/rev each.

Last but not least the connection between pump and driving motor can be the cause of an unacceptably high noise emission.

Even when the mounting space is limited there are suitable means and components to reduce the noise significantly. The vibration of the pump body, created by high alternating forces in the rotating group and the pulsation of the output flow excite every part of the system connected to the pump mechanically or hydraulically.

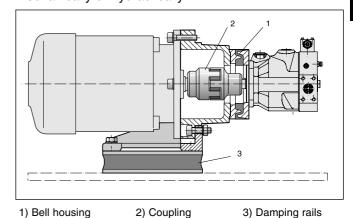


Figure 1: Components to avoid vibration transfer from the pump to the drive/installation and their position in the power unit (numbers refer to the text on the left)

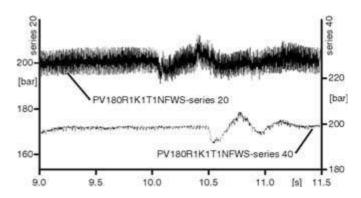


Figure 2: Comparison of the pressure pulsation in a system with 6 old PV pumps versus the same system with 6 PVplus pumps. The pulsation reduction effect of the precompression volume is evident.

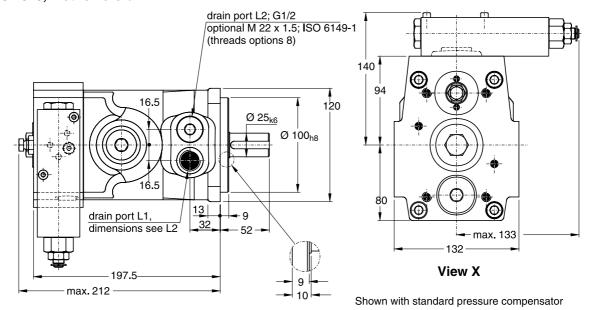
Other measures

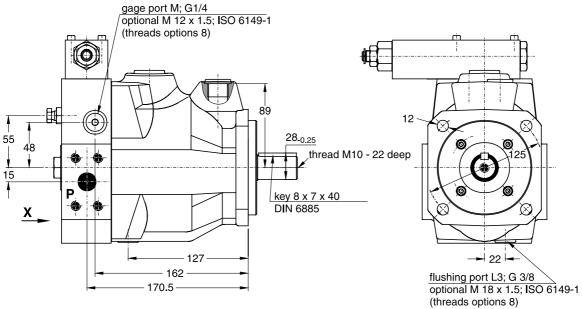
Small diameter tubes do not only cause high flow speeds, turbulences inside the tubes and cavitation in the pump, they also produce noise.

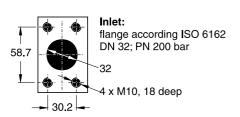
Only correctly sized connections of the largest possible diameter according to the port size of the pump should be used.

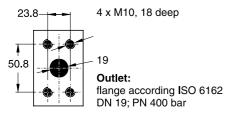


PVM016 - 028, metric version

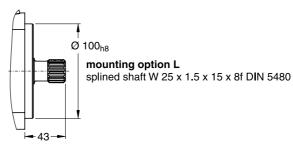








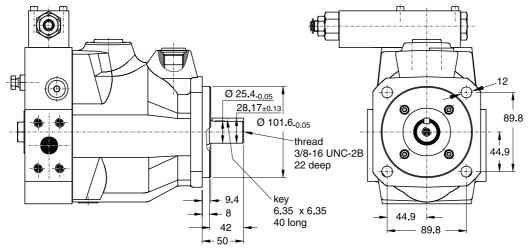
The pump shown above has **mounting option K** and **thru drive option T** (prepared for thru drive).



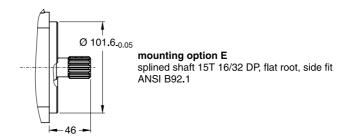


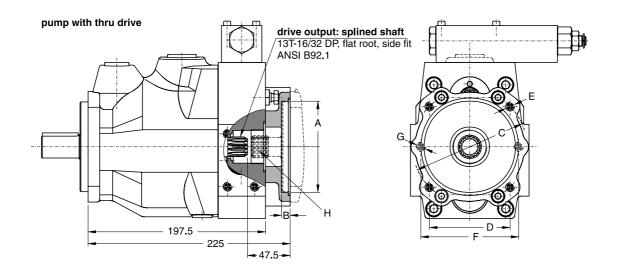


PVM016 - 028, SAE version and thru drive



Shown above is mounting option D



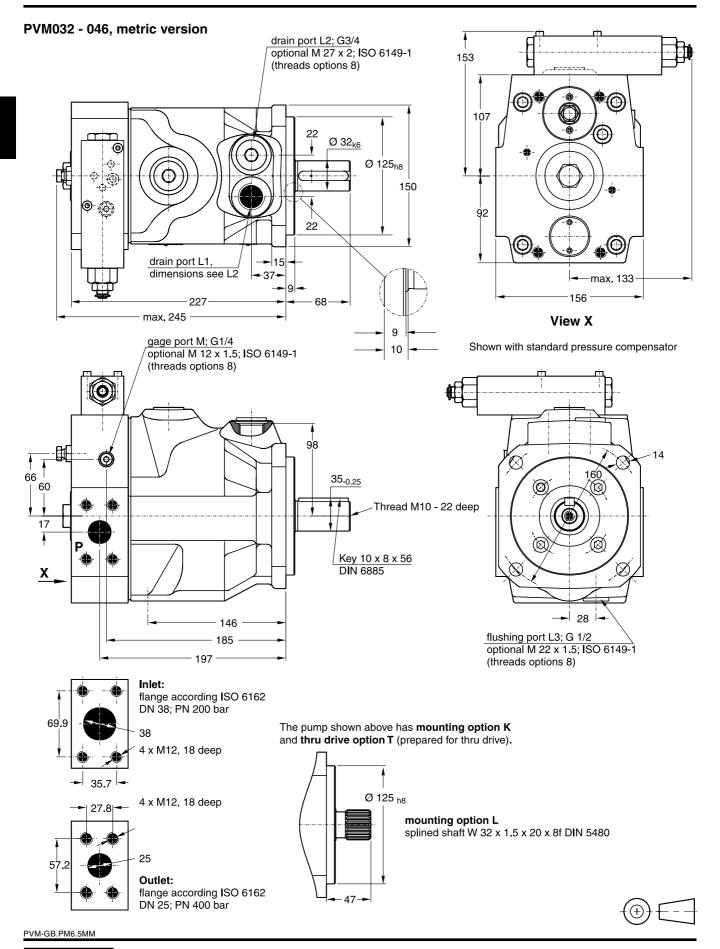


Thru	Thru shaft adapters are available in the following dimensions:								
Α	В	С	D	E	F	G			
63	10	85	-	M8	100	M8			
80	10	103	-	M8	109	M10			
100	12	125	-	M10	n. avail.	n. avail.			
50.8	10	-	-	-	82	M8			
82.55	10	-	-	-	106	M10			
101.6	12	-	89.8	M10	n. avail.	n. avail.			

 $\mbox{\bf Dimension H}$ and available coupling sleeves, see page 1-54.

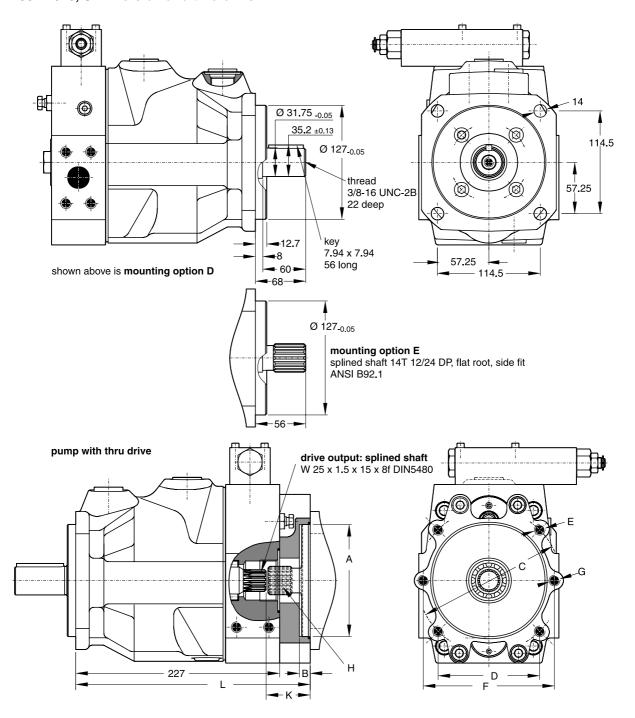








PVM032 - 046, SAE version and thru drive

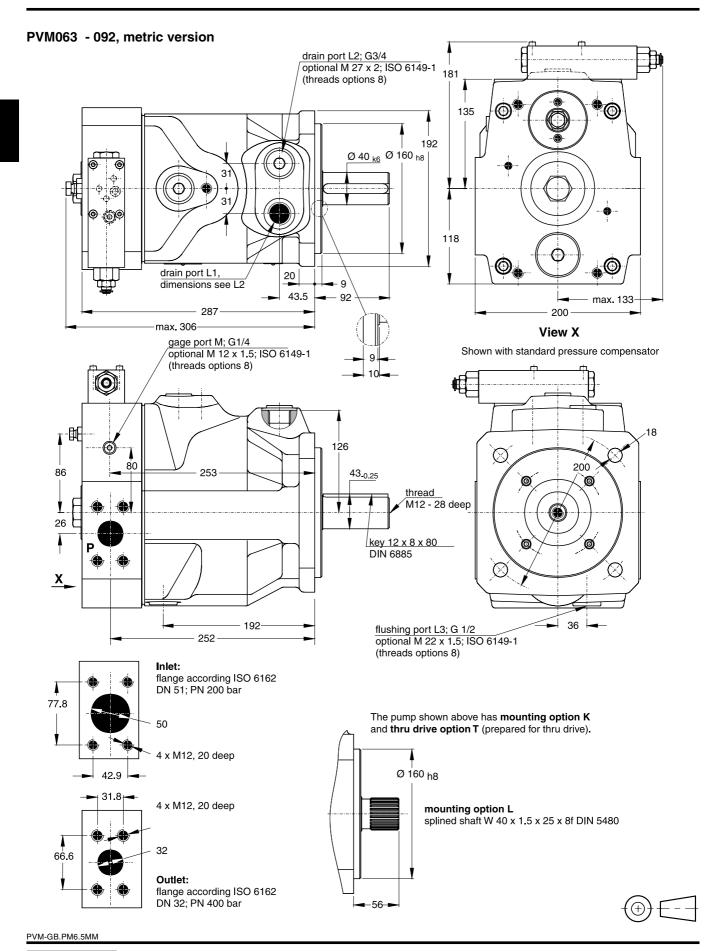


	Thru shaft adapters are available in the following dimensions:									
Α	В	С	D	E	F	G	K	L		
63	8.5	85	-	M8	100	M8	49	261		
80	8.5	103	-	M8	109	M10	49	261		
100	10.5	125	-	M10	140	M12	49	261		
125	12	160	-	M12	n. avail.	n. avail.	49	261		
82.55	8	-	-	-	106	M10	49	261		
101.6	11	-	89.8	M10	146	M12	49	261		
127	13.5	-	114.5	M12	n. avail.	n. avail.	64	276		

Dimension H and available coupling sleeves, see pages 1- 54.

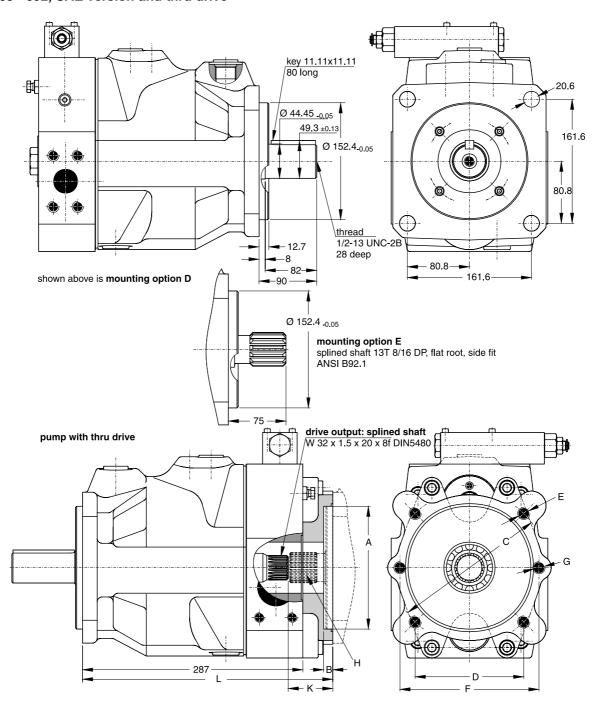








PVM063 - 092, SAE version and thru drive



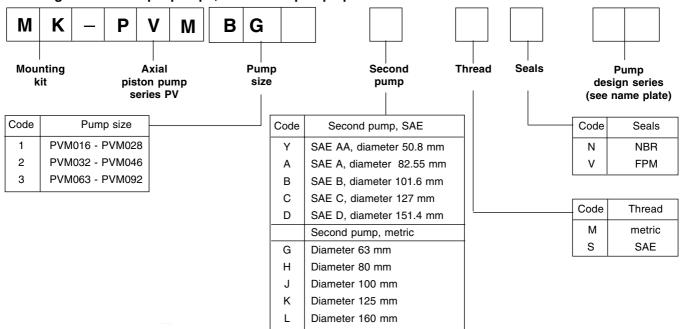
	Thru shaft adapters are available in the following dimensions:									
Α	В	С	D	E	F	G	K	L		
63	10	85	-	M8	100	M8	58	326		
80	10	103	-	M8	109	M10	58	326		
100	12	125	-	M10	140	M12	58	326		
125	12	160	-	M12	180	M16	58	326		
160	12	200	-	M16	n. avail.	n. avail.	58	326		
82.55	10	-	-	-	106	M10	58	326		
101.6	12	-	89.8	M12	146	M12	58	326		
127	14	-	114.5	M12	181	M16	58	326		
152.4	14	-	161.6	M20	n. avail.	n. avail.	78	360		

Dimension H and available coupling sleeves, see pages 1-54.



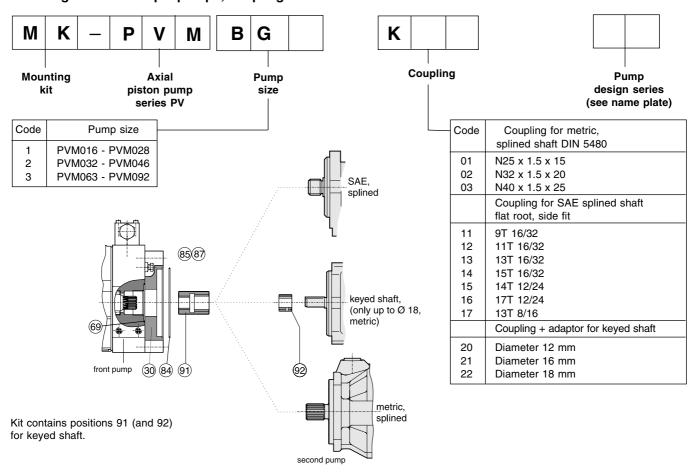


Mounting kits for multiple pumps, for second pump option



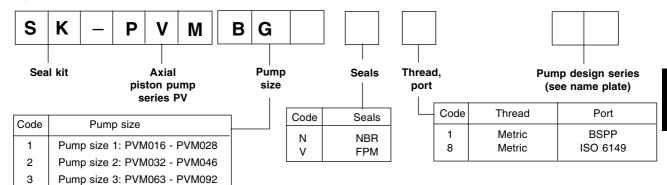
Kit contains positions 30, 69, 84 85 and 87, see drawing below

Mounting kits for multiple pumps, couplings

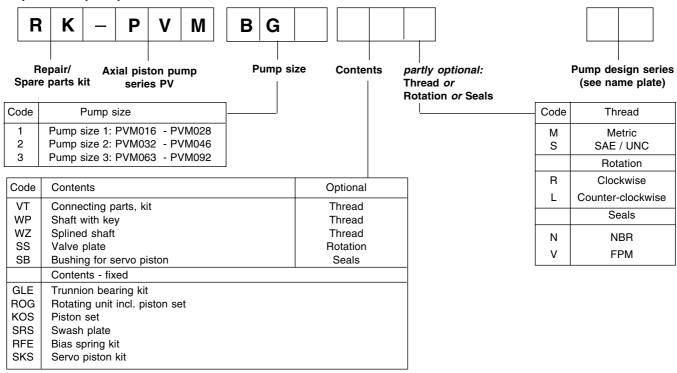


Kits

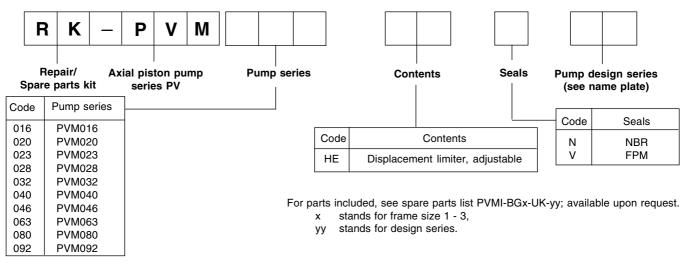




Repair and spare parts kits



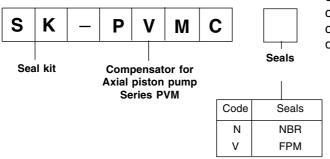
Repair and spare parts kits for adjustable displacement limiter





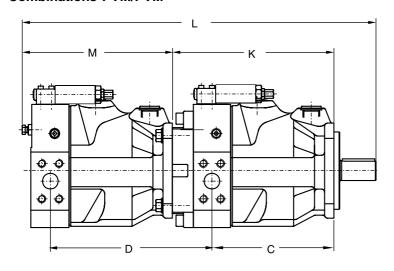
Pump Combinations

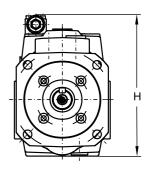
Seal-kit Compensator



The seal kit contains all the seals required for single compensators, LVDT and feed valves. For 2-valve compensators, please order 2 seal kits. Details on spare compensator lists and ordering codes are given in documentation PVMI-PVC-UK.

Combinations PVM/PVM



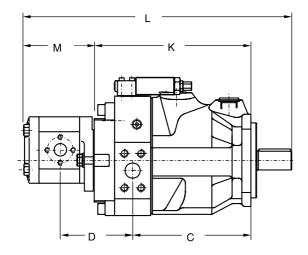


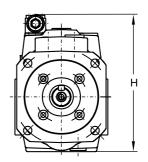
Main pump	Second pump	Interface main pump	L	С	D	Н	К	М
DV/14040 000 000	F)////// 000 000							
PVM016, 020, 023	PVM016, 020, 023							
or 028	or 028	100 B4 HW	489	170.5	225	206	225	212
DV/14000 040 040	D) (1 40 40 000 000							
PVM032, 040 or 046	PVM016, 020, 023							
	or 028	125 B4 HW	541	197	235.5	231	261	212
	PVM032, 040 or 046		574	197	261	231	261	245
PVM063, 080 or 092	PVM016, 020, 023							
	or 028	160 B4 HW	630	252	244.5	285	326	212
	PVM032, 040 or 046		663	252	271	285	326	245
	PVM063, 080 or 092		724	252	326	285	326	306



Pump Combinations

Combinations PVM/PGP





Main pump	Second pump	Interface main pump	L*	С	D*	н	К	М
PVM016, 020, 023 or 028 PVM032, 040 or 046	PGP505 PGP505	100 B4 HW 125 B4 HW	420 472	170.5 197	124 133.5	206 231	225 261	99 - 146 99 - 143
PVM063, 080 or 092	PGP511 PGP505	160 B4 HW	506 561	197 252	152 143.5	231 285	261 326	132 - 177 99 - 143
	PGP511		595	252	162	285	326	132 - 177

^{*} maximum length with largest displacement of a gear pump frame size

Max. transferable torque in Nm for the different shafts options:

Shaft code	PVM016-028	PVM032-046	PVM063-092
D	300	550	1320
E	300	610	1218
к	300 570		1150
L	405	675	1400
Max. torque transmitt. cap. at shaft end	170	275	560

Important notice

The max. allowable torque of the individual shaft must not be exceeded. For 2-pump combinations there is no problem because PV series offers 100% thru torque. For 3-pump combinations (and more) the limit torque can be reached or exceeded.

Therefore it is necessary to calculate the torque factor and compare it with the allowed torque limit factor in the table.

Required:	calculated			
	torque factor < torque limit factor			

To make the necessary calculations easier and more user friendly it is not required to calculate actual torque requirements in Nm and compare them with the shaft limitations. The table on the right shows limit factors that include material specification, safety factors and conversion factors.

Pump	Shaft	Torque limit factor
PVM016-028	D E K L	17700 17700 17700 20130
PVM032-046	D E K L	32680 36380 33810 40250
PVM063-092	D E K L	77280 72450 67620 83720

The **total torque factor** is represented by the sum of the individual torque factors of all pumps in the complete pump combination.

Total torque factor of the combination = sum of individual torque factors of all pumps

The **torque factor of each individual pump** is calculated by multiplying the max. operating pressure p of the pump (in bar) with the max. displacement Vg (in cm³/rev).

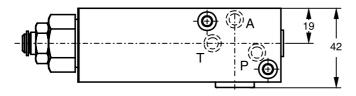
Torque factor of any pump

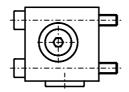
= p x Vg (pressure in bar x displacement in cm³/rev)

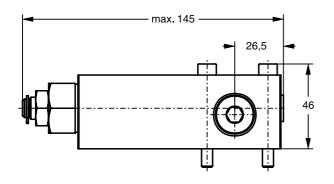


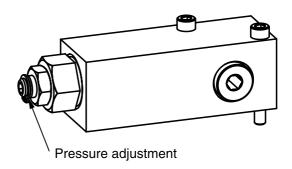
Compensator Dimensions

Dimensions standard pressure compensator, code ...MS

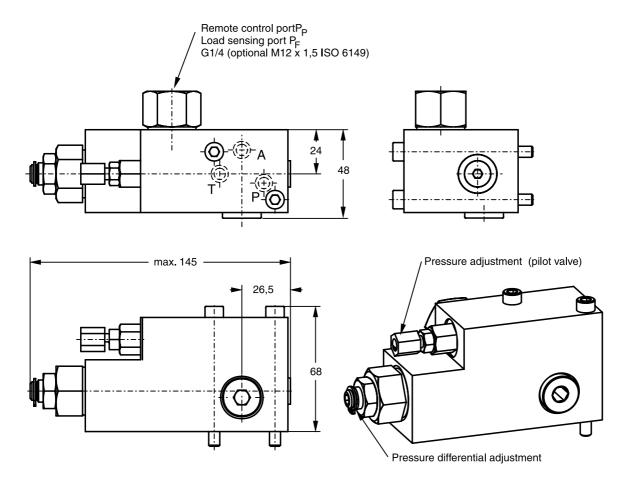






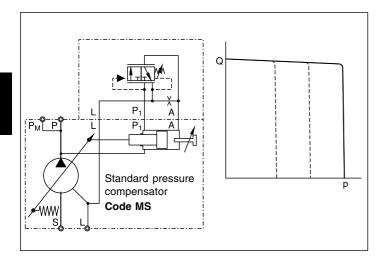


Dimensions remote pressure and load sensing compensators, codes ...RC and ...FC





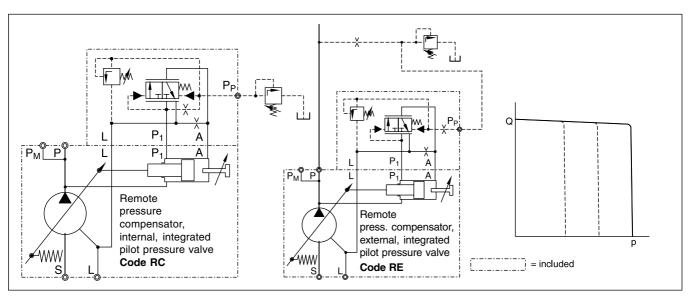
Pressure compensators

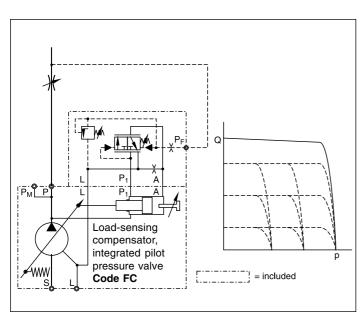


Standard pressure compensator Code MS

In the standard pressure compensator, the pressure on the pump outlet port is compared with the reference pressure set in the compensator valve spool. As long as this reference pressure is not reached, the control valve connects the large adjustment piston surface with the pump, and the compensator valve spool maintains the pump at full displacement.

If the reference pressure is reached and the pilot valve is moved against the compensator valve spool , port P1 is connected to A, the large adjustment piston surface is impinged with pressure, and the pump is limited. The displacement of the pump is controlled in order to match the flow requirement of the system. If the requirement is returned to zero, the pump then only delivers the quantity necessary to cover its own lubrication and pilot volume requirement.





Remote pressure compensator codes RC and RE

The system pressure is compared with a pilot pressure on the on the control valve for the remote pressure compensators. This pilot pressure can be influenced both by a pilot pressure valve integrated in the compensator and externally cased. The pilot pressure supply is governed internally by the compensator for code RC, for code RE externally via the pilot port Pp. The pilot pressure flow is approx. 1-1.2 l/min. The pilot valve can also be installed some distance away from the pump so that a pressure setting can be achieved from the control panel of the machine.

Load-sensing compensator code FC

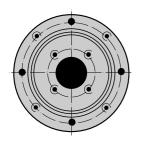
The pilot oil for the compensator is tapped behind a main stream resistor for load-sensing compensators. Therefore, the compensator keeps the pressure drop at the main stream resistor constant, beneath the reference pressure (set on the integrated pilot valve). Thus there is a flow control for the pump output flow. When the reference pressure is reached, the pilot valve opens and the pump is limited.

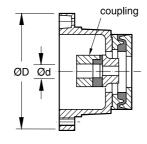
The mutual influence of flow and pressure compensation leads to the represented, strongly rounded characteristic curves.

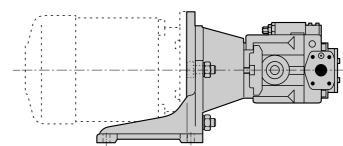


Accessories

Bell housing, coupling and foot flange







Can be purchased at:

Raja

Rahmer + Jansen GmbH Vorthstr. 1

58775 Werdohl, Germany

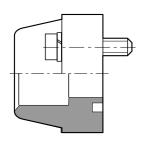
Tel.: (+2392) 5090, fax: (+2392) 4966

KTR

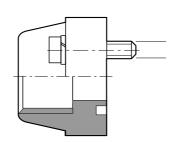
Kupplungstechnik GmbH Rodder Damm 48432 Rheine, Germany

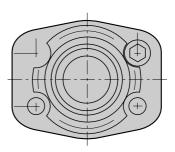
Tel.: (+5971) 798-0, fax: (+5971) 798443

Welding flange









Can be purchased at:

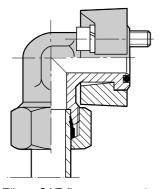
Havit Hydraulik GmbH & Co.

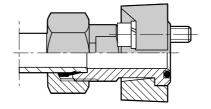
Münchner Str. 11

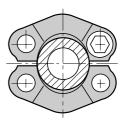
85123 Karlskron, Germany

Tel.: (+8450) 7031/7032, fax: (+8450) 7033

SAE-flange connections, pipe connection in accordance to DIN 2353







Elbow SAE-flange connection WFS

Straight SAE-flange connection GFS

Can be purchased at:

Parker Fluid Connectors, Tube Fittings Division

Am Metallwerk 9

33659 Bielefeld, Germany

Tel.: (+521) 4048-0, fax: (+521) 4048280



General Installation Information

Fluid recommendations

Premium quality hydraulic mineral oil fluids are recommended, like H-LP oils to DIN 51524, part 2. The viscosity range should be 25 to 50 mm²/s (cSt) at 50° C.Normal operating viscosity range between 12 and 100 mm²/s (cSt). Maximum start-up viscositiy is 320 mm²/s. Operating temperature -10 to + 70° C. For other fluids such as phosphoric acid esters or for other operating conditions consult your Parker representative for assistance.

Seals

NBR (nitrile) seals are used for operation with hydraulic fluids based on mineral oil. For synthetic fluids, such as phosphoric acid esters, fluorocarbon seals are required. Consult your Parker representative for assistance.

Filtration

For maximum pump and system component functionability and life, the system should be protected from contamination by effective filtration. Fluid cleanliness should be in accordance with ISO classification ISO 4406. The quality of filter elements should be in accordance with ISO standards.

Minimum requirement for filtration rate x (µm):

General hydraulic systems for satisfactory operation:

Class 19/15, to ISO 4406

 $x = 25 \mu m (\beta_{25} \ge 75) \text{ to ISO } 4572$

Hydraulic systems with maximised component life and functionability:

Class 16/13, to ISO 4406

 $x = 10 \ \mu m \ (\beta_{10} \ge 75) \ to \ ISO \ 4572$

It is recommended to use return line or pressure filters. Parker Filter Division offers a wide range of these filters for all common applications and mounting styles. The use of suction filters should be avoided, especially with fast response pumps.

Installation and mounting

<u>Horizontal mounting:</u> Outlet port side or top, inlet port side or bottom. Drain port always uppermost or rotated 90 degrees on the axis.

Vertical mounting: Shaft pointing upwards

Inlet (suction side): Install pump and suction line in such a way that the maximum inlet vacuum never exceeds 0.8 bar absolute. The inlet line should be as short and as straight as possible. A short suction line cut to 45° is recommended when the pump is mounted inside the reservoir, to improve the inlet conditions. All connections to be leak-free, as air in the suction line will cause cavitation, noise, and damage to the pump.

Drain line

The drain line must lead directly to the reservoir without restriction. The drain line must not be connected to any other return line. The end of the drain line must be below the lowest fluid level in the reservoir and as far away as possible from the pump inlet line. This ensures that the pump does not empty itself when not in operation and that hot airheated oil will not be recirculated.

For the same reason, when the pump is mounted inside the reservoir, the drain line should be arranged in such a way that a siphon is created. This ensures that the pump is always filled with fluid. The drain pressure must not exceed 1 bar. Drain line length should not exceed 2 metres. Minimum diameter should be selected according to the port size and a straight low pressure fitting with maximised bore should be used.

Shaft rotation and alignment

Pump and motor shafts must be aligned within 0.25mm T.I.R. maximum. A floating coupling must be used. Bell housings and couplings can be ordered at manufacturers listed in this catalogue. Please follow the coupling manufacturer's installation instructions. Consult your Parker representative for assistance on radial load type drives. An axial load on the pump shaft is not permitted.

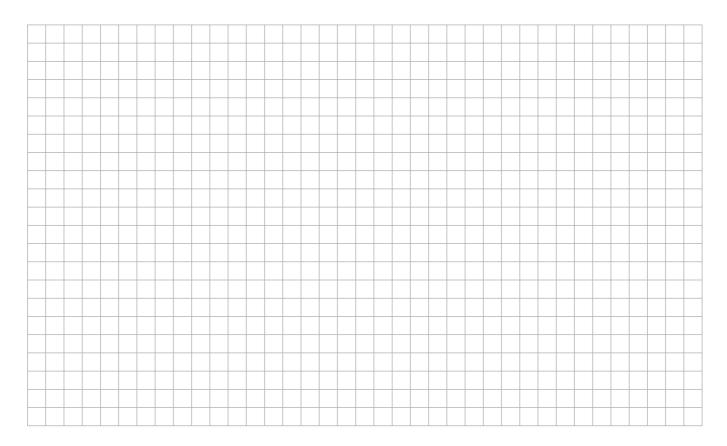
Start up

Prior to start up, as well as after long stand-still periods when it is possible that the pump body could have been emptied, the pump case must be filled with hydraulic fluid (use case drain port). Initial start up should be at zero pressure with an open circuit to enable the pump to prime. Pressure should only be increased once the pump has been fully primed. A quick on and off switching (inching mode) makes priming easier and enables quick filling of the displacement space in the pump.

Attention: Check motor rotation direction. See also the statements on hydraulic fluids in Chapter 12.

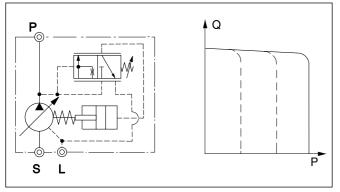
Catalogue	HY11-2500/Uk
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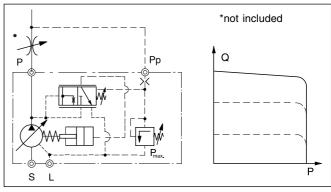




Introduction



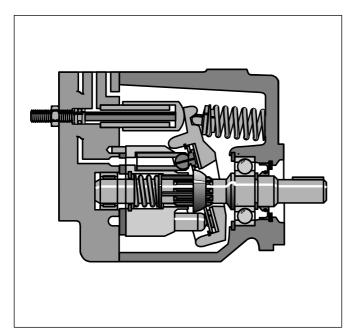
Pump with standard compensator, code: "omit"



Pump with load sensing, code: "A"

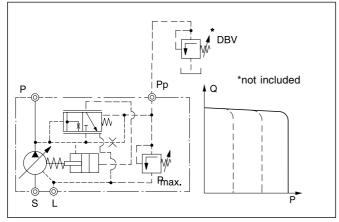
With thru shaft option for multiple pump options

Swash plate type for open circuit

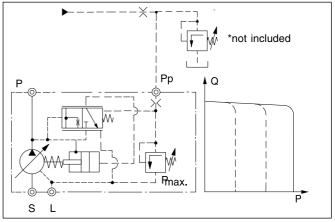


Representation of PVP16

- Mounting style metric or SAE.
- High strength cast iron body for high security and low noise.
- Port connections side, with SAE flange connection.
- Wide range of controls.
- Thru shaft.
- Selection of metric and SAE keyed shafts and SAE splined shafts.
- Compatible to HWBF (high water based fluids) up to 70 bar.



Pump with remote pressure compensator option: "M" (internal pilot supply)



Pump with remote pressure compensator option: "ME" (external pilot supply)



Axial Piston Pumps Series PVP

Technical data

Max. Displacement

16 to 140 cm³/rev

Pressure range Outlet

250 bar Nominal pressure p_N :

Maximum pressure p_{max} : 350 bar

with HWBF type fluids

on request

maximum 0.7 bar Inlet

minimum 0.8 bar absolute

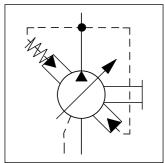
values for 1500 rpm

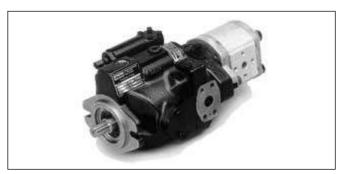
Drain port maximum 0.7 bar

Minimum speed 500 rpm

Temperature range -40°C to +70°C





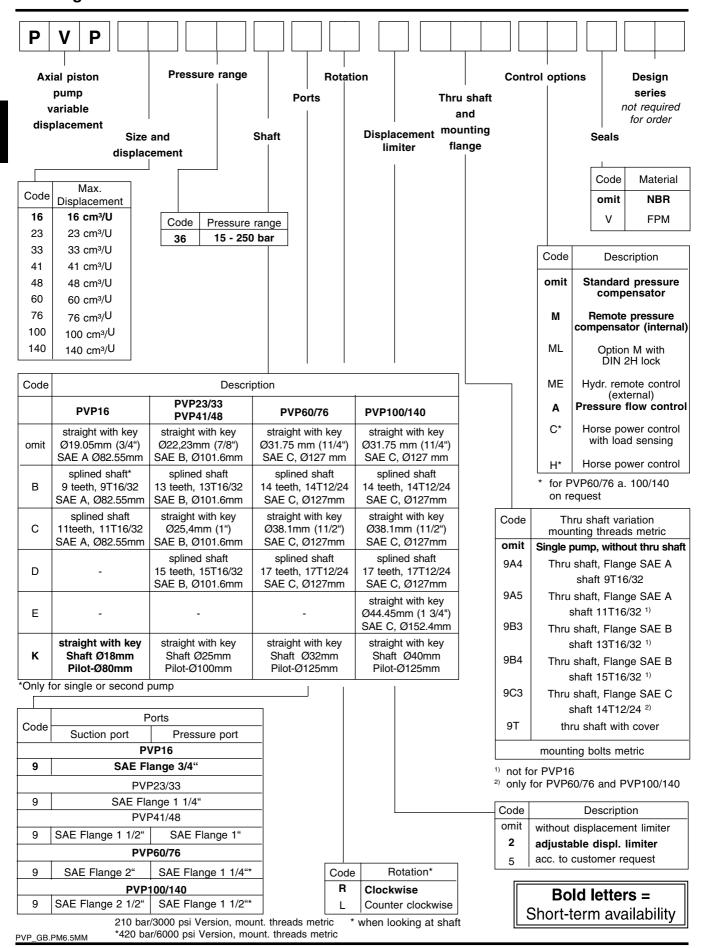


PVP with gear pump mounted

Selection table

Model	Max. displacement in cm³/rev	Delivery I/min at 1500 rpm and 20 bar	Input power at 1500 rpm and P _N in kW	Speed max. rpm	Weight in kg
PVP 16	16	24.6	11.3	3000	13.2
PVP 23	23	34.5	15.3	3000	20.4
PVP 33	33	49.5	21.3	3000	20.4
PVP 41	41	61.5	27.7	2800	25
PVP 48	48	72.0	33.6	2400	25
PVP 60	60	90.0	41.7	2200	41
PVP 76	76	114.0	52.8	2200	41
PVP 100	100	150.0	69.5	1800	82
PVP 140	140	210.0	97.3	1800	82

Ordering Code





Catalogue	HY11-2500/U	k
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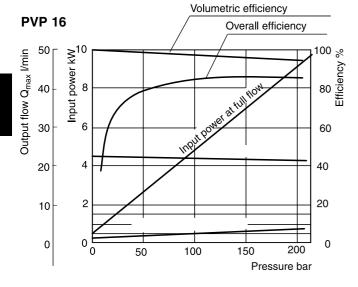
Axial Piston Pumps **Series PVP**

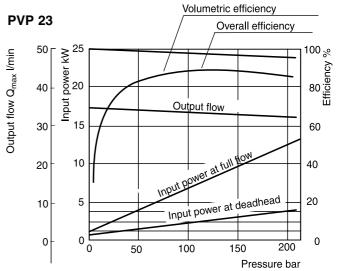
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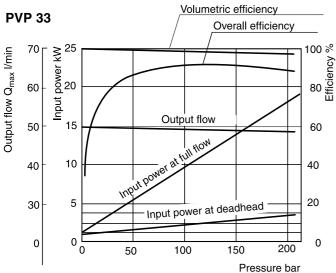
Notes	Series PVP

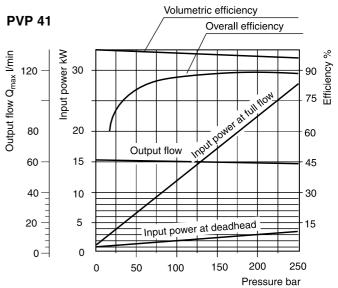


Characteristic Curves





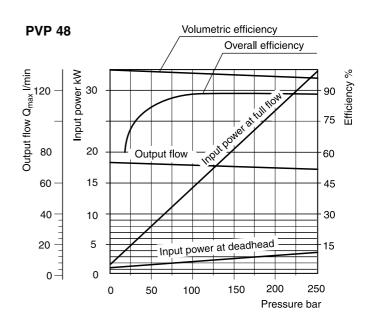




Flow rate at 1500 rpm Input power consumption and efficiency

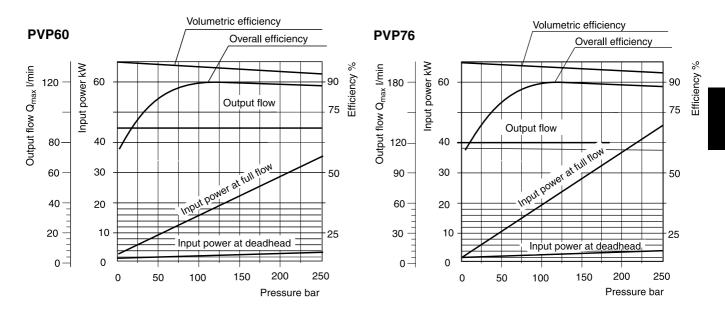
The values for the input power apply for fully compensated pumps.

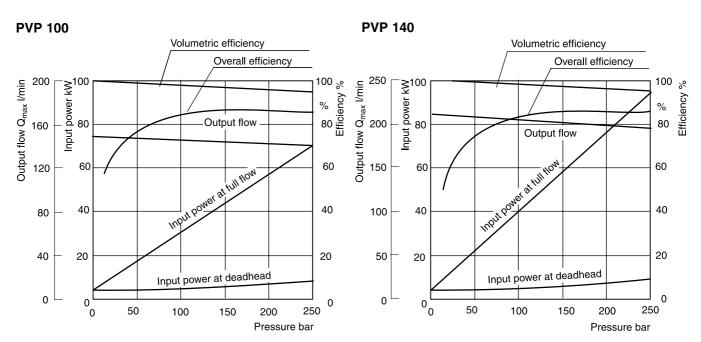
Values for flow, efficiency, and power consumption measured at 35° C and viscosity 30 mm²/s.





Characteristic Curves





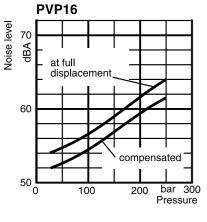
Flow rate at 1500 rpm Input power consumption and efficiency

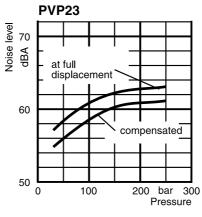
The values for the input power apply for fully compensated pumps.

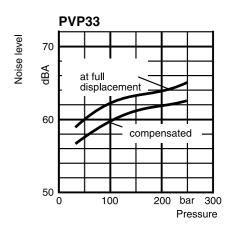
Values for flow, efficiency, and power consumption measured at 35° C and viscosity 30 mm²/s.

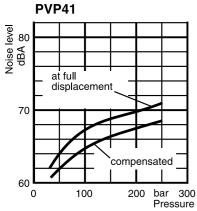


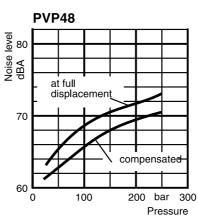
Noise Level

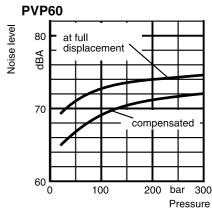


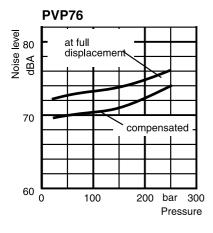


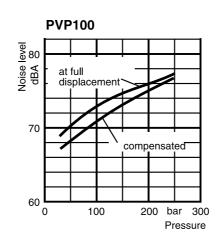


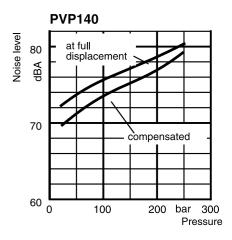








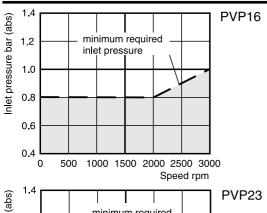


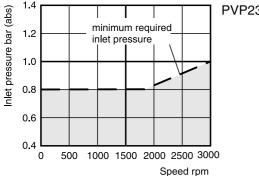


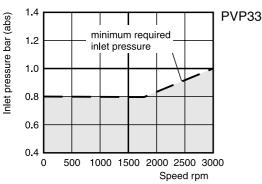
Typical sound level for single pumps, measured in unechoic chamber according to DIN 45 635, part 1 and 26. microphone distance 1 m. speed: n = 1500 min-1.

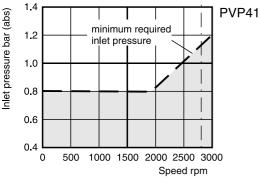
All data measured with mineral oil viscosity 30 mm 2 /s (cSt) at 50 $^{\circ}$ C.

Suction pressure requirements







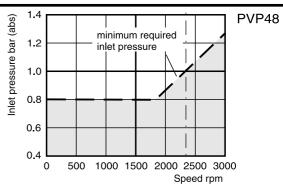


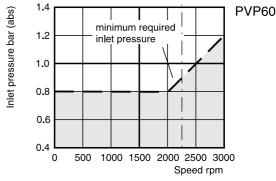
Suction limits for pumps from the PVP series

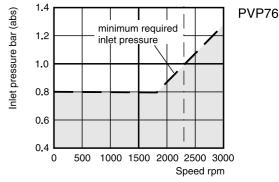
The pressures given represent the minimum required pressures on the inlet port of the pump (connection flange). Note that long, narrow suction lines, bends, T-pieces, and other add-ons can cause a considerable drop in pressure.

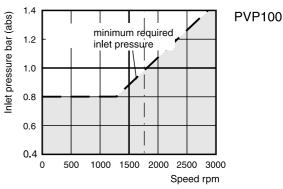
Suction filters must also be carefully dimensioned and serviced regularly.

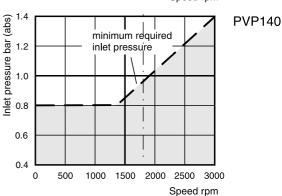
Low oil temperatures and dynamic processes (quick control movements) add to the loss in pressure.









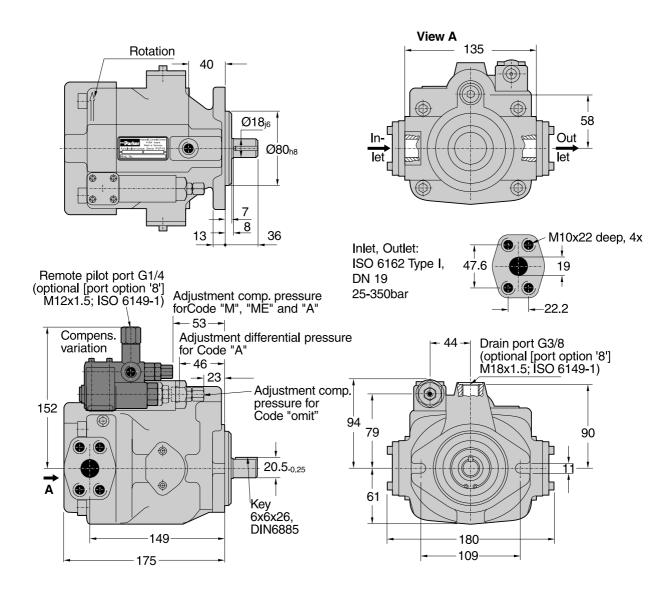




Metric Version

PVP16

The drawing shows a pump for **clockwise** rotation. For counter clockwise rotation, inlet port and outlet port are on the opposite sides. Compensator is always on the outlet port side.





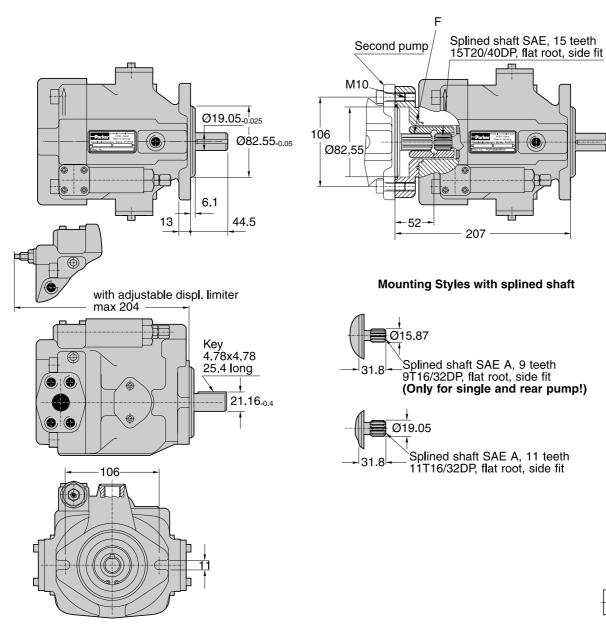
SAE and Thru Shaft Option

PVP16

The drawing shows a pump for **clockwise** rotation. For counter clockwise rotation, inlet port and outlet port are on the opposite sides. Compensator is always on the outlet port side.

Mounting style SAE and adjustable displacement limiter

Thru shaft option

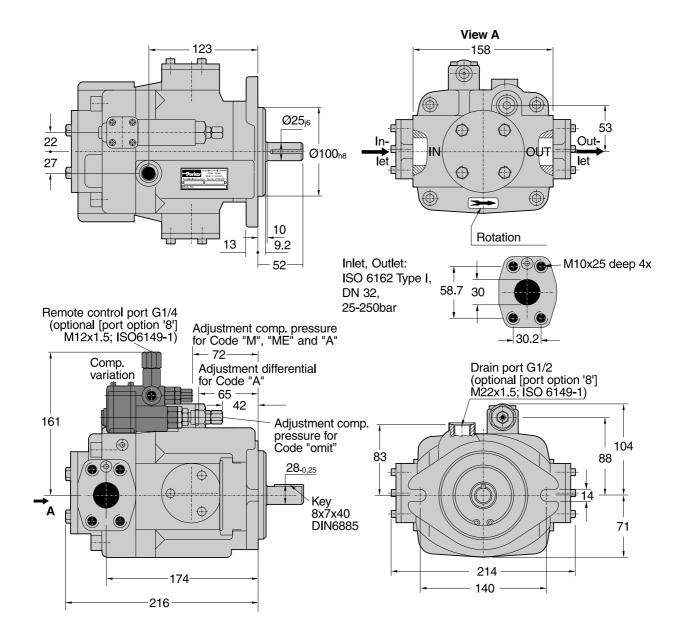




Metric Version

PVP23 and 33

The drawing shows a pump for **clockwise** rotation. For counter clockwise rotation, inlet port and outlet port are on the opposite sides.







SAE and Thru Shaft Option

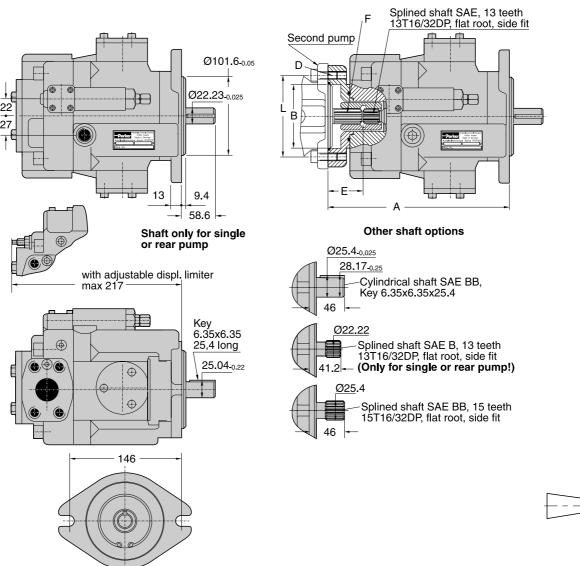
PVP 23 und 33

The drawing shows a pump for **clockwise** rotation. For counter clockwise rotation, inlet port and outlet port are on the opposite sides.

Variation	Α	B + 0.03	D	E	F	L
9 A 4	238.5	82.55	M10	54	9T16/32DP	106
9 A 5	238.5	82.55	M10	54	11T16/32DP	106
9B3	252.5	101.6	M12	68	13T16/32DP	146

Mounting style SAE and adjustable displacement limiter

Thru shaft option



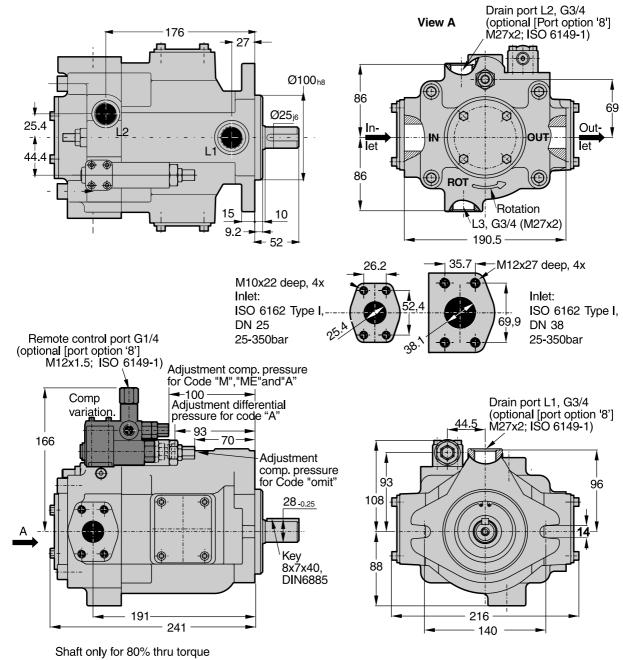




Metric Version

PVP41 and 48

The drawing shows a pump for **clockwise** rotation. For counter clockwise rotation, inlet port and outlet port are on the opposite sides.





SAE and Thru Shaft Option

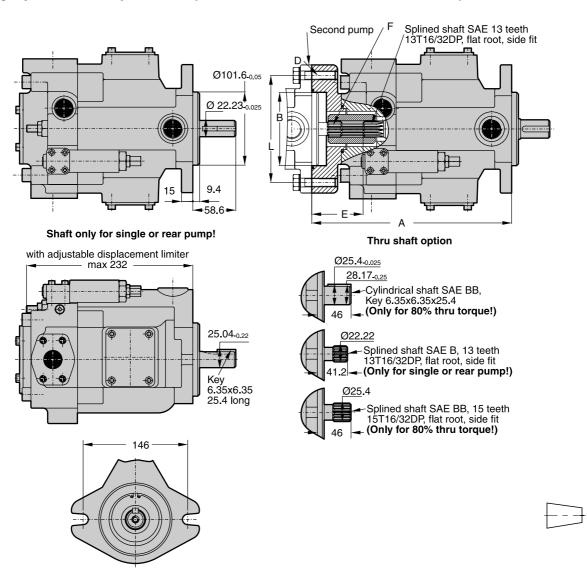
PVP 41 and 48

The drawing shows a pump for **clockwise** rotation. For counter clockwise rotation, inlet port and outlet port are on the opposite sides.

		D	E	F	L
263.5	82.55	M10	54	9T16/32DP	106
263.5	82.55	M10	54	11T16/32DP	146
277.5	101.6	M12	68	13T16/32DP	146
277.5	101.6	M12	68	15T16/32DP	146
	263.5 277.5	263.5 82.55 277.5 101.6	263.5 82.55 M10 277.5 101.6 M12	263.5 82.55 M10 54 277.5 101.6 M12 68	263.5 82.55 M10 54 11T16/32DP 277.5 101.6 M12 68 13T16/32DP

Mounting style SAE and adjustable displacement limiter

Thru shaft option

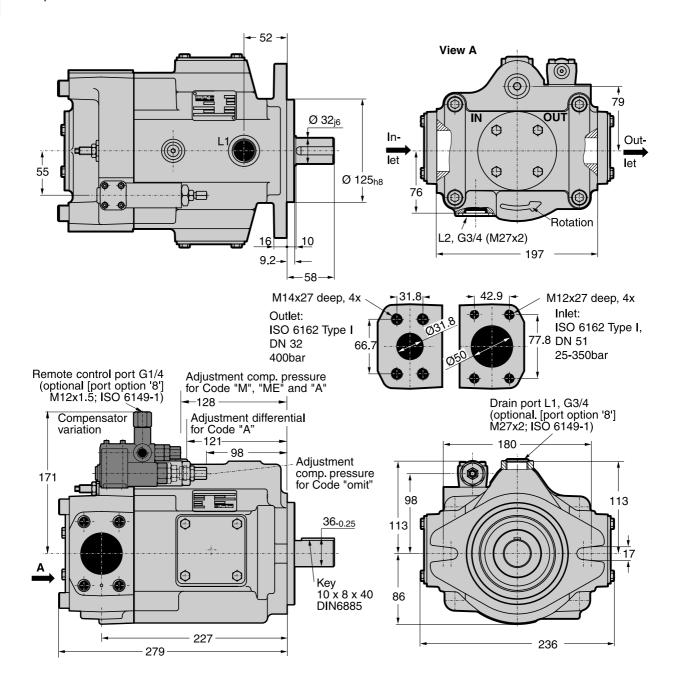




Metric version

PVP60 and 76

The drawing shows a pump for **clockwise** rotation. For counter clockwise rotation, inlet port and outlet port are on the opposite sides. Compensator is always on the outlet port side.







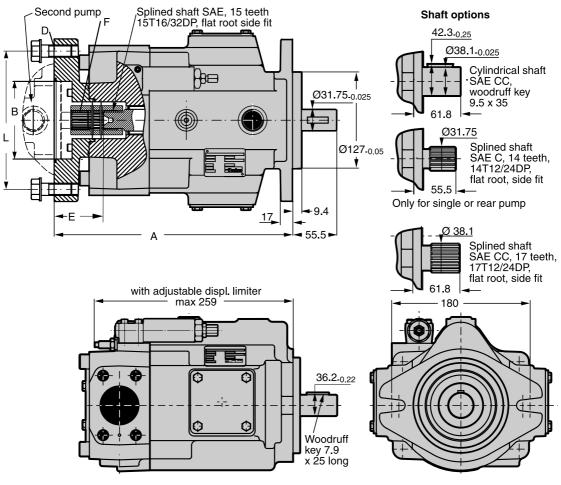
SAE and Thru Shaft Option

PVP60 and 76

The drawing shows a pump for **clockwise** rotation. For counter clockwise rotation, inlet port and outlet port are on the opposite sides. Compensator is always on the outlet port side.

Variation	A	B +0.03	D	E	F	L
9 A 4	298	82.55	M10	45	9T16/32DP	106
9B3	312	101.6	M12	59	13T16/32DP	146
9B4	312	101.6	M12	59	15T16/32DP	146
9C3	321	127	M16	68.5	14T12/24DP	181

Mounting style SAE, adjustable displacement limiter and thru shaft option

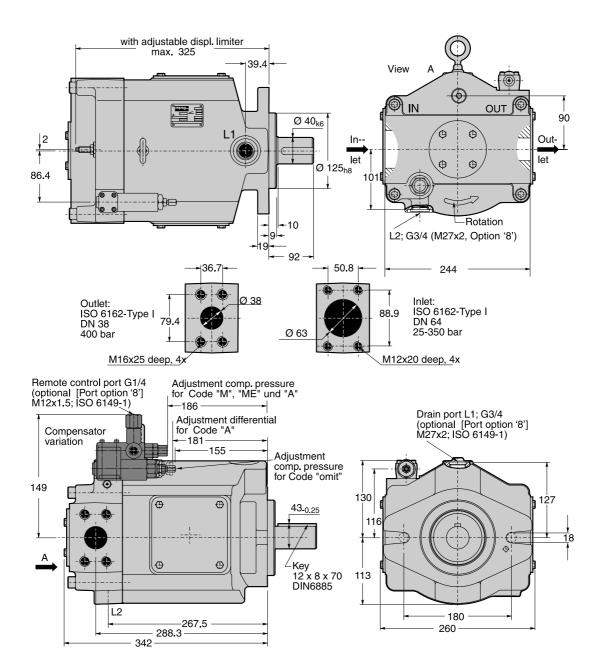




Metric Version

PVP 100 and 140

The drawing shows a pump for **clockwise** rotation. For counter clockwise rotation, inlet port and outlet port are on the opposite sides. Compensator is always on the outlet port side.







SAE and Thru Shaft Option

PVP100 and 140

The drawing shows a pump for **clockwise** rotation. For counter clockwise rotation, inlet port and outlet port are on the opposite sides. Compensator is always on the outlet port side.

Variation	Α	B + 0.03	D	E	F	L
9 A 4	364.5	82.55	M10	52	9T16/32	106
9B3	378.5	101.6	M12	66	13T16/32	146
9B4	378.5	101.6	M12	66	15T16/32	146
9C3	388	127	M16	75.5	14T12/24	181

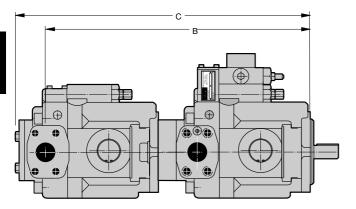
Mounting style SAE and thru shaft option

Shaft options 35.3-0.25 Ø31.75-0.025 Cylindrical shaft SAE C Woodruff key 7.9 x 25 55.5 Only for single pump Splined shaft SAE, 14 teeth /14T12/24DP, flat root side fit Second pump Ø31.75 Splined shaft SAE C, 14 teeth, 14T12/24DP, flat root, side fit Ø 38.1.0.025 Only for single or rear pump Ø38.1 Splined shaft SAE CC, 17 teeth, 17T12/24DP, Ø 127_{-0.05} В flat root, side fit 61.8 49.4_{-0.25} Ø 44.45_{-0.025} Cylindrical shaft SAE D, Key: 11.1 x 11.1 x 38.1 Ø 152.4_{-0.05} **←** 74.7> ▐₽₽ 0 (⊕ 4<u>2.4_{-0.25}</u> Ð €, Woodruff key 9.53 x 34.9 181 for SAE C flange 228.6—for SAE D flange



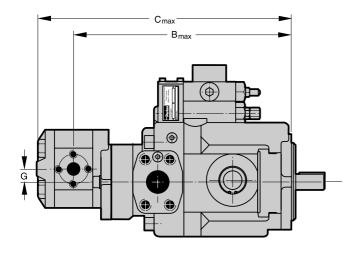
Pump Combinations

Combinations PVP/PVP



Main pump	Second pump	В	С
PVP16	PVP16	356	382
PVP23/33	PVP16	387.5	413.5
PVP23/33	PVP23/33	426.5	468.5
PVP41/48	PVP16	412.5	438.5
PVP41/48	PVP23/33	451.5	493.5
PVP41/48	PVP41/48	468.5	518.5
PVP60/76	PVP16	446.5	472.5
PVP60/76	PVP23/33	485.5	527.5
PVP60/76	PVP41/48	502.5	552.5
PVP60/76	PVP60/76	539.5	588
PVP100/140	PVP16	513.5	539.5
PVP100/140	PVP23/33	552.5	594.5
PVP100/140	PVP41/48	569.5	619.5
PVP100/140	PVP60/76	615	667
PVP100/140	PVP100/140	676.3	730
1	1	ı	I

Combinations PVP/PGP (gear pump)



Attention: check max. torque on driving shaft. see page 83.

Main pump	Second pump	B* _{max}	C* _{max}
PVP16	PGP505	276.5	350
PVP23/33	PGP505	308	381.5
PVP41/48	PGP505	333	406.5
PVP41/48	PGP511	365.5	454
PVP60/76	PGP505	367.5	441
PVP60/76	PGP511	400	488.5
PVP100/140	PGP505	434	507.5
PVP100/140	PGP511	466.5	555

^{*} PGP505/PGP511 with lowest displacement for their size.

Note

The upper drawing shows a pump combination PVP33+PVP33, the lower drawing a combination PVP33+PGP505. The values in the table show the dimensions for the pump combination of the first column.

Thru drive, shaft load limitations

The max. transferable torque in Nm for the different shafts options are:

Shaft code	PVP16	PVP23/33	PVP41/48	PVP60/76	PVP100/140
omit	116	209	209	641	641
В	56	209	209	641	641
С	102	337	337	1218	1218
D		337	337	1218	1218
E					1320
К	105	250	250	650	1250

Important notice

The max. allowable torque of the individual shaft must not be exceeded. Therefore it is necessary to calculate the torque factor and compare it with the allowed torque limit factor in the table.

Required:	calculated torque factor < torque limit factor
-----------	--

To make the necessary calculations easier and more user friendly it is not required to calculate actual torque requirements in Nm and compare them with the shaft limitations.

Pump	Shaft	Torque limit factor
D) (D4.0	"omit"	6480
PVP16	С	5800
	К	5870
	"omit"	12040
PVP23/33	С	19470
	D	19470
	K	14450
	С	19490
PVP41/48	D	19490
	K	14400
	"omit"	37000
PVP60/76	С	70350
	D	70350
	K	37620
	С	69720
PVP100/140	D	69720
	E	75600
	К	71540

The table shows limit factors that include material specification, safety factors and conversion factors.

Total torque factor of the combination = sum of individual torque factors of all pumps

The torque factor (charcateristic value) of each individual pump is calculated by multiplying the max. operating pressure p of the pump (in bar) with the max. displacement Vg of the pump (in cm³/rev).

Torque factor of any pump

= p x Vg (pressure in bar x displacement in cm³/rev)



Compensators

Standard pressure compensator, code: "omit"

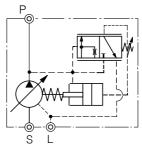
Description

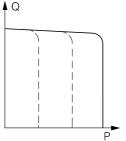
The standard pressure compensator controls the output flow (displacement, swash plate angle) in a way that when the set pressure is reached, the output flow of the pump just covers the flow needed by the system.

As long as the system pressure is below the set pressure - set by adjusting spring pre-load at the compensator - compensator spool is moved by the spring in a position where the servo piston is connected to the pump case drain port. The servo spring keeps the pump at full displacement.

When system pressure at pump outlet reaches the set pressure the compensator spool is moved against the spring into a position where the servo piston gets connected to the pump pressure port. The pressure moves the servo piston against the servo spring and reduces the swash plate angle of the pump. The displacement of the pump is reduced to exact the output flow required to keep the system pressure on the set level.

When no flow is needed by the system and the set pressure is reached the pump is compensated to deadhead. The displacement only covers the flow requirements of the pump (leakage of rotating group and compensator pilot flow).





Remote pressure compensator,

Code: "M" (internal pilot supply)

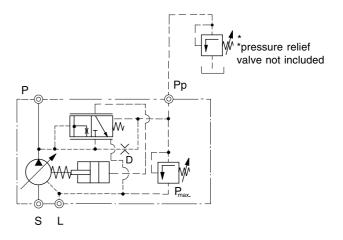
Description

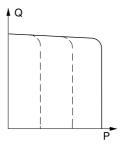
The remote hydraulic pressure compensator is controlled by a pressure relief valve set up outside of the pump, however, the pilot supply takes place internally using a orifice D.

The pressure relief valve must be designed for a flow rate of approx.1.2 l/min.

The built-on pressure relief valve pre-set at the factory for the maximum system pressure of p_{max} , protects the pump from overload. An arrangement of several pressure limiting valves (pressure stages) is possible, in connection with one or more directional valves.

The pressure compensator of the pump is set at a differential pressure. When setting the system pressure and when adjusting with one or more pressure limiting valves, the accumulative effect of the compensator must be taken into account.





Recommended Parker valves:

For inline mounting:

DUDB20*R15**

** Code for pressure ranges

PVP GB.PM6.5MM

For subplate mounting:

PVAC1PCMNS or

Continuous pressure valves RE06M**T2N1*01*X580 with integrated electronics
Continuous pressure valves RE06M**w2V1XP

Continuous pressure valves RE06M**w2V1XP Subplate G 3/8 SP1D23BA910



Compensators

Remote pressure compensator,

Code: "ME" (external pilot supply)

Description

The remote hydraulic pressure compensator is controlled by a pressure relief valve set up outside of the pump, however, the pilot supply takes place externally using a orifice with 0.8 mm diameter.

The pressure relief valve must be designed for a flow rate of ≥ 1.2 l/min.

The built-on pressure relief valve pre-set at the factory for the maximum system pressure of p_{max} , protects the pump from overload. An arrangement of several pressure limiting valves (pressure stages) is possible, in connection with one or more directional valves.

The pressure compensator of the pump is set at a differential pressure. When setting the system pressure and when adjusting with one or more pressure limiting valves, the accumulative effect of the compensator must be taken into account.

Recommended Parker valves: For inline mounting:

DUDB20*R15**

*0.8 orifice and DBV are not included in delivery

Ø0,8*

For subplate mounting:

Axial Piston Pumps

Series PVP

PVAC1PCMNS or

Continuous pressure valves RE06M **T2N1*01*X580 with integrated electronics

Continuous pressure valve DSAE 1007P07*LA*X580 Subplate G 3/8 SP1D23BA910

Load sensing compensator, Code: "A" (Load Sensing)

Description

The delivery of the pump is controlled independent of load using the pressure load sensing compensator and a throttle valve DV (optional prop. DC valve) built into the pressure line. The throttle valve causes a pressure drop in the line of Δp . Using a pilot line, the pressure behind the throttle valve is directed to the compensator, whose sliding spool has the effect of a 2 way pressure compensator together with the adjustable pilot control cartridge spring. A change in the throttle opening – opening or closing the valve – produces a change in the pressure drop Δp . A smaller pressure drop increases the pump's delivery, and vice versa.

The built-on pressure limiting valve pre-set at the factory for the maximum system pressure of p_{max} , protects the pump from overload.

Throttle valves without a 2 way pressure compensator are used as throttles with a manual or electric-proportional adjustment.

Recommended Parker valves:

For inline mounting:

9N1200S

PVP_GB.PM6.5MM

*throttle valve not included in delivery Pp Pp Pp Rax. Q

For subplate mounting:

Proportional valves series D31FH or D31FT



^{**}Code for pressure ranges

General Installation Information

Fluid recommendations

Premium quality hydraulic mineral oil fluids are recommended, like H-LP oils to DIN 51524, part 2. The viscosity range should be 25 to 50 mm²/s (cSt) at 50° C.Normal operating viscosity range between 12 and 100 mm²/s (cSt). Maximum start-up viscositiy is 320 mm²/s. Operating temperature -10 to + 70° C. For other fluids such as phosphoric acid esters or for other operating conditions consult your Parker representative for assistance.

Seals

NBR (nitrile) seals are used for operation with hydraulic fluids based on mineral oil. For synthetic fluids, such as phosphoric acid esters, fluorocarbon seals are required. Consult your Parker representative for assistance.

Filtration

For maximum pump and system component functionability and life, the system should be protected from contamination by effective filtration. Fluid cleanliness should be in accordance with ISO classification ISO 4406. The quality of filter elements should be in accordance with ISO standards.

Minimum requirement for filtration rate x (µm):

General hydraulic systems for satisfactory operation:

Class 19/15, to ISO 4406

 $x = 25 \mu m (\beta_{25} \ge 75) \text{ to ISO } 4572$

Hydraulic systems with maximised component life and functionability:

Class 16/13, to ISO 4406

 $x = 10 \mu m \ (\beta_{10} \ge 75) \text{ to ISO } 4572$

It is recommended to use return line or pressure filters.Parker Filter Division offers a wide range of these filters for all common applications and mounting styles. The use of suction filters should be avoided, especially with fast response pumps.

Installation and mounting

Horizontal mounting: Outlet port side or top, inlet port side or bottom. Drain port always uppermost.

Vertical mounting: Shaft pointing upwards

Inlet (suction side): Install pump and suction line in such a way that the maximum inlet vacuum never exceeds 0.8 bar absolute. The inlet line should be as short and as

straight as possible. A short suction line cut to 45° is recommended when the pump is mounted inside the reservoir, to improve the inlet conditions. All connections to be leak-free, as air in the suction line will cause cavitation, noise, and damage to the pump.

Drain port

Compensation may cause short-term (20 to 30 ms) flow increase. Please consider for dimensioning.

Drain line

The drain line must lead directly to the reservoir without restriction. The drain line must not be connected to anv other return line. The end of the drain line must be below the lowest fluid level in the reservoir and as far away as possible from the pump inlet line. This ensures that the pump does not empty itself when not in operation and that hot aerated oil will not be recirculated.

For the same reason, when the pump is mounted inside the reservoir, the drain line should be arranged in such a way that a siphon is created. This ensures that the pump is always filled with fluid. The drain pressure must not exceed 1 bar. Drain line length should not exceed 2 metres. Minimum diameter should be selected according to the port size and a straight low pressure fitting with maximised bore should be used.

Shaft rotation and alignment

Pump and motor shafts must be aligned within 0.25mm T.I.R. maximum. A floating coupling must be used. Bell housings and couplings can be ordered at manufacturers listed in this catalogue. Please follow the coupling manufacturer's installation instructions. Consult your Parker representative for assistance on radial load type drives.

Start up

Prior to start up, as well as after long stand-still periods when it is possible that the pump body could have been emptied, the pump case must be filled with hydraulic fluid (use case drain port). Initial start up should be at zero pressure with an open circuit to enable the pump to prime. Pressure should only be increased once the pump has been fully primed. A quick on and off switching (inching mode) makes priming easier and enables quick filling of the displacement space in the pump.

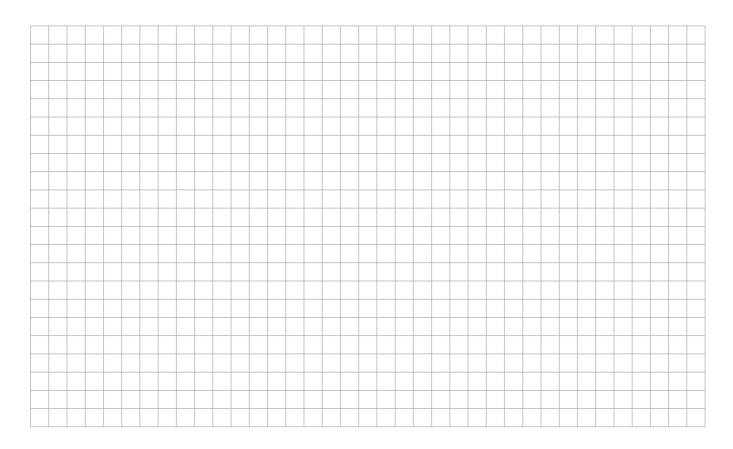
Attention: Check motor rotation direction. Compare Chapter 12.

PVP GB.PM6.5MM

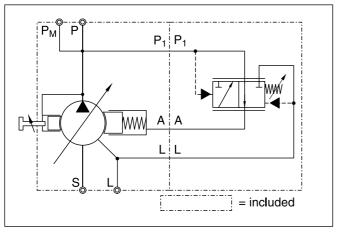


Catalogue HY11-2500/U

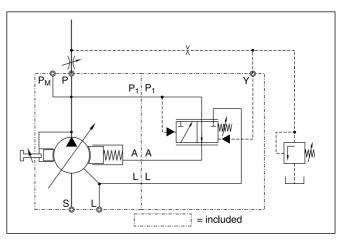
Catalogue HY11-2500/UK Notes		







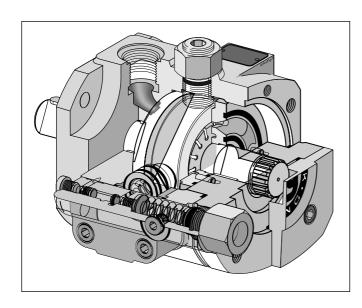




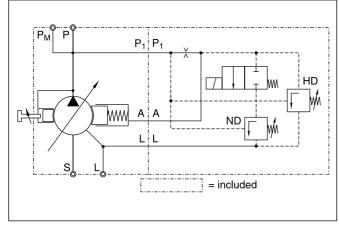
Pump with pressure flow compensator, Code PVM

With thru shaft option for multiple pump options

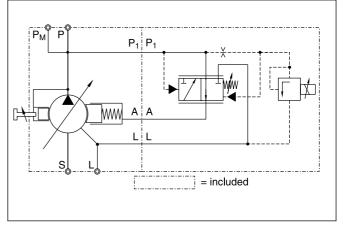
for open circuit



- Mounting pattern according to VDMA 24560/1 specification.
- 4 bolt flange ISO 3019/2 (metric).
- Fast response.
- Wide range of controls for diverse tasks.
- Low noise level.
- Good efficiency.



Pump with two-stage compensator, Code PVH



Pump with proportional pressure compensator, Code PVL



Characteristics

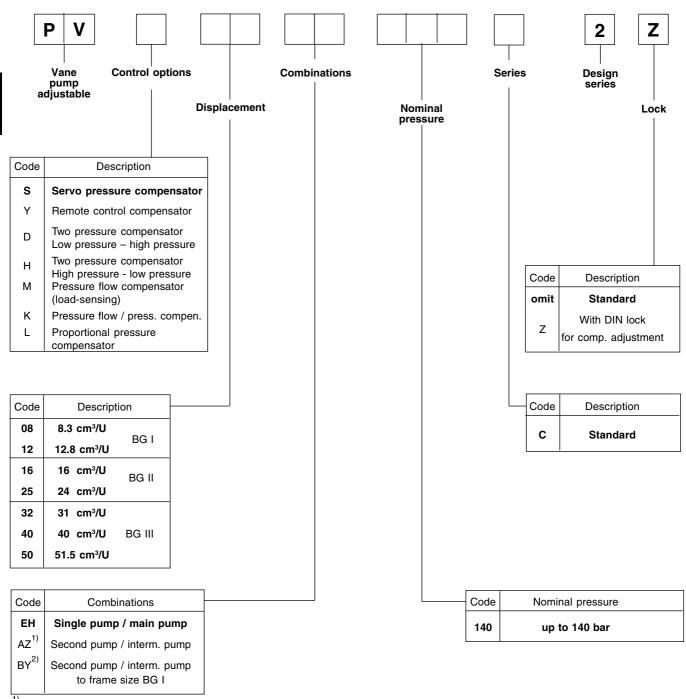
Displacement 8 - 50 cm³/rev Pressure range Outlet Nominal pressure 140 bar Inlet 1.0 bar 0.8 bar abolute **Drain port** maximum 0.5 bar Speed ranges 1000 - 1800 rpm Press. fluid temperature -10°C ...+70°C Viscosity range 22 - 100 mm²/s $800 \text{ mm}^2/\text{s}$ (short-term at start up) Rotation clockwise



Characteristics and weights

Model	Displacement in cm ³ /rev	Output flow at 1500 rpm in l/min	Input power at nominal pressure in kW	Weight in kg single pump	Weight in kg main pump	Weight in kg intermediate pump	Weight in kg second pump
PVS08	8.3	12	3.65	8.9	8.9	8.8	8.8
PVS12	12.8	19	5.0	8.9	8.9	8.8	8.8
PVS16	16	23	8.7	18.1	16.9	18.0	16.8
PVS25	24	35	9.9	18.1	16.9	18.0	16.8
PVS32	31	45	12.7	33.2	30.8	33.0	30.6
PVS40	40	60	15.9	33.2	30.8	33.0	30.6
PVS50	51.5	75	19.7	33.2	30.8	33.0	30.6

Ordering Code



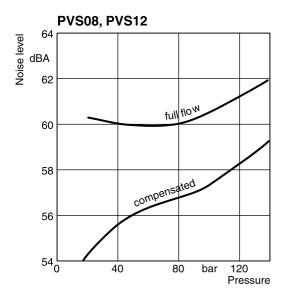
only for BG II and BG III

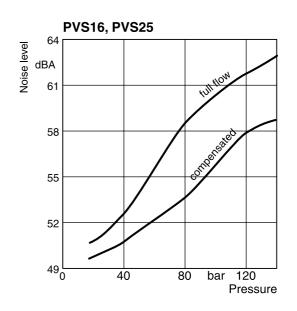
Bold letters = Short-term availability

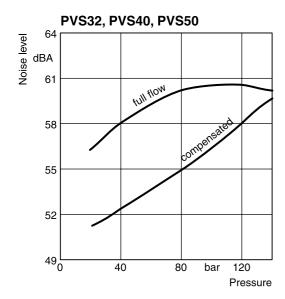


only for BG I

Noise Levels







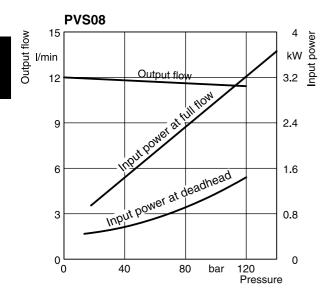
Typical noise levels for single pumps, measured in an unechoic chamber according to DIN 45 635 Microphone distance 1 m.

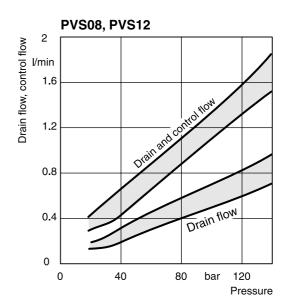
Speed n = 1.500 rpm

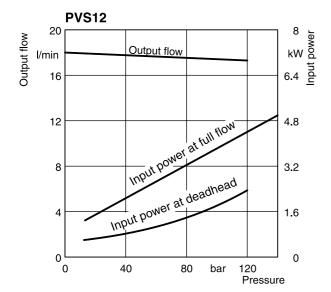
All values measured with mineral oil at a viscosity of 30 mm²/s and 50 $^{\circ}$ C.

Characteristic Curves

Size I, Size 08 and 12







Characteristic curves determin. at speed n = 1500 rpmAll values were measured with mineral oil at a viscosity of 30 mm 2 /s and 50 $^\circ$ C.

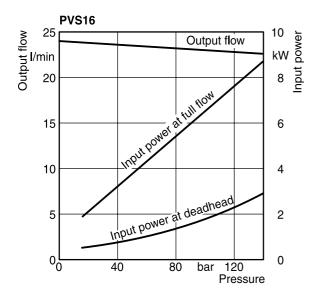
All characteristics shown are typical. They can deviate by up to 5% of the shown values depending on production tolerances of new pumps under certain conditions.

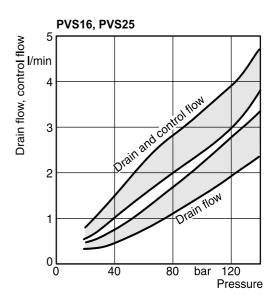
Please note: The values shown for drain and pilot oil apply for quasi-static operation (constant operation conditions).

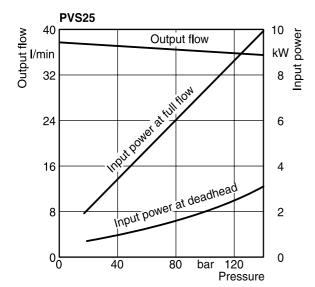
During the pilot processes, significantly higher pilot oil flows can take place in short-term and can exceed 20 I/ min in extreme cases. Therefore, it is absolutely necessary to set up the drain line without restrictions and as short as possible to avoid unacceptably high pressure peaks in the pump body.

Characteristic Curves

Size II, Size16 and 25







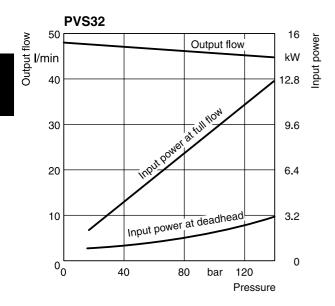
Characteristic curves determin. at speed $\,$ n = 1500 rpm All values were measured with mineral oil at a viscosity of 30 mm²/s and 50 $^{\circ}$ C.

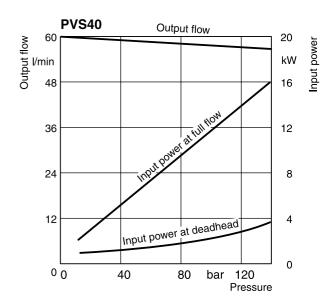
All characteristics shown are typical. They can deviate by up to 5% of the shown values depending on production tolerances of new pumps under certain conditions.

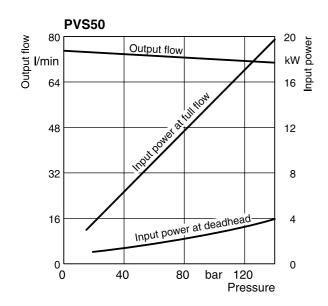
Please note: The values shown for drain and pilot oil apply for quasi-static operation (constant operation conditions).

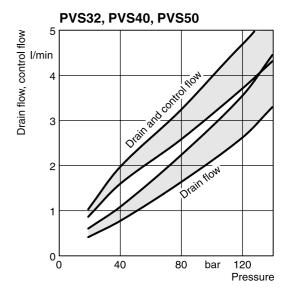
During the pilot processes, significantly higher pilot oil flows can take place in short-term and can exceed 20 l/min in extreme cases. Therefore, it is absolutely necessary to set up the drain line without restrictions and as short as possible to avoid unacceptably high pressure peaks in the pump body.

Size III, Size 32, 40, 50









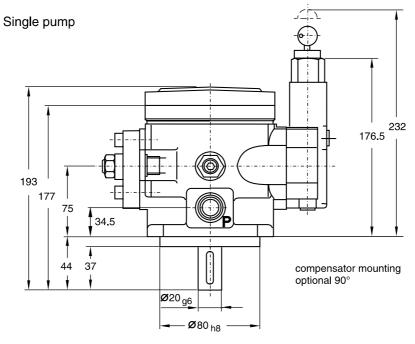
Characteristic curves determin. at speed $\,$ n = 1500 rpm All values were measured with mineral oil at a viscosity of 30 mm²/s and 50° C.

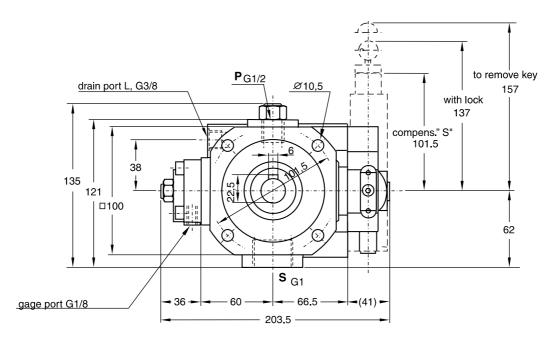
All characteristics shown are typical. They can deviate by up to 5% of the shown values depending on production tolerances of new pumps under certain conditions.

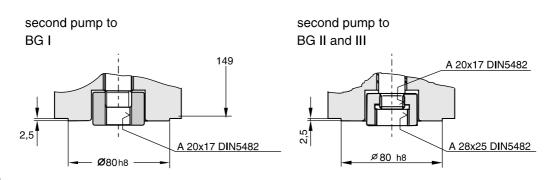
Please note: The values shown for drain and pilot oil apply for quasi-static operation (constant operation conditions).

During the pilot processes, significantly higher pilot oil flows can take place in short-term and can exceed 20 l/min in extreme cases. Therefore, it is absolutely necessary to set up the drain line without restrictions and as short as possible to avoid unacceptably high pressure peaks in the pump body.

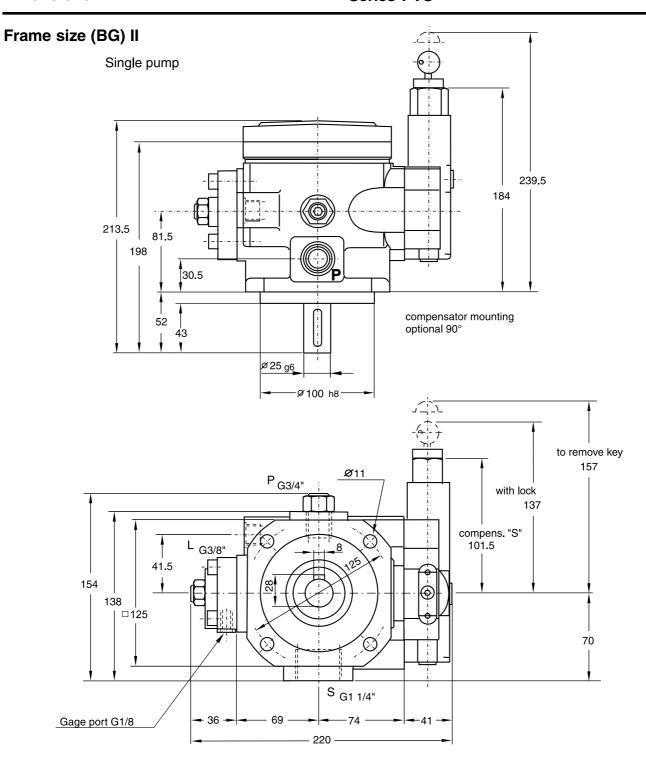
Frame size (BG) I



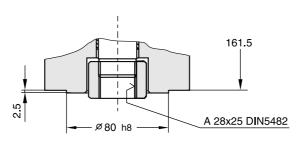






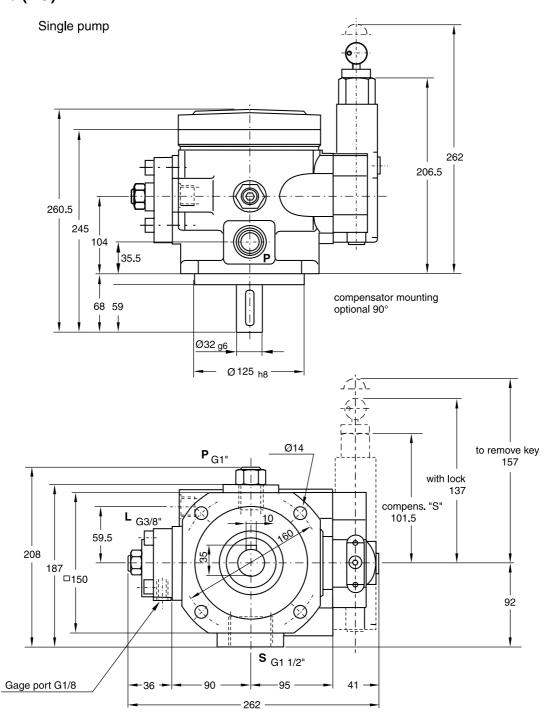




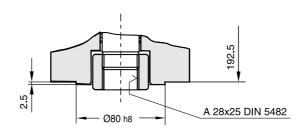




Frame size (BG) III

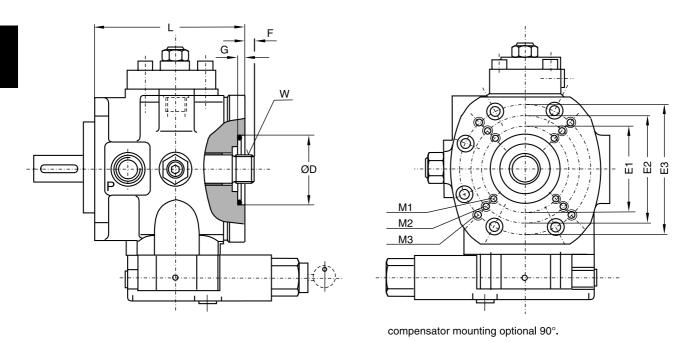






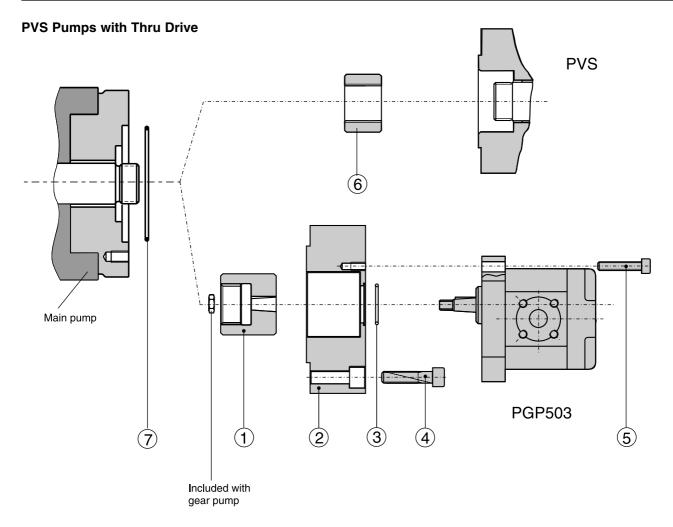


Thru drive



Pump	L	D	F	G	Thru drive shaft "W"	М1	M2	М3	E1	E2	E3
PVS 08 or 12 PVS 16 or 25 PVS 32, 40, or 50	133 146 177	₈₀ H7	7	4.5	B20x17 DIN 5482 B28x25 DIN 5482 B28x25 DIN 5482	M8 M8 M8	- M10 M10	- - M12	100 100 100	- 125 125	- - 160

Pump Combinations



The drawing displays the mounting possibilities for Parker pumps. Other pumps on request.

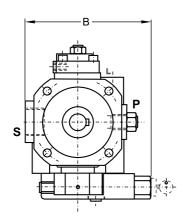
Table

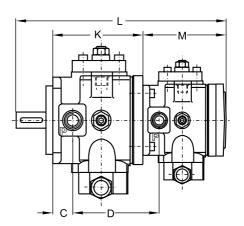
Mounting parts for pump combinations

Main pump	Second pump	Coupling Pos.: 1	Adapter Pos.: 2	O-Ring Pos.: 3	Screw Pos.: 4	Screw Pos.: 5	Coupling Pos.: 6	O-Ring Pos.: 7
PVS 08-12	PVS 08-12	-	-	-	-	-	HR10047482	2-151-V747-75
	PGP503	HR10056670	HR10056667	HR01090121	M8x35	M6x25	-	2-151-V747-75
PVS 16-25	PVS 08-12	-	-	-	-	-	HR10047479	2-151-V747-75
	PVS 16-25	-	-	-	-	-	HR10047342	2-151-V747-75
	PGP503	HR10056673	HR10056667	HR01090121	M8x35	M6x25	-	2-151-V747-75
PVS 32-50	PVS 08-12	-	-	-	-	-	HR10047479	2-151-V747-75
	PVS 16-25	-	-	-	-	-	HR10047342	2-151-V747-75
	PVS 32-50	-	-	-	-	-	HR10047342	2-151-V747-75
	PGP503	HR10056673	HR10056667	HR01090121	M8x35	M6x35	-	2-151-V747-75



Combinations PVS/PVS



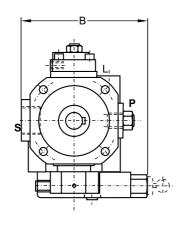


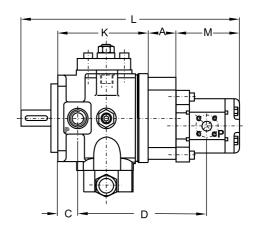
The compensators can only be mounted in the direction shown.

Main pump	Second pump	Interface main pump	В	С	К	М	D	L
PVS 08 or 12	PVS 08 or 12	80 B4 HW	163.5	34,5	133	149	133	326
PVS 16 or 25	PVS 08 or 12	100 B4 HW	171.5	30.5	146	149	150	347
	PVS 16 or 25					161.5	146	359.5
* PVS 32, 40, or 50	PVS 08 or 12					149	176	394
	PVS 16 or 25	125 B4 HW	193.5	35.5	177	161.5	172	406.5
	PVS 32, 40, or 50					192.5	177	437.5

 $^{^{*}}$ Without lock, the compensators can be optionally mounted rotated 90°, in the following combinations: PVS 32/40/50 + PVS 16/25

Combinations PVS/GP (gear pump)





Compensator can be optionally mounted, rotated $90^{\circ}.$

Main pump	Second pump	Interface main pump	A	В	С	D*	К	L*	M*
PVS 08 or 12 PVS 16 or 25 PVS 32, 40, or 50	PGP503	80 B4 HW 100 B4 HW 125 B4 HW	38	163.5 171.5 193.5	34.5 30.5 35.5	183.5 200.5 226.5	133 146 177	265 278 309	94

 $^{^{\}star}$ maximum length with largest displacement of a gear pump frame size



PVS 32/40/50 + PVS 32/40/50

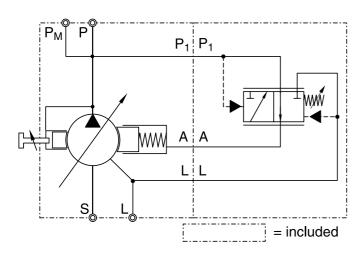
Series PVS

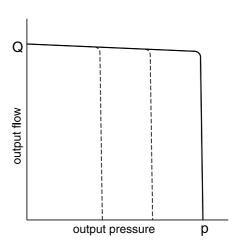
Compensator type S

PVS - Standard-pressure compensator.

The pressure is mechanically adjustable via the preload of the pilot control cartridge spring.

Schematic diagram and performance curves



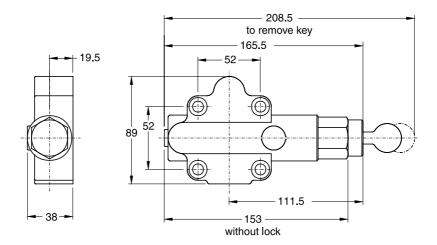


Task and function

When reaching the set pressure on the compensator, the pressure flow of the servo-controlled pump is automatically adjusted to the actual pressure flow requirement of the consumer.

Thus an undesired flow is avoided and only the required medium amount is delivered. As long as the system pressure is lower than the set pressure on the compensator, the stroke ring is kept in the position of maximum eccentricity, so that the pump continues its full delivery. If the system pressure exceeds the set compensator pressure, the control valve opens, and the pressure on the control piston is relieved. The stroke ring is moved by the auxiliary piston up to the central position to the point where the pressure flow corresponds to the system requirements at the set pressure. The pump is regulated.

Dimensions

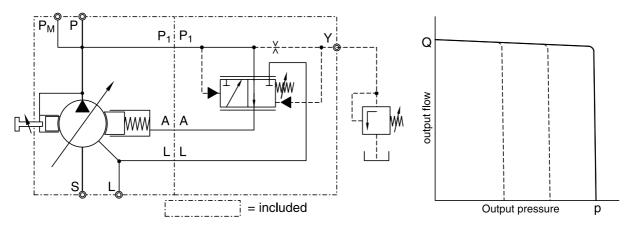




Compensator type Y

PVY - Remote controlled pressure compensator. Pressure is adjustable hydraulically via pilot valve connected to Y port.

Schematic diagram and performance curves



Task and function

The range of application for the remote control compensator is similar to the proportional compensator. The pump can be mounted in an inaccessible position (e.g. in an oil container). It is possible for the operating personnel to adjust the desired system pressure via a pressure limiting valve from a remote control desk. It should be noted however, that the response times for the pump increase with increasing control cable length. The remote control compensator functions in principle like a pilot-operated pressure limiting valve. In contrast to the servo pressure compensator, the force contained in the equilibrium of the system pressure on the

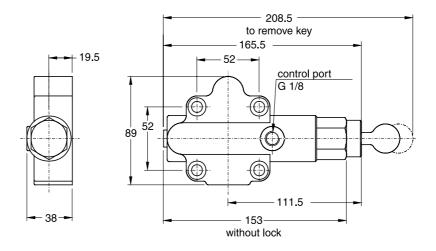
compensator piston is applied not only by the compensator valve spool, but also by additional pressure from the spool area together with an external pilot valve (pressure limiting valve). The actual control process in the pump corresponds to that of the servo pressure compensator (an external pilot valve is not included with the pump).



Note

For safety reasons, the Y port of the remote control compensator must never be closed. Otherwise, the pump will not compensate.

Dimensions



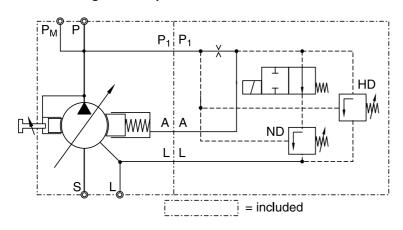


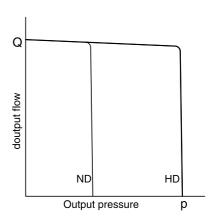
Compensator type D Low pressure - high pressure

Double Pressure Compensator

PVD – Two stage pressure compensator. High pressure and low pressure mechanically adjustable via spring pre-loading, electric switching.

Schematic diagram and performance curves



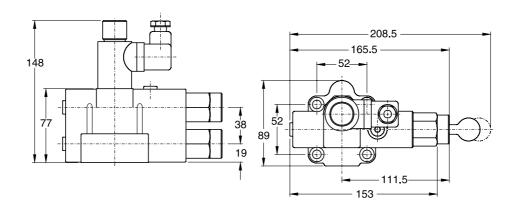


Task and function

The double pressure compensator offers the user the possibility to electrically select between two different pressures. Hydraulic systems, where a higher pressure is only needed in peaks, can be created to be very energy-saving, based on such a design. The double pressure compensator can also be labelled as a double servo pressure compensator, divided into low and high pressure stages. Both compensator pistons are connected together via an integrated directional valve.

Initially both compensator pistons are pressurised with system pressure at the unloaded directional valve. The compensator piston with the lower spring pre-loading is responsible for the system pressure. If the directional valve piston is changed over from LP to HP via electrical signal, the connection to the low pressure compensator piston is interrupted. Then, only the high pressure compensator piston is connected to the pilot oil space. The actual control process for the pump corresponds to one from a servo pressure compensator.

Dimensions

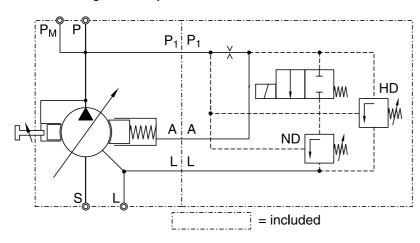


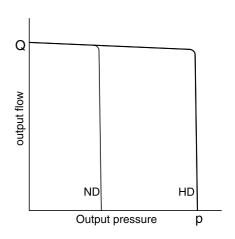


Compensator type H Low pressure – high pressure

 $PV\underline{H}$ – Two stage pressure compensator. High pressure and low pressure mechanically adjustable via spring pre-loading, electric switching.

Schematic diagram and performance curves



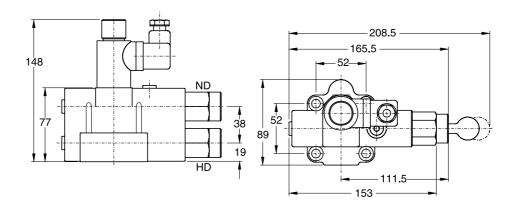


Task and function

The double pressure compensator offers the user the possibility to electrically select between two different pressures. Hydraulic systems, where a lower pressure is only needed for short intervals, can be created very easily, based on such a design. The double pressure compensator can also be labelled as a double servo pressure compensator, divided into low and high pressure stages. Both compensator pistons are connected together via an integrated directional valve.

Only the high pressure stage is pressurised with system pressure at the unloaded directional valve. If the directional valve piston is changed over from HP to LP via electrical signal, the connection to the low pressure compensator piston is created. Both compensator pistons are then connected with the pilot oil space. The compensator piston with the lower spring pre-loading is responsible for the system pressure. The actual control process for the pump corresponds to one from a servo pressure compensator.

Dimensions





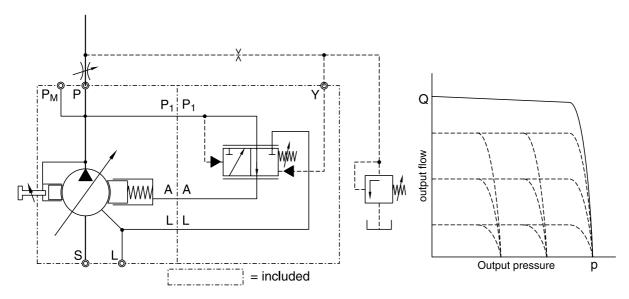
Pressure Flow Compensator

Compensator type M Pressure flow compensator

PVM - Pressure flow compensator.

Flow adjustable via main stream throttle valve, load pressure independent flow control, no internal pressure compensation.

Schematic diagram and performance curves



Task and function

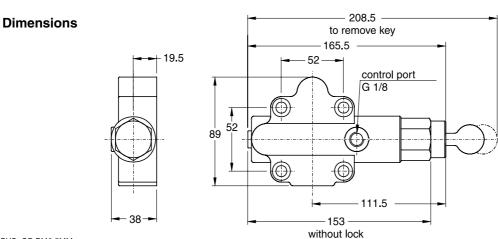
The pressure flow compensator is responsible to keep the pressure flow of the pump to the metering position (orifice, choke, proportional valve, etc.) constant despite fluctuations in load and input speed. However, it must be remembered that this compensation is not possible at \mathbf{Q}_{max} . To ensure proper control behaviour, a maximum of approx. 2/3 \mathbf{Q}_{max} should be worked with. The necessary constant pressure differential is achieved for constant flow at the metering position by directing both

pressures (pressure before and after the metering position) on to the compensator piston, such that the lower pressure (pressure behind the metering position) with the compensator valve spool works against the pump pressure (2 way pressure compensator function). (Throttle and external pressure limiting valve are not included with the pump).



Note

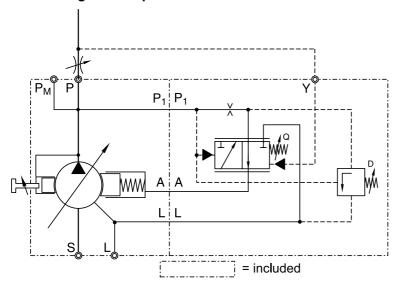
When using the pressure flow compensator, max. pressure protection via an external pressure limiting valve is absolutely necessary. Otherwise, the pump will not compensate.

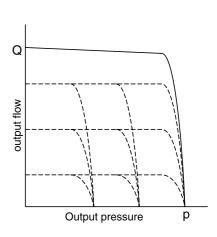


Compensator type K Pressure flow pressure compensator

 $PV\underline{K}$ - Pressure flow - pressure compensator. Flow adjustable via main stream throttle valve, pressure mechanically adjustable via pre-load spring, load pressure independent flow control.

Schematic diagram and performance curves



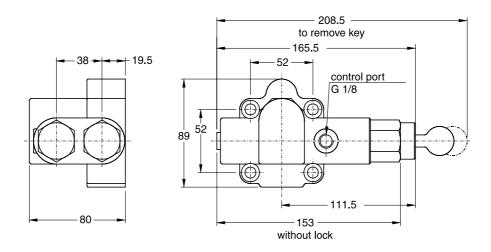


Task and function

The pressure flow - pressure compensator is a compensation device that was specially designed for use in load sensing systems. The displacement control is executed dependent on load, i.e. the optimal ratio of pressure and flow are set independent of pending load pressure on the consumer (e.g. hydro-motor). A characteristic feature of all load sensing compensators is the feedback of the load pressure (Y). In systems with variable load pressures, this control is characterised by the energetic and functional superiority compared to conventional compensators. On the compensator are two basic

system-dependant settings undertaken, the setting for the differential pressure (Δp) necessary for the pressure flow (Q) and the setting of the maximum pressure (D). The setting (Q) is a result of the differential pressure (Δp) with which a metering position is flown through (orifice, choke, proportional valve, etc.). If the load pressure is altered at the consumer or the pressure in the feedback (Y), the pump decreases or increases its pressure until the differential pressure set on (Q) is reached again (2 way pressure compensator function). This process takes place continuously, up until the pressure set on (D) is reached.

Dimensions



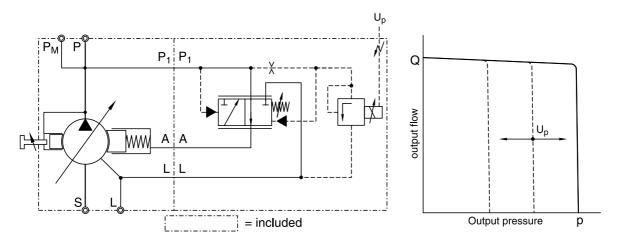


Series PVS

Compensator type L **Proportional pressure compensator**

PVL - Proportional pressure compensator. Pressure can be adjusted electrically using a proportional solenoid and control electronics.

Schematic diagram and performance curves



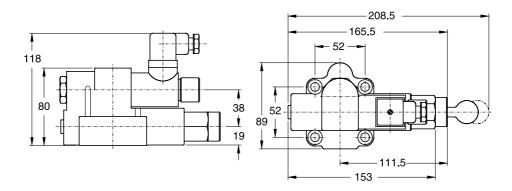
Task and function

The range of application for the proportional compensator is similar to the remote control compensator. The pump can be mounted in an inaccessible position. It is possible for the operating personnel to adjust the desired system pressure from a remote control desk, manually or by a program.

Further advantages are the controllable process of the

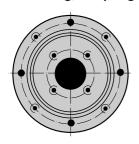
transition between various command settings, the reproducibility of the control pressure, and fast response times. The principle of the proportional compensator is similar to the servo pressure compensator. Setting the pressure does not take place at the compensator, but instead through infinitely variable control via a pilot valve with proportional solenoid.

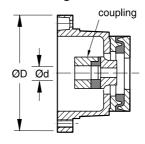
Dimensions

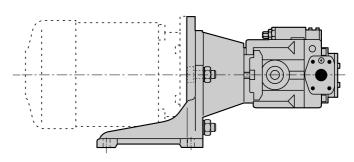




Bell housing, coupling and foot flange







Can be purchased at:

Raja

Rahmer + Jansen GmbH Vorthstr. 1 58775 Werdohl, Germany

Tel.: (+2392) 5090, fax: (+2392) 4966

KTR

Kupplungstechnik GmbH Rodder Damm 48432 Rheine, Germany

Tel.: (+5971) 798-0, fax: (+5971) 798443

General Installation Information

Fluid recommendations

Premium quality hydraulic mineral oil fluids are recommended, like H-LP oils to DIN 51524, part 2. The viscosity range should be 25 to 50 mm²/s (cSt) at 50° C.Normal operating viscosity range between 12 and 100 mm²/s (cSt). Maximum start-up viscositiy is 320 mm²/s. Operating temperature -10 to + 70° C. For other fluids such as phosphoric acid esters or for other operating conditions consult your Parker representative for assistance.

Filtration

For maximum pump and system component functionability and life, the system should be protected from contamination by effective filtration. Fluid cleanliness should be in accordance with ISO classification ISO 4406. The quality of filter elements should be in accordance with ISO standards.

Minimum requirement for filtration rate x (μm):

General hydraulic systems for satisfactory operation:

Class 19/15, to ISO 4406

 $x = 25 \mu m (\beta_{25} \ge 75)$ to ISO 4572

Hydraulic systems with maximised component life and functionability:

Class 16/13, to ISO 4406 $x = 10 \mu m (\beta_{10} \ge 75)$ to ISO 4572

It is recommended to use return line or pressure filters.Parker Filter Division offers a wide range of these filters for all common applications and mounting styles. The use of suction filters should be avoided, especially with fast response pumps.

Installation and mounting

<u>Horizontal mounting:</u> Outlet port side or top, inlet port side or bottom. Drain port always uppermost.

Vertical mounting: Shaft pointing upwards

Inlet (suction side): Install pump and suction line in such a way that the maximum inlet vacuum never exceeds 0.8 bar absolute. The inlet line should be as short and as straight as possible. A short suction line cut to 45° is

recommended when the pump is mounted inside the reservoir, to improve the inlet conditions. All connections to be leak-free, as air in the suction line will cause cavitation, noise, and damage to the pump.

Drain line

The drain line must lead directly to the reservoir without restriction. The drain line must not be connected to any other return line. The end of the drain line must be below the lowest fluid level in the reservoir and as far away as possible from the pump inlet line. This ensures that the pump does not empty itself when not in operation and that hot aerated oil will not be recirculated.

For the same reason, when the pump is mounted inside the reservoir, the drain line should be arranged in such a way that a siphon is created. This ensures that the pump is always filled with fluid. The drain pressure must not exceed 1 bar. Drain line length should not exceed 2 metres. Minimum diameter should be selected according to the port size and a straight low pressure fitting with maximised bore should be used.

Shaft rotation and alignment

Pump and motor shafts must be aligned within 0.25mm T.I.R. maximum. A floating coupling must be used. Bell housings and couplings can be ordered at manufacturers listed in this catalogue. Please follow the coupling manufacturer's installation instructions. Consult your Parker representative for assistance on radial load type drives. An axial load on the pump shaft is not permitted.

Start up

Prior to start up, as well as after long stand-still periods when it is possible that the pump body could have been emptied, the pump case must be filled with hydraulic fluid (use case drain port). Initial start up should be at zero pressure with an open circuit to enable the pump to prime. Pressure should only be increased once the pump has been fully primed. A quick on and off switching (inching mode) makes priming easier and enables quick filling of the displacement space in the pump.

Attention: Check motor rotation direction. See also the statements on hydraulic fluids in Chapter 12.

